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# Improved model for transmission probabilities of edge-welded bellows based on TPMC simulations



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# ABSTRACT

Many complex vacuum systems include edge-welded bellows. Their simulation in the molecular flow regime with a Test Particle Monte Carlo (TPMC) code, such as MolFlow+, can take considerable amounts of computing power and time. Therefore we investigated the change of the transmission probability of a bellow compared to a cylindrical tube in TPMC simulations. The results were used to develop an empirical model to simplify the geometry of a bellow in a TPMC model by replacing it with a cylindrical tube of extended length, consequently compensating the reduced conductance of the bellow. The simulated transmission probabilities of a variety of different bellow lengths have been used to fit the parameters of the model. Each bellow geometry simulated with MolFlow + comprised two cylindrical tubes, one at each end, and the central bellow section. The geometry of each bellow is described by two quantities. The first one is the total length of the geometry model normalized to the inner diameter of the bellow. The second one is the fraction of the bellow section with regard to the total length. Simulating a tube instead of a long bellow reduced the simulation time by a factor of up to 1000, while the error introduced through the replacement of the bellow with a cylindrical tube of modified length was in most cases negligible.

#### 1. Introduction

Edge-welded bellows are important and common components of vacuum systems. Due to multiple scattering of gas particles inside the bellow elements, the transmission probability of gas particles through a bellow can be significantly reduced in comparison to a cylindrical tube of the same length. Therefore, when aiming for a realistic simulation of a vacuum system, edge-welded bellows can not be neglected without a loss of accuracy of the simulation. On the other hand, the inclusion of edge-welded bellows significantly complicates the simulation. In a typical Test Particle Monte Carlo (TPMC) vacuum simulation, the geometry of all components of the vacuum system is approximated e.g. by a triangle mesh. When comparing a cylindrical tube with a bellow of the same length, the latter increases the amount of triangles needed easily by a few orders of magnitude, therefore increasing the computation time of the simulation considerably. The increased number of hits of the walls inside the narrow bellow elements and the resulting longer trajectories until the particle leaves the bellow, add to the computation time, too. To our knowledge, empirical formulae for the transmission probabilities of edge-welded bellows have not been presented in the literature so far, while for the case of cylindrical tubes, empirical formulae for arbitrary tube lengths exist. The aim of this paper is to provide an empirical formula for the transmission probability of edgewelded bellows of arbitrary length. By using this formula, we propose a method for replacing edge-welded bellows with cylindrical tubes of appropriately adjusted length in vacuum simulations, which leads to a reduction of simulation time without a significant loss of accuracy.

The paper is organized as follows. In Sec. 2, we shortly explain the dynamics of rarefied gases, present analytic formulae for transmission probabilities for cylindrical tubes as given in literature and extend the discussion to the case of edge-welded bellows. In Sec. 3, we give a short introduction to the TPMC software MolFlow + which is used for our TPMC simulations. We proceed to describe the principal method and relevant parameters of our bellow simulations. In Sec. 4, we present the results of our simulations and discuss the dependence of the transmission probability on the bellow element angle. Additionally, we propose a new empirical model for the transmission probability of edge-welded bellows in dependence of their geometry parameters and analyze the increase in computation time when including edge-welded bellows in a vacuum simulation in comparison to a cylindrical tube. We complete the analysis with a proposal of a method for removing edge-welded bellows from vacuum simulations, therefore decreasing the time needed

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**Fig. 1.** CAD model of an edge-welded bellow. The edge-welded bellow is parametrized by the length *l* and diameter *d* of the tube, the length of the bellow segment  $l_B$ , and the height *h* and width *w* of a bellow element. The bellow angle  $\alpha_B$  is determined via the ratio h/w.

for a simulation, without losing the desired accuracy of the simulation.

#### 2. Dynamics of rarefied gases

# 2.1. Flow of rarefied gases through cylindrical tubes

The physical problem of the flow of rarefied gases through cylindrical tubes has been analyzed both theoretically and experimentally for a long time, e.g. by Knudsen who proposed analytic formulas for the conductance of cylindrical tubes and the gas rate that flows through them [1]. These formulas were corrected and improved by Smoluchowski [2] and Clausing [3]. The latter additionally derived an analytic formula of the transmission probability W of cylindrical tubes with arbitrary length, i.e. the probability of a gas particle to flow through the tube, in the form of an integral equation. For tubes with arbitrary length l and diameter d, this integral can not be solved analytically in closed form, but it can be solved through numerical integration. For special configurations, however, the integral is approximately solvable. For tubes that are very short compared to their diameter, Clausing found the analytic solution

$$W \approx 1 - \frac{l}{d} \qquad (l \ll d), \tag{1}$$

while for very long tubes with respect to their diameter, the well-known limit

$$W \approx \frac{4}{3} \frac{d}{l} \qquad (l \gg d) \tag{2}$$

applies. Berman found an analytical expression, approximating the solution for tubes of arbitrary lengths with good accuracy [4]. A mathematically simpler solution is given by the empirical formula

$$W = \frac{14 + 4\frac{l}{d}}{14 + 18\frac{l}{d} + 3\left(\frac{l}{d}\right)^2}$$
(3)

described in Ref. [5]. For arbitrary values of l/d the relative deviation from the analytic result is below 0.6%. This general form of the transmission probability is used in our proposed model for the description of edge-welded bellows. It is easy to verify that Eq. (3) converges to Eqs. (1) and (2) for the respective limits of l/d.

#### 2.2. Flow of rarefied gases through edge-welded bellows

The geometry of edge-welded bellows is significantly more complex than that of cylindrical tubes. Depicted in Fig. 1 is a computer-aided design (CAD) model of a typical edge-welded bellow, consisting of two cylindrical tubes on both ends and the actual bellow element in between the two tubes.<sup>1</sup> The geometry of the bellow is characterized by the total length of the tube l, the inner diameter of the tube d, the length  $l_B$  of the bellow element and the height h and width w of a bellow segment, as depicted in the figure. Via the trigonometric relation

$$\tan(\alpha_B) = \frac{2h}{w}, \tag{4}$$

the height of the bellow is directly connected to the opening angle  $\alpha_B$  of the bellow segment. Gas particles that enter the bellow typically scatter numerous times inside the bellow segments. Depending on the geometry parameters of the bellow, this leads to an increased beaming effect towards the walls of the bellow or towards the inlet of the tube. As a consequence, edge-welded bellows have a lower transmission probability for gas particles than cylindrical tubes of the same length and diameter. Therefore, bellows are important and non-negligible components of vacuum systems, especially when aiming for an accurate simulation of the system.

The integral formula for the transmission probability W as derived by Clausing was quite complicated even for the simple case of cylindrical tubes with arbitrary length. Due to the more complex geometry of edge-welded bellows, it is therefore not expected that an analytic formula for the transmission probability can be derived from first principles, and neither is it expected that such a formula would have a simple analytic solution. In order to quantify the transmission probability for a bellow, we use an *ansatz* that is motivated by the empirical form of Eqs. (1) and (3):

$$W = \frac{1 + c_1 \frac{l}{d}}{1 + c_2 \frac{l}{d} + c_3 \left(\frac{l}{d}\right)^2} \,.$$
(5)

The coefficients  $c_i$  are *a priori* undetermined and could be fixed by e.g. measurements or TPMC simulations. We want to point out that these coefficients depend on the other geometry parameters of the bellow. In the case of bellows used in actual vacuum systems, the bellow height and width are usually fixed. Hence, we choose the coefficients  $c_i$  to depend on the length  $l_B$  of the actual bellow element, only. As our analysis described in Sec. 4.1 shows, this is a valid approximation for typical bellows. In order to determine the parameters  $c_i$ , we simulated several variations of bellow geometries, as described in Sec. 3.2.

<sup>&</sup>lt;sup>1</sup> The geometry of hydroformed bellows, often used for systems where less flexibility is required, differs slightly from the triangular edge-welded CAD model presented in Fig. 1. In first approximation, the model is still valid, since the exact shape of the bellow element has only a minor influence on the transmission probability.

#### 3. TPMC simulations of edge-welded bellows

Before we describe the method and parameters of our bellow simulations, we first explain the basic concepts of the TPMC program MolFlow+. The publicly available C++ code has been developed and maintained by R. Kersevan et al. [7] and is hosted by CERN [8].

#### 3.1. Introduction to MolFlow +

The C++ code MolFlow+ is a TPMC program for the simulation of vacuum systems in the regime of molecular flow. The program allows both for the import of CAD geometries as well as the creation of models of vacuum systems inside MolFlow + itself. In the following, we only want to present the basic concepts of the program. A detailed description of the functionality of the software can be found in Ref. [9].

When importing a model of a vacuum system into MolFlow+, the program approximates the geometry of the model by dividing it into a polygonal mesh, with the individual elements of the mesh being called facets. For each facet, the user can define its individual properties, e.g. the sticking coefficient  $\alpha$  of the facet, its outgassing rate or the reflection behavior of the facet with respect to colliding gas particles. In the TPMC simulation, the trajectories of gas particles start from the facets that are chosen as gas sources, with a rate corresponding to the outgassing rate, and their path through the vacuum set-up is tracked via a ray-tracing algorithm. Collisions of rays with a facet correspond to the gas particle hitting the respective wall of the vacuum set-up. The simulation ends when the gas particle is absorbed with a probability of  $\alpha$  by the corresponding facet. Otherwise, it is emitted again as another ray, usually following a cosine law. Three counters are assigned to each facet, counting the desorption, hit, and absorption of particles. The two main quantities, generated by the TPMC simulation, that are necessary for our analysis are:

- **Desorptions** *N*<sub>D</sub>: Total number of gas particles that are emitted from a source facet. In our simulation, particles are only desorbed from the inlet of the bellow.
- **Absorptions**  $N_A$ : Total number of gas particles that are absorbed by a specific facet. In our simulations, the facets corresponding to the inlet and the outlet of the bellow have a sticking coefficient  $\alpha = 1$ , equivalent to an ideal absorber, while the rest of the facets have  $\alpha = 0$ .

#### 3.2. Method of the bellow simulations

For the TPMC simulation of the transmission probabilities with MolfFlow + we constructed models of edge-welded bellows with a large variety of geometry parameters, as exemplarily depicted in Fig. 1. The models were designed using the open-source software Blender [11]. The circular inlet facet was defined simultaneously as an ideal desorber with an angular distribution following the cosine law as well as an ideal absorber, i.e.  $\alpha = 1$ . The outlet facet is only an ideal absorber. This definition of the transmission probability model assumes that the tube is attached to large vacuum chambers at both ends. For all other facets, the sticking probability has been set to zero. This configuration corresponds to a set-up where gas particles enter the bellow only from one side, which is a valid approximation since we are only interested in the transmission probability of the bellow itself. The simulation of each bellow continued until the number of absorptions  $N_A$ by the outlet facet was of the order  $\mathcal{O}(10^5 - 10^6)$ , ensuring sufficient statistical accuracy. This quantity, together with the total amount of emitted gas particles  $N_D$ , allows the calculation of the transmission probability W by the formula

$$W = \frac{N_A}{N_D} \,. \tag{6}$$

20°C. Thus, the simulated transmission probabilities depend only on the geometry, and are independent of the temperature. The temperature is only introduced, when the conductance of a bellow is calculated as a product of the transmission probability and the incoming volumetric flow rate through the aperture of the inlet facet. The reflection of particles from the walls can be controlled by several parameters. The standard setting is diffuse reflection<sup>2</sup> with a thermal accommodation coefficient  $A_{\rm CC} = 1$ . After each hit of the wall, this leads to a new random direction following Knudsen's cosine law. With  $A_{CC} = 1$  the energy of the reflected particle only depends on the wall temperature and not from he incoming velocity. The velocity distribution of the thermalized particles is derived from a Maxwell-Boltzmann distribution. Details are described in Ref. [10]. The transmission probabilities of bellows described in this work have been simulated with diffuse reflection and full thermalization ( $A_{CC} = 1$ ) of the particles after each hit.

We performed two studies on the transmission probabilities of the bellows, as described in the following:

- 1. Qualitative analysis of the transmission probability  $W(\alpha_B)$  in dependence on the geometry parameters of the single bellow segments. For this study, we only considered the case of a pure bellow, i.e. we set  $l_B/l = 1.0$ . In a first step bellow segments with a fixed width w/d = 0.025 and two values of l/d = 2.0 and 5.0 were investigated. For these two values, the height h/d of the bellow segments was varied between the values  $h/d \in [0.000, 0.7161]$  in 13 steps, corresponding to opening angles  $\alpha_B \in [0^\circ, 89^\circ]$ . In a second step the width w/d for l/d = 5.0 bellows and three different heights h/d was varied as well. The results of this study are presented in Sec. 4.1.
- 2. Quantitative analysis of the transmission probability  $W(\alpha_B)$  in dependence of the total length l and the length of the bellow  $l_B$ . For this study, the geometry parameters of the bellow segments were fixed to the values w/d = 0.025 and h/d = 0.25, corresponding to an angle  $\alpha_B \approx 87.14^\circ$ . Both the total length of the tube  $l/d \in [0.5, 20.0]$  as well as the length of the bellow element compared to the total length of the tube  $l_B/l \in [0.0, 1.0]$  have been varied. For cylindrical tubes  $(l_B/l = 0)$  three additional lengths were simulated at l/d = 50, 500, and 5000, increasing the reliability of the fitted parameters to very long tubes. The results of this study are presented in Sec. 4.2.

# 4. Results of the simulations

In this section, we report the results of the simulations of the different geometries introduced in the previous section. Additionally, a procedure to remove edge-welded bellows from vacuum simulations is proposed, in order to save simulation time without introducing large errors due to the missing bellows.

# 4.1. Dependence of the transmission probability on the bellow angle

The range of angles  $\alpha_B$  of fully expanded and compressed bellow elements is very limited. For the full program of commercially available bellows in Ref. [6] the angles cover a range of  $\alpha_B \in [79.9^\circ, 89.2^\circ]$ . In general the opening angles are more restricted for larger heights h/d. For h/d > 0.5 the angles are larger than 82°.

The first study investigates the dependence of the transmission probability on the geometry of the single bellow segment. The simulated transmission probabilities  $W(\alpha_B)$ , together with their statistical error bars, are plotted in Fig. 2 as a function of the bellow angle  $\alpha_B$ , defined in Eq. (4). The upper plot of Fig. 2 shows the influence of the height h/d for a fixed width w/d = 0.025 and two values of the baffle

In the model we assumed isothermal conditions with all facets set to

 $<sup>^{2}</sup>$  MolFlow + allows to chose between diffuse and specular reflection. In the latest version these two can also be mixed.



**Figure 2.** Results of the study on angle dependence. **Top:** Shown is the transmission probability  $W(\alpha_B)$  in dependence on the bellow angle  $\alpha_B$  as defined in Eq. (4) for two distinct, fixed values of l/d. For large  $\alpha_B$ ,  $W(\alpha_B)$  is approximately linear with the bellow angle. The transmission probability for all  $\alpha_B \gtrsim 10^{\circ}$  is significantly smaller than the one for  $\alpha_B = 0$ . For values near  $\alpha_B \approx 55^{\circ}$ , the transmission probability reaches a minimum. The gray area indicates angle ranges for standard bellows [6], while the dashed line indicates the angle that was chosen for our quantitative analysis presented in Sec. 4.2. **Bottom:** This plot shows the variation of  $W(\alpha_B)$  for three different heights h/d for a single baffle length. The width w/d of the baffle segments varies with the  $1/\tan(\alpha_B)$ . The gray area takes into account that for h/d > 0.5 only bellows with  $\alpha_B > 82^{\circ}$  are available.

length l/d ( $l_B/l = 1.0$ ). The gray area indicates the typical range of standard bellow angles [6], mentioned above. While the absolute value of  $W(\alpha_B)$  differs for the two different lengths of the tube, the functional

dependence of *W* on  $\alpha_B$  is approximately the same for both lengths. The bottom plot of Fig. 2 shows the influence of the width  $w = 2 h/\tan(\alpha_B)$  on the transmission probability for one baffle length and several different heights h/d. A qualitative analysis of the plots allows us to formulate four statements:

- 1. The lower boundary  $\alpha_B = 0$  represents a cylindrical tube. In the vicinity of this value, the transmission probability is approximately independent of  $\alpha_B$  within a few percent.
- 2. For all values  $\alpha_B > 0^\circ$ , the transmission probability is lower than the one at  $\alpha_B = 0$ , showing that in general, a bellow has a significantly lower transmission probability than a cylindrical tube of the same length.
- 3. The transmission probability reaches a minimum for angles in the vicinity of  $\alpha_B \approx 55^{\circ}$ .
- 4. In the typical range for commercially available bellows (grey areas) W(α<sub>B</sub>) is approximately linear over α<sub>B</sub>. The slope, however, depends clearly on the height *h*/*d* of the bellow segment. Within the gray areas in Fig. 2, W(α<sub>B</sub>) varies only by about 2%.

Qualitatively, one can understand these observations by physical reasoning. Illustrated in Fig. 3 is a single bellow element with trajectories of gas particles for (a) a very small bellow angle, (b) a bellow angle  $\alpha_B \approx 55^\circ$  and (c) a steep bellow angle. In the illustration, the inlet of the bellow is at the left side. For small angles, the bellow approximately resembles a cylindrical tube. Hence, the scattering of the gas particles is effectively the same as in the case of a tube. There is no increased backward beaming, since the bellow element acts as a nearly diffuse reflector. This reflective behavior is depicted in gray by the reflection distribution, in this case approximately given by a cosine distribution. For very large angles, the gas particles scatter frequently inside the bellow elements. The net effect of these numerous scattering events is that the gas particles leave the bellow element with an increased beaming towards the opposite wall of the bellow, as indicated by the gray reflection distribution. Due to this, the transmission probability becomes almost independent of  $\alpha_{\rm B}$  for large angles, since the beaming towards the opposite side of the bellow does not influence  $W(\alpha_B)$  much. The large difference of transmission probabilities between large  $\alpha_B$  and  $\alpha_B = 0$  can be explained by the reduction of particle trajectories emitted towards the outlet of the bellow, which would otherwise add to the strong forward beaming effect of long cylindrical tubes. Finally, for  $\alpha_B \approx 55^\circ$ , the backwards beaming becomes maximal for the majority of bellow elements, leading to a minimal transmission



**Fig. 3.** Sketch of a bellow element for (a) very small bellow angles  $\alpha_B$ , (b) angles  $\alpha_B \approx 55^\circ$  where  $W(\alpha_B)$  becomes minimal and (c) very large  $\alpha_B$ . Gas particles, whose paths are depicted by arrows in red, enter the bellow from the left and scatter on the walls of the bellow with distribution functions of the diffuse reflection as depicted by the gray ellipses and arrows. In case (a), the bellow is effectively a cylindrical tube and gas particles reflect by the cosine law, while in case (c), multiple scattering inside each bellow segment effectively makes it a gas source with increased beaming towards the inside walls of the bellow. In case (b), the backwards scattering for incoming particles on the bellow elements is maximal, leading to a backwards beaming effect and a minimal transmission probability of the bellow. (For interpretation of the references to colour in this figure legend, the reader is referred to the Web version of this article.)



**Fig. 4.** (Top) Fit of Eq. (5) on the TPMC simulations of a cylindrical tube  $(l_B/l = 0)$ . The black circles represent the simulated transmission probabilities W(l/d), plotted over tube length l/d. The solid line represents the result of the fit. (Bottom) Fit of Eq. (5) on the TPMC simulations of a pure bellow  $(l_B/l = 1)$ . Like before, the black circles and the solid line represent the simulated transmission probabilities W(l/d) and the result of the fit, respectively. In both cases the dependence of the transmission probability on the length of the tube (bellow) is well described by Eq. (5).

probability for this angle, as indicated by the gray distribution function which is directed towards the entrance of the tube. Due to small variation of  $W(\alpha_B)$  in the angle region of standard bellows of around  $\pm 2\%$  and the long simulation time for a full coverage of all possible parameter variations, we restrict ourselves to one value of  $\alpha_B$  in the next section.

#### 4.2. New model for the transmission probabilites of bellows

For typical bellows, the geometry parameters are chosen such that  $w/d \ll 1$  and  $\alpha_B > 80^\circ$  [6]. With fixed values of w/d = 0.025 and h/d = 0.25, the transmission probability *W* can be fully parametrized as

a function of l/d and  $l_B/l$ . Fixing the values of w/d and h/d also fixed the bellow angle  $\alpha_B$  to the value indicated by the dashed line in Fig. 2.

For  $l_B/l = 0.0$  the transmission probability function  $W(l/d, l_B/l)$  for an edge-welded bellow should turn into Eq. (5) for a cylindrical tube, which we call  $W_0(l/d)$ . The parameters  $c_{1...3}$  in the formula are equivalent to the empirical result from Eq. (3). For the case of a pure bellow  $(l_B/l = 1)$  the same ansatz is used, called  $W_1(l/d)$ , but with different parameters  $c_{4...6}$ . The independent fits of the simulation results with Eq. (5) are shown in Fig. 4 for both cases.

In order to test the consistency of the simulated data with the empirical formula in Ref. [5] for a cylindrical tube, we plotted the simulation results for the case  $l_B/l = 0.0$  over the results of Eq. (3). The



**Fig. 5.** Comparison of the TPMC simulation results with the empirical formula in Eq. (3). The dots represent the simulated transmission probabilities  $W_{sim}$  with  $l_B/l = 0.0$  over the transmission probabilities  $W_{emp}$ , calculated with Eq. (3) for various  $l/d \in [0.5, 5000.0]$ . The straight line fitted to the points has a slope  $1.0013 \pm 0.0004$ , demonstrating the excellent agreement between Eq. (3) and the TPMC simulations for cylindrical tubes.



**Fig. 6.** Selection of simulation results for different bellow parameters. Each dot represents the simulated transmission probability *W* for a fixed set of geometry parameters l/d and  $l_B/l$ . The statistical errors of the simulation results are of  $\mathcal{O}(10^{-4})$ . Therefore, the error bars are not visible for most of the points. For fixed values of l/d the linear dependence of *W* on  $l_B/l$  is indicated by straight lines. The lines represent the fit results of Eq. (7), with the parameters  $c_i$  given in Table 1.

Table 1

Parameters  $c_i$  and their statistical errors of Eq. (7), fitted to the simulated transmission probabilities  $W(l/d, l_B/l)$ .

Parameter	Fit value	Standard error
c <sub>1</sub>	0.2915	0.0020
<i>c</i> <sub>2</sub>	1.3018	0.0026
<i>c</i> <sub>3</sub>	0.2225	0.0015
<i>c</i> <sub>4</sub>	0.2707	0.0069
c <sub>5</sub>	1.5267	0.0077
c <sub>6</sub>	0.2777	0.0077



**Fig. 7.** Effect of the modeling of the geometry of a bellow on the simulation time. Plotted is the time needed for the simulation of  $N_A = 10^6$  adsorptions on the outlet surface of the bellow in seconds for the entire range of  $l_B/l \in [0.0, 1.0]$  for fixed values of l/d = 4, w/d = 0.025 and h/d = 0.25. The simulation time increases by more than three orders of magnitude compared to a cylindrical tube ( $l_B/l = 0$ ).

resulting plot is shown in Fig. 5. The straight line fitted to the points has a slope  $1.0013 \pm 0.0004$ . This demonstrates the excellent agreement between the widely used approximation of Eq. (3) and our TPMC simulations for cylindrical tubes.

The dots in Fig. 6 represent the simulated transmission probabilities  $W(l/d, l_B/l)$  for a variaty of geometry parameters l/d and  $l_B/l$ . For constant values of l/d the simulation results show a linear drop between  $W_0(l/d)$  and  $W_1(l/d)$ . Thus, for arbitrary values of  $l_B/l \in [0, 1]$ , the transmission probability can be interpolated by a linear equation:

$$V(l/d, l_{B}/l) = W_{0}(l/d) + \frac{l_{b}}{l} \cdot (W_{1}(l/d) - W_{0}(l/d))$$

$$= \left(1 - \frac{l_{B}}{l}\right) \cdot \frac{1 + c_{1} \cdot \frac{l}{d}}{1 + c_{2} \cdot \frac{l}{d} + c_{3} \cdot \left(\frac{l}{d}\right)^{2}}$$

$$+ \left(\frac{l_{B}}{l}\right) \cdot \frac{1 + c_{4} \cdot \frac{l}{d}}{1 + c_{5} \cdot \frac{l}{d} + c_{6} \cdot \left(\frac{l}{d}\right)^{2}}$$
(7)

The full set of parameters  $c_{1...6}$  has been determined by fitting Eq. (7) to all of the simulated data. The results for the six parameters are presented in Table 1, together with their statistical errors. The straight lines in Fig. 6 have been calculated with Eq. (7). They connect the simulated points which share the same value of l/d. The individual simulations are in very good agreement with this empirical model. Due to the high number of simulated particles, the statistical error of each result is of  $\mathcal{O}(10^{-4})$ . For most of the points the error bars are too small to be visible. For values of l/d > 10 the number of different  $l_B/l$  simulations was restricted to fewer values, due to the rapidly increasing computation time. For the same reason the high statistics of adsorbed particles of  $\mathcal{O}(10^{6})$  could not be maintained for larger values of  $l_B/l$ . It had to be reduced to  $\mathcal{O}(10^{5})$  adsorptions. The simulation of the single point at  $W(l/d = 5000, l_B/l = 0)$  was stopped due to the long simulation time, after reaching a statistical uncertainty of better than 1%.

For  $l \gg d$  the ratio  $W_1/W_0$  of the transmission probabilities of a pure bellow and a cylindrical tube, parametrized by Eq. (7), converges against a constant value:

$$\frac{W_1(l \gg d)}{W_0(l \gg d)} = \frac{c_3 \cdot c_4}{c_1 \cdot c_6} = 0.74 \pm 0.03$$
(8)

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For cylindrical tubes the transmission probability  $W_0$  converges towards  $(1.322 \pm 0.013_{\text{stat}}) \cdot d/l$  for  $l \gg d$ , which agrees well with the literature value of Eq. (2). The transmission probability  $W_1$  for pure bellows converges towards  $(0.98 \pm 0.04_{\text{stat}} \pm 0.02_{\text{syst}}) \cdot d/l$  for  $l \gg d$ . The systematic error represents the spread of  $W_1$  in Fig. 2 for different values of h/d and w/d.

#### 4.3. Comparison of simulation time for bellows and tubes

In order to investigate the effect of accurately modeled edge-welded bellows in a TPMC simulation with respect to the simulation time, the geometry parameter set l/d = 4, w/d = 0.025 and h/d = 0.25 is used as a benchmark. For values  $l_{\mathbb{R}}/l \in [0.0, 1.0]$  with steps of 0.05, each bellow model was simulated until the number of adsorptions reached  $N_A = 10^6$ on the outlet surface of the bellow. The required simulation time is shown in Fig. 7. As can be seen in the logarithmic plot, the simulation time considerably increases with the length of the bellow section, represented by the fraction  $l_B/l$ . For each additional bellow element, the number of facets in the simulation increases, as well as the time a particle spends trapped inside the narrow bellow elements. This leads to a non-linear increase of the simulation time. For values of l/d > 10 with  $l_B/l \in [0.5, 1.0]$  a simulation was not practicable anymore with the same accuracy, since the time required to reach 10<sup>6</sup> adsorptions was too long. This analysis shows that bellows have a significant effect on the simulation time, which can rise by several orders of magnitudes.

## 4.4. Replacement of a bellow with an adapted cylindrical tube

Due to the substantial increase in CPU time when modeling an edgewelded bellow in a TPMC simulation, we suggest a method to optimize vacuum simulations without a reduction of the statistical significance of the result. The transmission probability W of a bellow in a vacuum system can be estimated by Eq. (7). Since the simulation of a cylindrical tube with the same transmission probability W as the bellow is much faster, one can replace the bellow by a slightly longer cylindrical tube with an effective length l'. The extended length would compensate the larger conductance of the tube with regard to the bellow. With the *ansatz*  $W = W(l/d, l_B/l) = W(l'/d, 0)$ , one finds for the effective length l'

$$\frac{l'}{d} = \frac{c_1 - c_2 W + \sqrt{(c_1 - c_2 W)^2 + 4 c_3 W (1 - W)}}{2 c_3 W} \,. \tag{9}$$

by inverting Eq. (7) for  $l_B/l = 0$ . Thus, for reducing the simulation time for vacuum setups containing edge-welded bellows, we propose the following approach:

- 1. For each bellow in the vacuum setup, calculate its approximate transmission probability W with Eq. (7), using the parameters in Table 1.
- 2. Calculate the effective length l' of a cylindrical tube with the same transmission probability *W* as the bellow, using Eq. (9).
- 3. Replace the bellow in the TPMC model with a cylindrical tube of length *l*'.

The proposed substitution changes the angular distribution of the trajectories of the particles leaving the bellow. In long tubes the well known beaming effect is more pronounced than in bellows. Therefore, this method is most accurate if the bellow connects to vacuum chambers with larger dimensions than the diameter of the bellow, where the beaming effect plays only a minor role in the subsequent distribution of the simulated particles.

The transmission time through a tube is faster than the transmission time through a bellow, where the particles spend more time in the crevices of the bellow segments. Most TPMC simulations investigate the state of a vacuum system that has already reached equilibrium. However, if the dynamic development of the pressure over time is an issue, the proposed method should be used with a grain of salt.

Apart from these two special cases the proposed substitution of a bellow by a cylindrical tube with an appropriate length l' can lead to a considerable reduction of the simulation time.

# 5. Conclusions

We have simulated the transmission probability W of edge-welded belows for a wide range of design parameters with the TPMC code MolFlow+. An empirical model was developed to describe the results in a concise way.

The analysis shows that for typical edge-welded bellows, the geometrical parameters of the individual bellow elements, or more specifically, the width and height of the bellow segments, and the related bellow angle  $\alpha_B$ , have only a minor influence on the transmission probability, adding a systematic uncertainty of 2%. In most applications its influence can be neglected, leading to a quantitative description, which only depends on the ratio of the length and diameter of the bellow.

The quantitative model of W, presented in this work, examined the more general case, where the bellow features cylindrical tubes at both ends. It is parameterized with the total length over diameter l/d and the relative length  $l_B/l$  of the bellow. For a fixed total length, W drops linearly with the increasing share  $l_B/l$  of the bellow section. For fixed  $l_B/l$ , the dependence of W on l/d follows the functional form of the commonly used empirical formula (Eq. (3)) of the transmission probability for a cylindrical tube. The results from a large number of simulations with different bellow and tube lengths have been used to determine the six parameters of the model.

For very long bellows the model converges towards a constant ratio  $W_1/W_0 = 0.74 \pm 0.03$  between the transmission probabilities of a bellow and a cylindrical tube. With the commonly used approximation  $W_0 = 4 d/3 l$  for very long tubes, the transmission probability of long bellows can be estimated as  $W_0 = d/l$  within the errors of the simulations.

In an attempt to reduce the prohibitively long TPMC simulation times for long bellows, a method is proposed, where the empirical model is used to calculate the appropriate length of a cylindrical tube, which replaces the bellow in the TPMC model. This method can reduce the simulation time by several orders of magnitude, while preserving the statistical significance of the simulation.

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