

Resonant Self-Actuation Based on Bistable Microswitching

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Abstract: We present the design, simulation, and characterization of a magnetic shape-memory alloy (MSMA) film actuator that transitions from bistable switching to resonant self-actuation when subjected to a stationary heat source. The actuator design comprises two Ni-Mn-Ga films of 10 μm thickness integrated at the front on either side of an elastic cantilever that moves freely between two heatable miniature permanent magnets and, thus, forms a bistable microswitch. Switching between the two states is induced by selectively heating the MSMA films above their Curie temperature T_c . When continuously heating the permanent magnets above T_c , the MSMA film actuator exhibits an oscillatory motion in between the magnets with large oscillation stroke in the frequency range of 50–60 Hz due to resonant self-actuation. A lumped-element model (LEM) is introduced to describe the coupled thermo-magnetic and magneto-mechanical performance of the actuator. We demonstrate that this performance can be used for the thermomagnetic energy generation of low-grade waste heat ($T < 150^\circ\text{C}$) with a high power output per footprint in the order of $2.3 \mu\text{W}/\text{cm}^2$.

Keywords: bistable actuator; thermomagnetic generator; energy harvesting; energy conversion; heat transfer; ferromagnetism; Heusler alloys; magnetic shape-memory alloys; Ni-Mn-Ga; thin films



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1. Introduction

Magnetic shape-memory alloys (MSMAs) are multifunctional materials that offer both thermal shape-memory, magnetic properties, as well as various multiferroic coupling effects [1]. The unique features of MSMAs are due to pronounced magnetic ordering and large magnetocrystalline anisotropy, resulting in an energetically preferred orientation of magnetic moments. In addition, MSMAs undergo a first-order martensitic phase transformation, which involves a large abrupt change in both the structural and magnetic properties. The phase transformation proceeds in a reversible manner, enabling a thermal shape-memory effect as well as superelasticity. The presence of coupled ferromagnetic and metastable ferroelastic (martensitic) domains in MSMA materials gives rise to multiferroic coupling effects, such as magnetic field-induced reorientation and magnetic shape-memory effect [2]. A prominent example is the Heusler alloy Ni-Mn-Ga [2–4], which is ferromagnetic at room temperature. Metamagnetic SMAs, such as the quaternary Heusler alloys Ni-Mn-X-Y ($X = \text{Ga, In, Sn}$; $Y = \text{Co, Fe, Al}$), are non-magnetic or antiferromagnetic in the martensitic state at low temperatures and undergo an abrupt change of magnetization with narrow hysteresis at the first-order transformation to the austenitic state [5].

Due to these properties, MSMAs exhibit different modes of combined sensing and actuation capabilities [6,7]. As a consequence, MSMA actuators may be designed as a single piece of material but may still perform various functions. This option is particularly interesting for applications in small dimensions, where the effect size, multifunctionality, and ease of fabrication are important factors. Previous approaches to the development

of miniature-scale MSMA actuators include the thinning of bulk single crystals to foil specimens [8,9], the deposition of MSMA films [10–12], as well as 3D printing [13]. So far, several actuation mechanisms have been exploited at miniature scales. MSMA linear actuators have been developed using the magnetic field-induced reorientation effect in single crystalline Ni–Mn–Ga [14–17]. A bidirectional microactuator has been developed consisting of a polycrystalline Ni–Mn–Ga double-beam cantilever, which generates antagonistic bending forces near a miniature permanent magnet due to an electrical current-induced transition between ferromagnetic martensite and paramagnetic austenite. Thus, pulsed heating causes fast oscillatory motion of the cantilever in microscanner applications [18]. The abrupt change of magnetization in the first-order phase transformation in metamagnetic SMA films has been considered in [19,20] to enable thermally induced switching of magnetic forces.

Another field of application of MSMA actuation is thermomagnetic generation (TMG) at miniature scales [21–25]. Film-based TMGs have been developed using metamagnetic [22] and ferromagnetic SMA films [23–25]. The MSMA film actuator was integrated at the front of a freely movable cantilever near a heatable permanent magnet serving at the same time as the magnetic field and heat source. In this case, due to the large surface-to-volume ratio, rapid heat transfer is achieved through direct mechanical contact between the MSMA film and heat source, enabling a short duration of the thermal duty cycle in the order of the mechanical eigenoscillation. When heating the permanent magnet above the Curie temperature T_c , this performance gives rise to resonant self-actuation, which is characterized by a large cantilever deflection and a high oscillation frequency [24,25]. Recent demonstrators based on Heusler alloy films of Ni–Mn–Ga showed oscillation frequencies in the order of 100 Hz. By integrating a pick-up coil at the cantilever front, electrical power densities of up to 120 mW/cm^3 have been generated [24,25].

In this investigation, we extend this approach by integrating two MSMA film actuators on opposite sides of the cantilever front and using two heatable permanent magnets above and below the cantilever to form a bistable actuation system. Bistable systems are of special interest both for actuation, as energy is only consumed in case of switching, and for energy harvesting, as the power output could be enhanced by the additional bistable forces. A lumped-element model (LEM) is presented to determine the magneto-mechanical performance of the new actuator design as a function of the MSMA film temperature, which complements the experimental study of the dynamic mechanical actuator performance.

We demonstrate that the MSMA film actuator shows bistable switching between two stable end positions due to thermally induced switching of magnetic latching forces, which transitions to resonant self-actuation in between the end positions when continuously heating the permanent magnets above T_c . We report on large oscillation strokes in the order of 20% of the cantilever length at oscillation frequencies of 50–60 Hz, which allows for thermomagnetic generation with a power per footprint of up to $2.3 \mu\text{W/cm}^2$.

2. Material Properties

Sputtering is the most favorable deposition method to obtain thick films on a large scale, as is required for MEMS technology [6,12]. The resulting film structure depends on various parameters including the substrate, deposition temperature, sputtering power, and annealing conditions. Consequently, the as-prepared films can exhibit a polycrystalline structure [26–28] or an epitaxial relation to the substrate [10,29]. A process has been developed for the fabrication of freestanding polycrystalline Ni–Mn–Ga films showing 10 M martensite at room temperature [26,30]. Here, the MSMA films are fabricated by magnetron sputtering with the target composition of $\text{Ni}_{49.5}\text{Mn}_{28}\text{Ga}_{22.5}$. Film deposition is performed on polyvinyl alcohol (PVA) substrates that can be dissolved after sputtering. A sputtering power of 200 W is used and the sputtering time is adjusted to obtain a homogenous film thickness of 10 μm . Argon gas flow is maintained at $230 \text{ mm}^3 \text{ s}^{-1}$, and the substrate temperature is kept at 50 °C. Under these conditions, the as-deposited films are amorphous and require heat treatment to adjust the crystal structure and phase transformation proper-

ties. The effect of heat treatment on the performance of polycrystalline Ni–Mn–Ga films is described in [26]. The final composition after heat treatment of 10 h at 800 °C is determined by the inductive plasma method to be Ni_{51.4}Mn_{28.3}Ga_{20.3}.

Our investigations showed that the sputtering power can be used as a means to control the chemical composition of the films. Therefore, the Ni and Mn contents of the films decrease with increasing sputtering power for both targets, whereas the Ga content increases. However, little change in composition is observed for different heat treatment temperatures between 650 and 1000 °C and heat treatment times between 1 h and 10 h. A temperature-dependent measurement of the electrical resistance is shown in Figure 1. The measurement reveals the typical features of a first-order martensitic phase transformation, namely, a jump-like drop upon heating and a hysteresis upon subsequent cooling. The finish temperatures of the martensitic and austenitic transformations, M_f/A_f , are determined to be $-20/25$ °C. The measurement also exhibits a characteristic kink at 98 °C due to the ferromagnetic transition. Typical temperature-dependent magnetization characteristics are shown in the Appendix A (Figure A1). When heating the Ni–Mn–Ga films above T_c , the magnetization drops to zero, which will be used in the following to control the magnetic attraction forces.

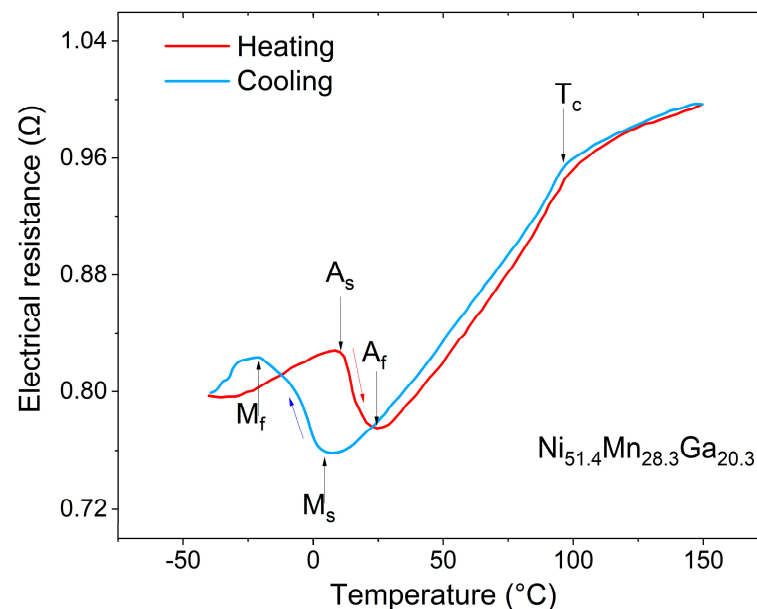


Figure 1. Four-wire electrical resistance measurements of the Ni–Mn–Ga films fabricated by magnetron sputtering. Legend: $A_{s/f}$ —austenite start/finish temperature, $M_{s/f}$ —martensite start/finish temperature, T_c —Curie temperature.

3. Layout and Operation Principle

The bistable actuator comprises two MSMA films that are mounted at the tip of a free-standing cantilever on either side, as shown in Figure 2a. Furthermore, two permanent magnets are arranged above and below the cantilever, providing magnetic fields to magnetize the MSMA films and magnetic field gradients to induce magnetic attraction forces on the MSMA films. These magnetic forces can be varied in a wide range by increasing/decreasing the temperature of the MSMA films above/below their Curie temperature T_c . The magnets are placed at an angle on both sides of the cantilever to achieve maximum contact. The coupling of the elastic and magnetic forces, F_{el} and F_m , in the system is illustrated schematically in Figure 2b. This coupling gives rise to bistable performance, i.e., the system can adopt two stable deflection states, whereby external energy is required to switch between these states. When providing sufficient external energy in the form of mechanical vibration, for instance, the cantilever tip would oscillate between the two deflection states. Here, we consider the supply of thermal energy.

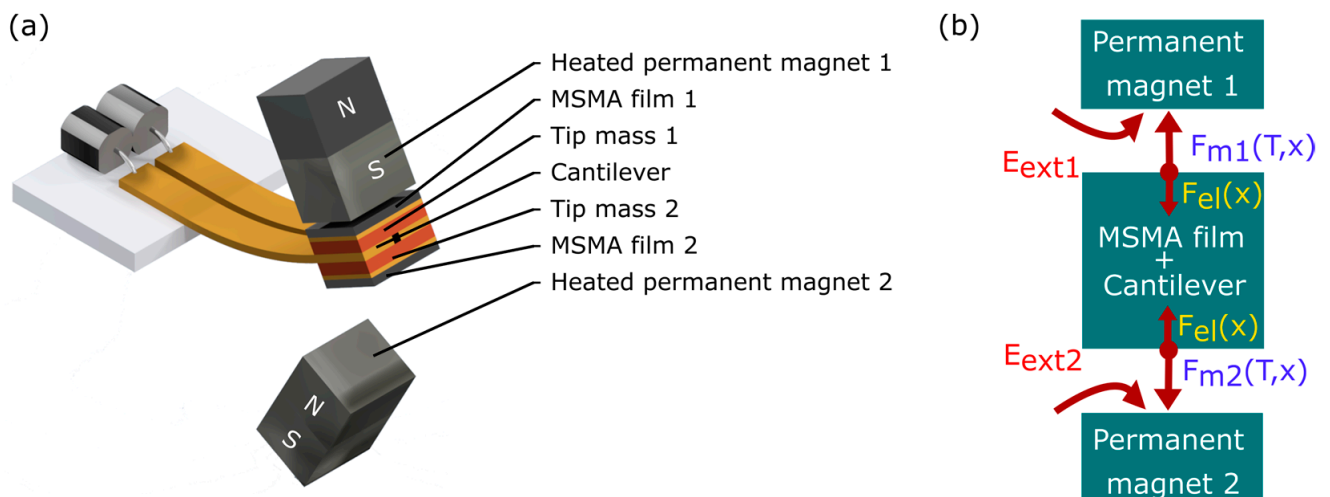


Figure 2. (a) Schematic of the bistable MSMA film actuator consisting of a free-standing cantilever, a tip mass, and an MSMA film on both sides of the cantilever allowing for two stable end positions; (b) schematic of the film actuator system and the elastic and magnetic forces, F_{el} and F_m , that act on the freely movable cantilever causing bistable performance. Switching between the two stable deflection states requires external energy E_{ext} .

One operation mode is bistable switching by the selective heating of the MSMA films. At room temperature, the MSMA films are in a ferromagnetic state and, therefore, the cantilever is attracted to either of the two permanent magnets. Any asymmetry results in a preferred initial deflection state. Selective heating of the MSMA film being in contact with its corresponding magnet above T_c causes a reduction in magnetization and a decrease in the magnetic attraction force F_m until it drops below the restoring force of the cantilever F_{el} . In this case, the contact is released and the cantilever deflects back towards its initial straight position. Due to inertial forces, the cantilever overshoots and is attracted to the opposite permanent magnet. Due to symmetry, resetting occurs when selectively heating the opposite MSMA film.

The second operation mode is a thermally induced oscillation between the two deflection states. In this case, both MSMA films are heated alternately in a periodic manner, which is easily accomplished by continuously heating the permanent magnets and maintaining their temperature above T_c . Then, the heating of the MSMA films occurs by direct contact with the heated magnets. Periodic heating and cooling of the MSMA films cause an oscillatory motion of the cantilever, as long as the thermal energy supply is maintained.

The cantilever is fabricated by laser micromachining of a CuZn film of 20 μm thickness. The inertial forces in the system are adjusted by providing additional tip masses on either side of the cantilever. The two MSMA films of Ni-Mn-Ga are attached on top of the respective tip masses, allowing for thermal decoupling from the cantilever beam. A photo of the final device is presented in the Appendix A.

4. Lumped Element Modelling of Performance

A lumped element model (LEM) in Matlab SIMSCAPE 2022b is introduced to describe the coupled magneto-thermo-mechanical coupling within the device and to optimize the performance parameters. If a small lateral displacement of the cantilever is neglected, the bending motion of the cantilever can be assumed to be one-dimensional. Thus, a spring mass damper system can be used to represent the dynamic motion of the microactuator. At the miniature scale, the gravitational force on the system can be neglected as well. Thus,

the effective net force F_{net} acting on the movable masses mi of the cantilever tip is described by the force balance as follows:

$$F_{net} = c\dot{x} + kx + \sum_{i=1}^2 m_i \ddot{x}_i + F_{mi} \quad (1)$$

whereby mi is approximated by

$$m_i = \frac{33}{140} m_{cant} + m_{fi} \quad (2)$$

The parameters m_{cant} and m_{fi} represent the masses of the cantilever and the magnetic SMA films, respectively. c and k denote the damping coefficient and the spring constant of the structure, respectively. The attraction force between the magnetic SMA films and the permanent magnets is denoted by F_{mi} , which is a function of both position and temperature.

$$F_{mi} = V_m M_{Ti} \frac{\partial B_{xi}}{\partial x_i} \quad (3)$$

Thereby, V_m denotes the volume of the MSMA films that are magnetized by the applied field of the permanent magnet. M_{Ti} is the magnetization of the films as a function of temperature and magnetic field, which is mapped experimentally as a function of the position of the two films. The position dependencies are used to determine the magnetic field gradients $\frac{\partial B_{xi}}{\partial x_i}$ analytically [31]. Since there are two magnets, the magnetic fields and field gradients are superimposed to calculate the resulting magnetic field at a given position.

The heat transfer at contact and the thermal dissipation during actuation are modeled by using thermal equivalent circuits. A Cauer model is utilized for representing different components of the device as a combination of thermal resistance and thermal capacitance [32]. Figure 3 shows a schematic of the thermal network of the bistable magnetic SMA microactuator device. The overall impedance of the system governs its heat intake and dissipation. This is given by Equation (4), where Z_{th} is the overall thermal impedance of the device, $R_{th(i)}$ is the thermal resistance, and $C_{th(i)}$ ($i = 1, 2, 3 \dots n$) is the thermal capacitance of the individual components, respectively. In order to obtain the thermal gradient within the individual components, each component is represented by a combination of several thermal resistances and capacitances with a resolution proportional to the number of resistances and capacitances representing one component. Depending on the dimensions of each component and thermal properties, the heat flow and dynamic temperature change are obtained [32].

$$Z_{th} = \frac{1}{C_{th(1)} + \frac{1}{R_{th(1)} + \frac{1}{C_{th(2)} + \frac{1}{R_{th(2)} + \frac{1}{\dots C_{th(n)} + \frac{1}{R_{th(n)}}}}} \quad (4)$$

Coupling both the thermal circuit model and the mechanical spring damper model allows for the time-resolved simulation of the thermal performance, as well as the dynamic actuation of the bistable device. In order to compare different multistable nonlinear systems and quantify their stability, a model-based description of the involved energies is appropriate. This allows the determination of available stable states at the energy minima and the energy barrier that needs to be overcome in order to switch between the states.

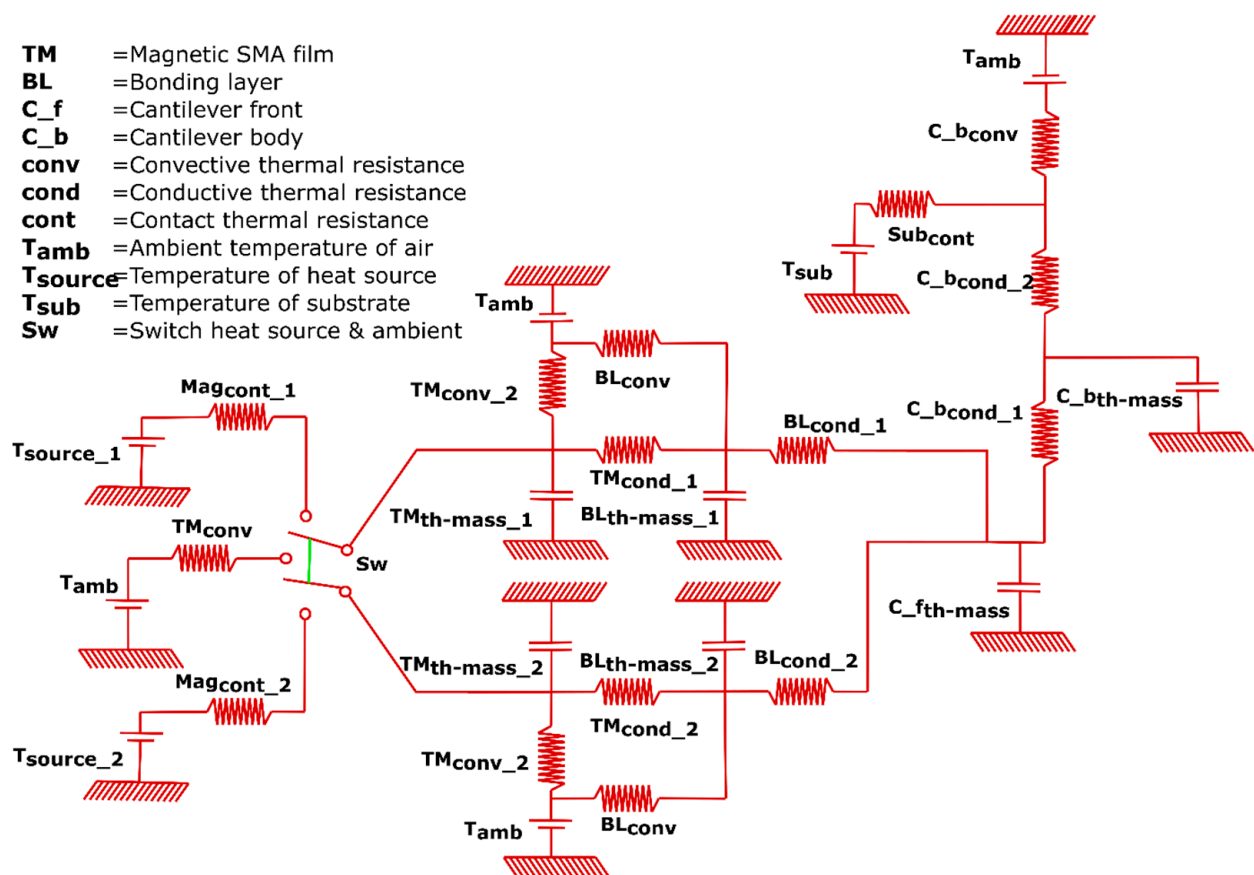


Figure 3. Thermal lumped element model of the bistable MSMA film actuator. Both MSMA films reject most of the heat transferred during contact with the heat source into the cantilever by conduction. Thus, the cantilever acts as the main heat sink. A minor fraction of the heat is dissipated by convection to the ambient air during actuation.

5. Resonant Self-Actuation

When continuously heating the permanent magnets above T_c , the cantilever undergoes an oscillatory motion, which is governed by the interplay of magnetic attraction, elasticity, inertia, and damping forces. In particular, the dynamics of heat transfer have an important influence on the oscillation performance. If heat transfer from the heated magnet to the MSMA film is slower than the duration of the cantilever eigen oscillations, the cantilever will rest at the magnet and possibly perform damped eigen oscillations until the magnetic attraction force decreases enough after a sufficient heating time of the MSMA film to retract from the magnet. On the other hand, if the cooling of the MSMA film is too slow to be attracted to the magnet surface, it will undergo chaotic oscillations without touching the magnet until the attraction force of one of the magnets is sufficiently large. In the optimum case, however, matching the time constants of heat transfer and mechanical oscillation will lead to resonant self-actuation [33]. In the present actuator design, the dimensions of the MSMA films are adjusted for a sufficiently large surface-to-volume ratio, and the eigenfrequency of the moving mass is tuned to meet the matching conditions. Once the coupled thermo-magneto-mechanical cycles are balanced, resonant self-actuation occurs, which is characterized by a continuous periodic oscillation with large strokes and high frequencies. Figure 4a shows the simulated time-resolved deflection of the cantilever tip in resonant self-actuation mode. The corresponding time-resolved changes in the MSMA film temperatures are shown in Figure 4b.

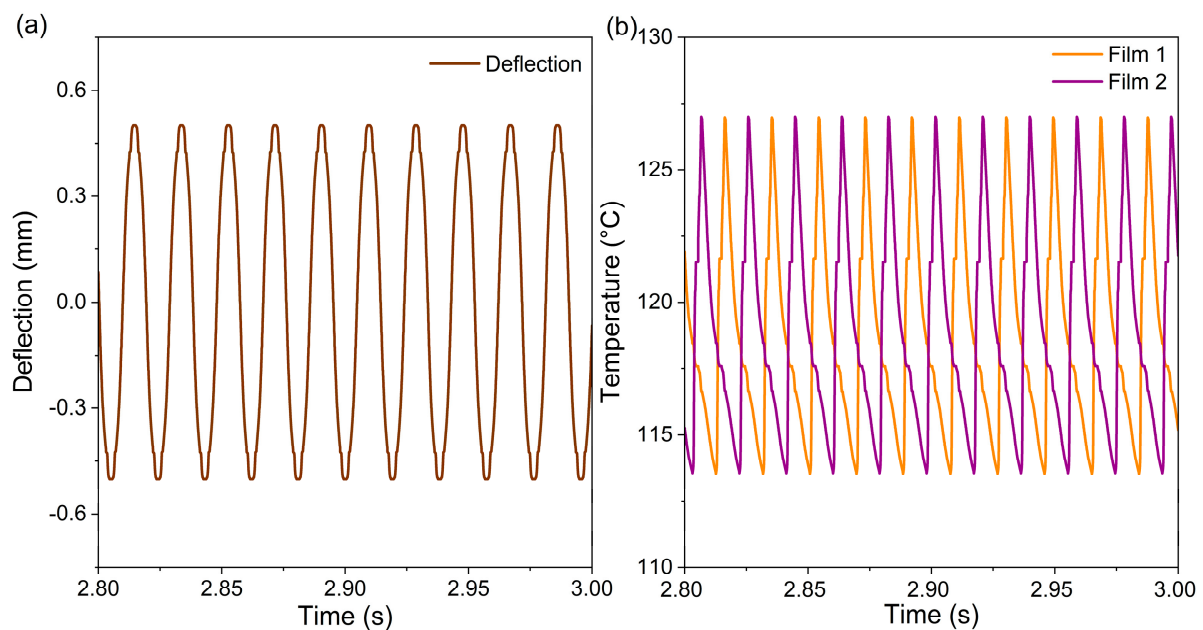


Figure 4. (a) LEM simulation of the time-resolved deflection of the cantilever tip of the bistable MSMA film actuator in resonant self-actuation mode; (b) time-resolved temperature change of the MSMA films 1 and 2 during resonant self-actuation.

Due to the symmetry of the bistable MSMA film actuator, the cantilever tip performs equal deflections in positive and negative directions. As the MSMA films are heated alternately while in contact with the permanent magnets, they undergo separate thermal cycles with a phase shift of 180 degrees for each mechanical cycle. The experimental results on the time-dependent performance of the bistable MSMA film actuator are presented in Section 7. Due to the constraints of the experimental setup, time-resolved measurements of the actuator deflections were not possible. Therefore, the simulation model was validated using temperature-dependent electrical measurements (see Section 7).

6. Transition from Bistable Switching to Resonant Self-Actuation

Once the permanent magnets are continuously heated above T_c , the bistable MSMA film actuator undergoes a transition from bistable switching to resonant self-actuation. Next, this transition is analyzed via LEM simulations of the evolution of magnetic attraction forces and the corresponding potential energies.

The magnetic attraction forces acting on the MSMA films 1 and 2 (F_{m1} and F_{m2}) result in an effective force $\langle F \rangle$ acting on the cantilever tip. Figure 5a shows the simulated change in the effective magnetic attraction force $\langle F \rangle$ versus cantilever deflection during the initial oscillation cycles before the MSMA film actuator reaches stationary operation conditions. In the first actuation cycle, the microactuator switches from the initial stable state at -0.5 mm to the second stable state at 0.5 mm, where it rests until selective heating of the MSMA film at contact with the magnet causes the microactuator to switch back to the initial position in the second actuation cycle. Within the first 16 cycles, the system transitions from the initial bistable switching mode to the resonant self-actuation mode. Thereby, the effective force maxima $\langle F_{max} \rangle$ at the permanent magnets decrease from ± 6.5 mN to about ± 2.5 mN. This reduction is caused by an increase in the average temperatures of the MSMA films until they reach stationary values. Figure 5b shows the magnetic attraction forces F_{m1} and F_{m2} , as well as the resulting effective magnetic force $\langle F \rangle$ under stationary operation conditions.

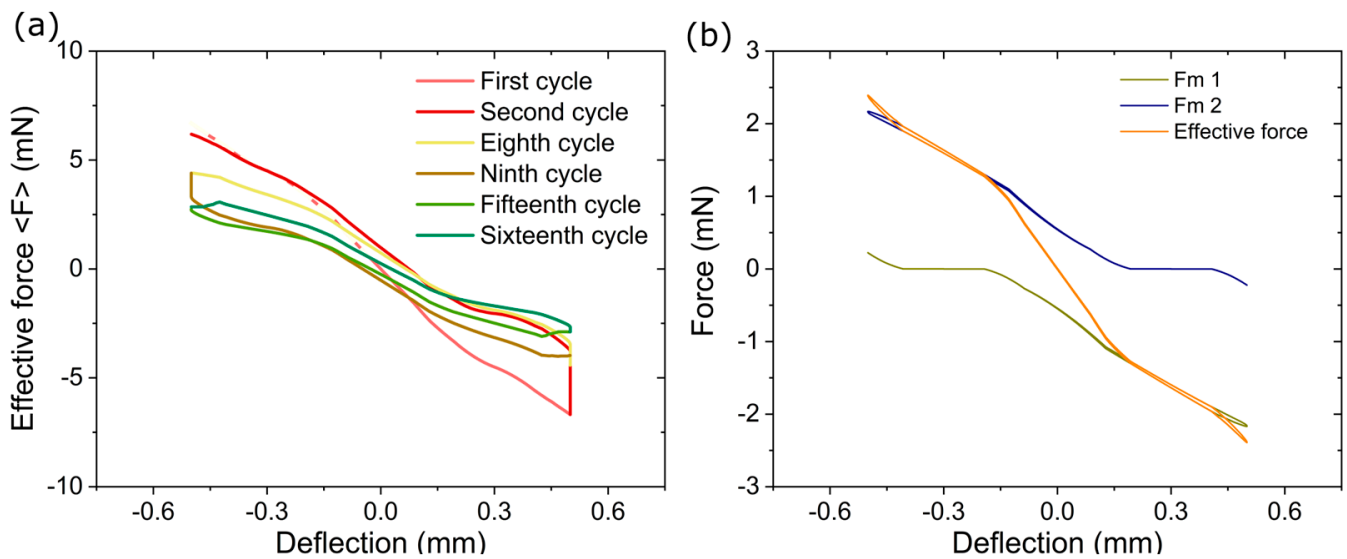


Figure 5. (a) LEM simulation of the effective magnetic attraction force $\langle F \rangle$ on the cantilever tip of the bistable MSMA film actuator versus deflection of the cantilever tip during the initial phase of operation for the selected actuation cycles as indicated. Stationary operation conditions are reached after about 16 actuation cycles; (b) simulated course of magnetic attraction forces F_{m1} , F_{m2} and the resulting effective magnetic force $\langle F \rangle$ in stationary condition.

Figure 6 shows the corresponding simulated changes in the potential energies, whereby the total potential energy comprises the contributions of the elastic potential energy of the cantilever and the magnetic potential energies of the two magnetic subsystems. Figure 6a depicts the change in the total potential energy versus deflection of the cantilever tip during the initial oscillation cycles before the MSMA film actuator reaches stationary operation conditions. The dotted line in Figure 6a indicates the symmetry of the total potential energy in the initial state. In the simulation, the initial position is at zero deflection in the center between the permanent magnets, where the potential energy shows a pronounced maximum of about $10 \mu\text{J}$. In addition, two potential minima occur at the surfaces of the two permanent magnets, which correspond to the two stable deflection states of the actuator. Without a continuous supply of thermal energy, the MSMA film actuator would deflect towards one of the two deflection states and rest. The continuous supply of thermal energy during the contact between the MSMA films and their respective permanent magnets strongly affects the potential energies. Within the first 16 cycles, the maximum potential energy decreases considerably to less than $3 \mu\text{J}$. Simultaneously, two pronounced potential energy minima evolve on either side of the actuator center, whereby they become increasingly pronounced and propagate towards the actuator center until they become stationary at about $\pm 2 \mu\text{m}$. This reduction is caused by an increase in the average temperatures of the MSMA films, which affects the magnetic forces (Figure 5) alike.

Figure 6b shows the potential energies of both magnetic subsystems, the elastic energy, and the resulting total energy versus deflection once stationary conditions are reached. In this case, the total energy exhibits two energy minima well apart from the maximum possible deflection states and a small energy barrier in the actuator center, which is sufficiently low to enable continuous oscillation. Continuous oscillations occur as long as the kinetic energy of the moving mass exceeds the barrier height. Therefore, a critical upper limit of the elastic potential energy exists, above which the energy barrier becomes too high for resonant self-actuation.

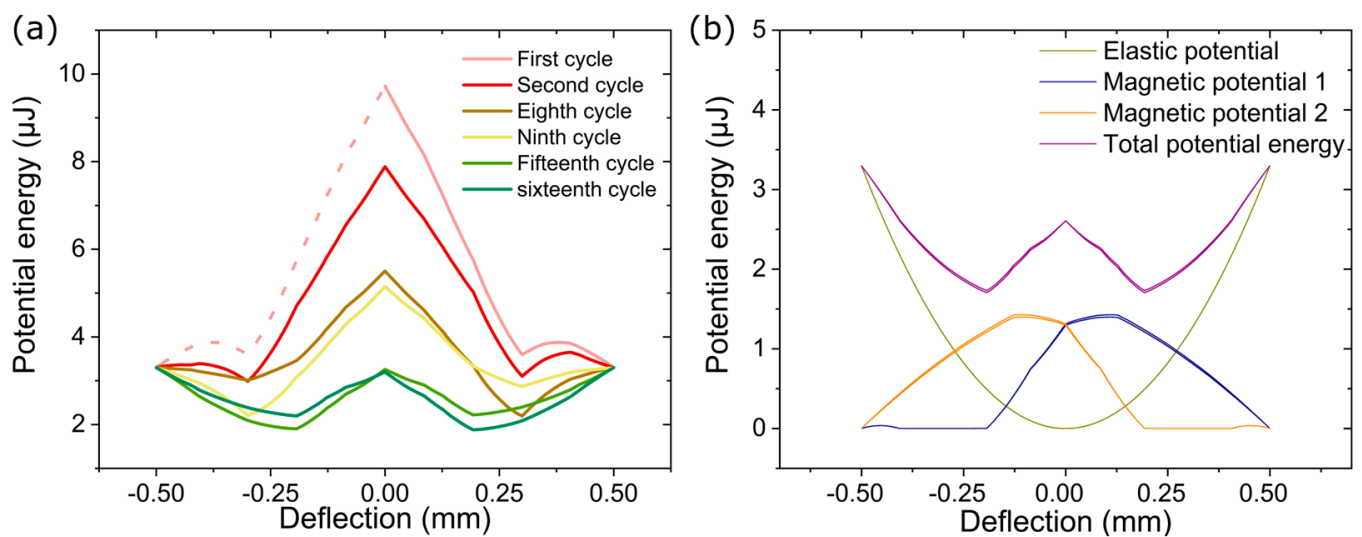


Figure 6. (a) LEM simulation of the total potential energy versus deflection of the cantilever tip of the bistable MSMA film actuator during the initial phase of operation for selected actuation cycles as indicated. Stationary operation conditions are reached after about 16 actuation cycles; (b) simulated course of potential energies of both magnetic subsystems, the elastic energy, and the resulting total energy in stationary condition after the sixteenth cycle.

7. Bistable Thermal Energy Harvesting Based on Resonant Self-Actuation

The bistable MSMA film actuator is highly attractive for thermal energy harvesting, as the operation mode of resonant self-actuation enables an optimum conversion of thermal energy into kinetic energy of mechanical eigenoscillation. By integrating two pick-up coils, 1 and 2, next to the MSMA films 1 and 2, respectively, instead of the tip masses, a symmetric energy harvester design is obtained. The core of the pick-up coil is made of copper to allow thermal conduction between the MSMA film and the cantilever. The outer sides of the coils are made of brass. The dimensions of the Cu core and brass layers are tailored for optimum heat transfer, following the guidelines given in [33]. To simulate the energy harvesting performance, the LEM is extended to include the electromagnetic coupling of the movable pick-up coil following the approach in [34]. In this case, the time-dependent performance of the bistable MSMA film actuator can be determined via LEM simulations and electrical measurements, which allows the validation of the simulation model.

Figure 7a,b show experimental and simulated electrical performance characteristics of the bistable thermal energy harvester, respectively, for a heat-source temperature of 150 °C. The output current is determined by connecting the two pick-up coils in parallel. The output occurs in pairs of large and small peaks due to the changing direction of the magnetic field gradient in the actuation center. The simulation model reproduces the current amplitude and frequency of the bistable MSMA film actuator very well. Further experimental and simulation results on the electrical current output at higher heat source temperatures of 160 and 170 °C are presented in the Appendix A (Figure A2), which further validate the LEM simulations. The achieved average power of a single device is 0.23 μW at a heat source temperature of 150 °C and a load resistance of 1000 Ohms, which corresponds to a power per footprint of about 2.3 μW/cm².

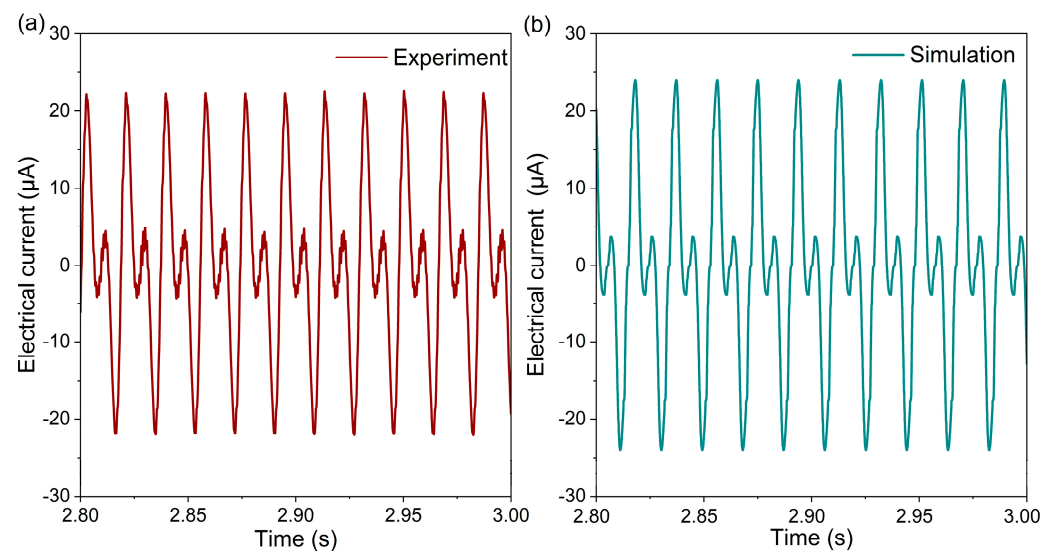


Figure 7. Performance of the bistable thermal energy harvester for a heat source temperature of 150 °C. (a) Experimentally determined electrical current in the pick-up coils connected in parallel; (b) corresponding LEM simulation.

8. Conclusions

A bistable magnetic shape-memory alloy (MSMA) film actuator is presented consisting of two Ni-Mn-Ga films at the front, on either side of a freestanding cantilever located between two heatable miniature permanent magnets. By selective heating of the MSMA films, the actuator operates as a bistable switch, whereby the cantilever can be switched between two deflection states at the two magnets. Continuous heating of the permanent magnets above their Curie temperature T_c gives rise to a thermally induced oscillation between the two deflection states. We demonstrate that the cantilever oscillation can be optimized by matching the time constants of the heat transfer and mechanical oscillation to achieve resonant self-actuation. A lumped-element model (LEM) is presented to determine the coupled thermal and magneto-mechanical performance of the bistable MSMA film actuator. Using LEM simulations, we investigate the transition from bistable switching to the resonant self-actuation mode. At low MSMA film temperatures, the magnetic potential of the system is higher compared to the elastic potential, and thus, the system rests in a stable deflection state at one of the two magnets. Once the MSMA film temperatures exceed T_c during contact with the permanent magnets, the actuator starts oscillating between the heated magnets. Therefore, the average temperature of the MSMA film increases, resulting in a decrease in magnetization, and a corresponding decrease in the magnetic attraction force and in the magnetic potential energy. Under stationary conditions, the total energy of the system exhibits two energy minima separated by a small energy barrier in the actuator center, which is low enough to enable continuous oscillation. In resonant self-actuation mode, the large oscillation stroke in the order of 20% of the cantilever length and high oscillation frequency at 50–60 Hz enable thermal power generation of 2.3 $\mu\text{W}/\text{cm}^2$ at a heat source temperature of 150 °C.

9. Patents

The authors hold a patent related to this work.

Author Contributions: Conceptualization, M.K.; design, M.K. and J.J.; fabrication, J.J., H.M. and M.O.; software and model validation, J.J.; investigation, J.J., H.M. and M.O.; writing—original draft preparation, J.J.; writing—review and editing, M.K. and J.J.; visualization, J.J.; supervision, project administration, and funding acquisition, M.K. All authors have read and agreed to the published version of the manuscript.

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Data Availability Statement: The data that support the findings of this study are available from the corresponding authors upon reasonable request.

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Conflicts of Interest: The authors declare no conflict of interest.

Appendix A

Appendix A.1. Simulation Parameters

Table A1. Summary of LEM simulation parameters. The parameters depend on the MSMA material and the detailed operation conditions of the TMG device. Table A1 also shows specific values used for LEM simulation of a TMG demonstrator device using Ni-Mn-Ga films of 10 μm thickness with a Curie temperature T_c of 371 K.

Parameter	TMG Device Based on a Ni-Mn-Ga Film	Reference
Length of each MSMA film	2 mm	This work
Width of each MSMA film	2 mm	This work
Thickness of each MSMA film	10 μm	This work
Density of each MSMA material	8020 kg/m^3	[35]
Length of the cantilever beam	5 mm	This work
Width of the cantilever beam	2 mm	This work
Thickness of the cantilever beam	20 μm	This work
Density of cantilever material	8500 kg/m^3	[36]
Length of bonding layer	2 mm	This work
Width of bonding layer	2 mm	This work
Thickness of bonding layer	10 μm	This work
Density of bonding layer	1250 kg/m^3	[37]
Young’s modulus of cantilever material (E)	1×10^{11} N/m^2	[38]
Contact stiffness beam-magnet (k_{cont})	1×10^4 N/m	This work
Structural damping (c)	1.12×10^{-5} Ns/m	This work
Impact damping (c_{cont})	0.1 Ns/m	This work
Thermal conductivity of MSMA material	23.2 W/mK	[39]
Specific heat capacity of MSMA material	490 J/kgK	[39]
Thermal conductivity of cantilever material	109 W/mK	[40]
Specific heat capacity of cantilever material	400 J/kgK	[36]
Thermal conductivity of bonding layer	0.225 W/mK	[37]
Specific heat capacity of bonding layer	2100 J/kgK	[37]
Area of thermal contact	4 mm^2	This work
Max. Conductive heat transfer coefficient (K_{cond})	8400 $\text{W}/\text{m}^2\text{K}$	This work
Max. Convective heat transfer coefficient (K_{conv})	120 $\text{W}/\text{m}^2\text{K}$	This work
Remanent magnetic field	1.07 T	This work
Number of turns of coil	800	This work
Area of coil	1.96×10^{-6} m^2	This work
Electrical resistance of coil (internal resistance)	250 Ω	This work
Electrical load resistance	400 Ω	This work
Operation temperature	423 K	This work

Appendix A.2. Experimental Performance Characteristics

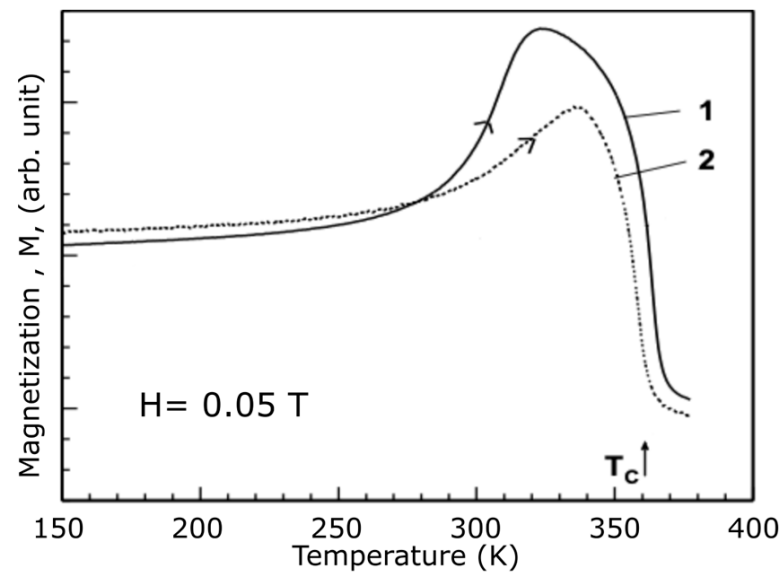


Figure A1. Typical low-field magnetization versus temperature curve of the investigated Ni-Mn-Ga films with two different thicknesses of 5 μm (1) and 0.4 μm (2) M, adapted from [41].

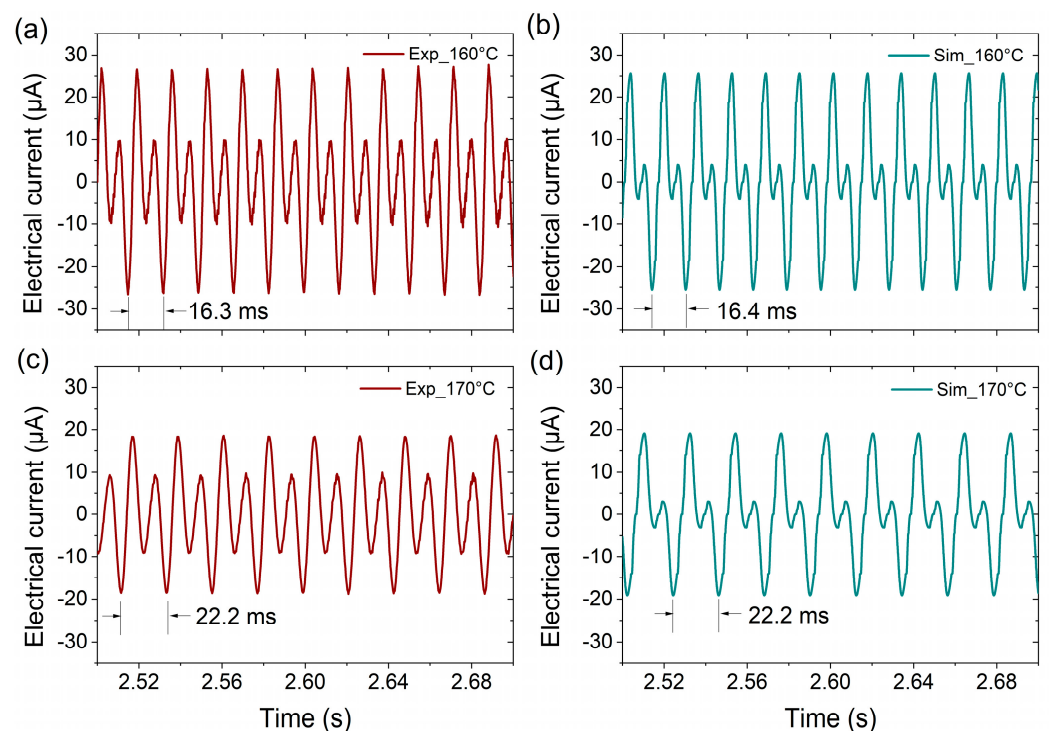


Figure A2. Performance of the bistable thermal energy harvester for heat source temperatures of 160 and 170 $^{\circ}\text{C}$. (a,c) experimentally determined electrical current in the pick-up coils connected in parallel, (b,d) are the corresponding LEM simulations. The simulation model reproduces the amplitude and frequency of the bistable MSMA film actuator very well, while the small peaks due to the changing direction of the magnetic field gradient are less pronounced.

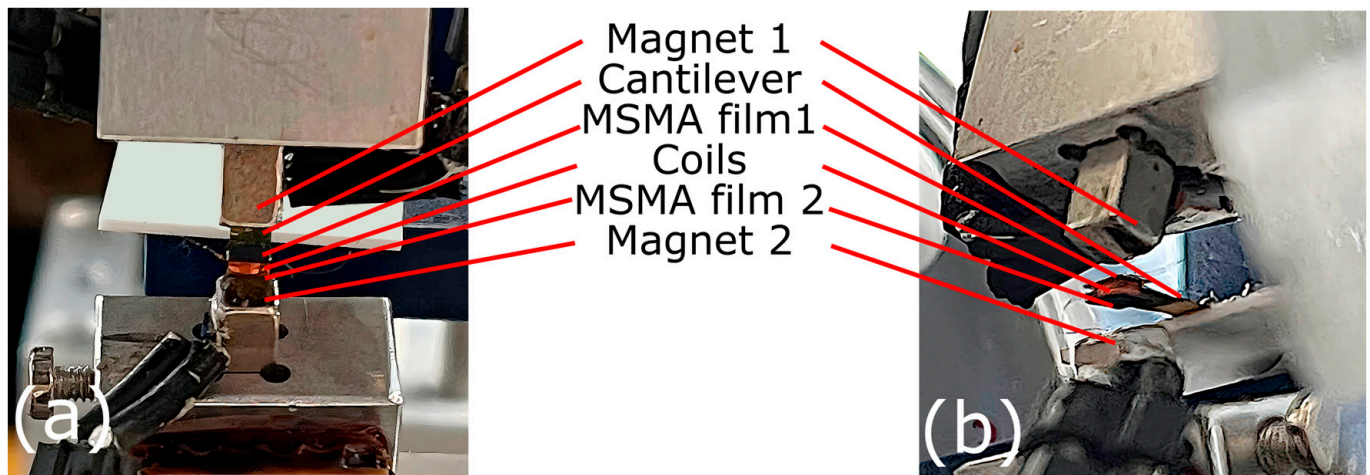


Figure A3. Experimental setup with the bistable MSMA film actuator shown in front view (a) and side view (b). It consists of a free-standing cantilever, two pick-up coils, and MSMA films 1 and 2 on both sides of the cantilever.

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