

KIT Results For Open Phase Of IAEA Benchmark CRP - I31038

A. Batta, A.G. Class
Institute for Thermal Energy Technology and Safety (ITES)

KARLSRUHER INSTITUT FÜR TECHNOLOGIE (KIT)

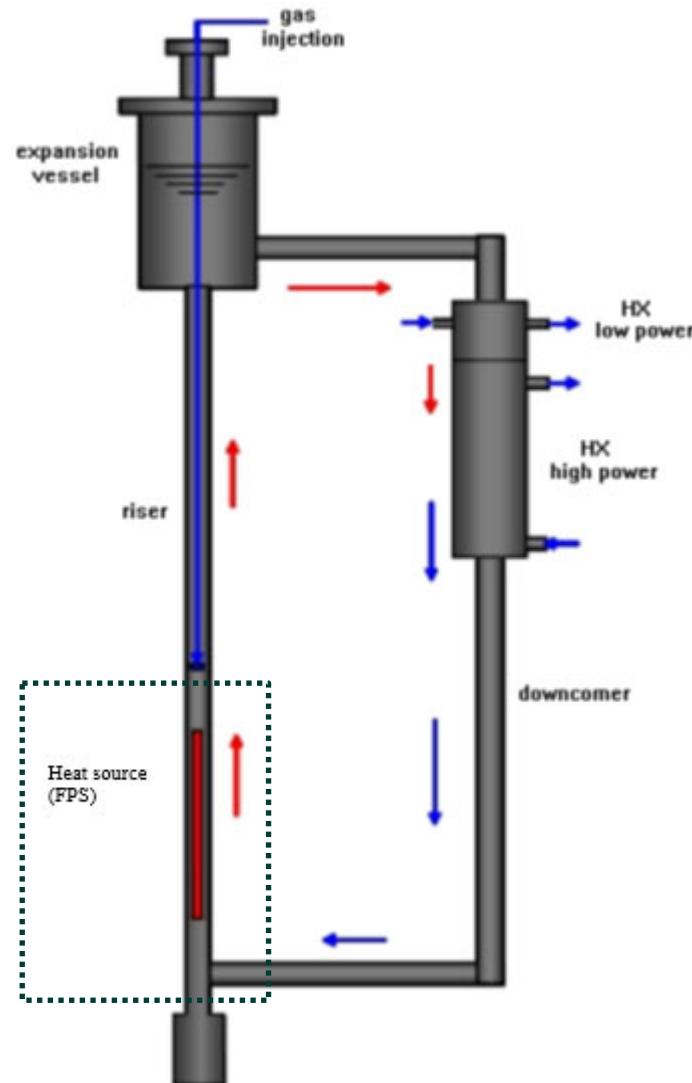


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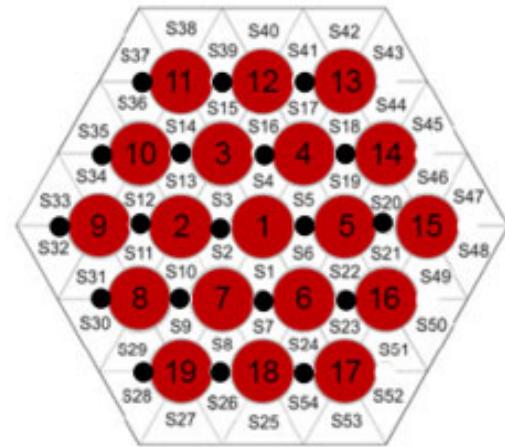
Schematic representation of the NACIE-UP primary loop



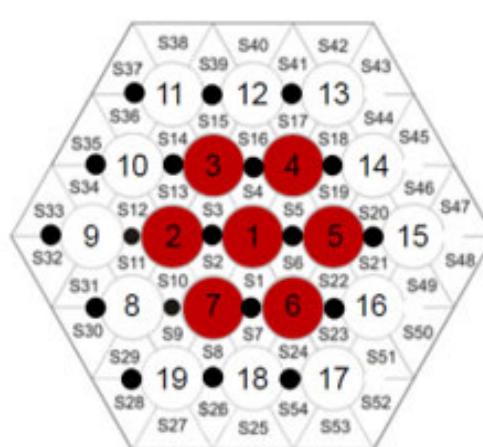
FPS design parameters

Parameter	Value
D_{pin}	6.55 mm
P	8.4 mm
P/D	1.2824 mm
d_{wire}	1.75 mm
P_{wire}	262 mm
L_{total}	2000 mm
L_{active}	600 mm
$D_{H,nom}$	3.84 mm

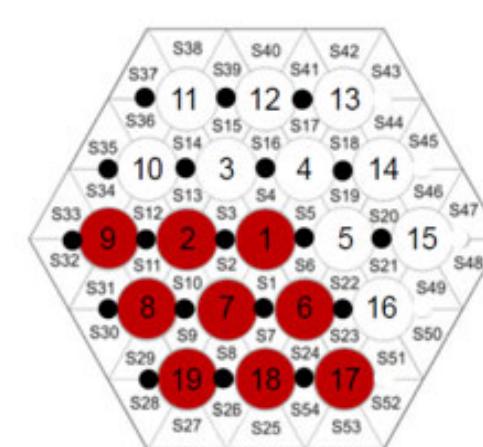
Benchmark of Transition from Forced to Natural Circulation Experiment with Heavy Liquid Metal Loop



ADP10
full heating, 30kW



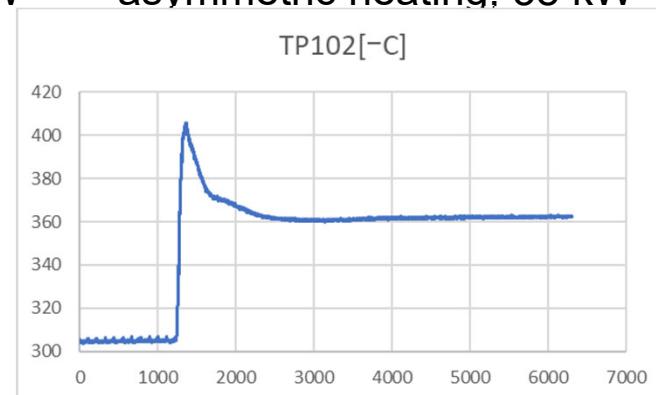
ADP06
central heating, 30kW



ADP07
asymmetric heating, 38 kW

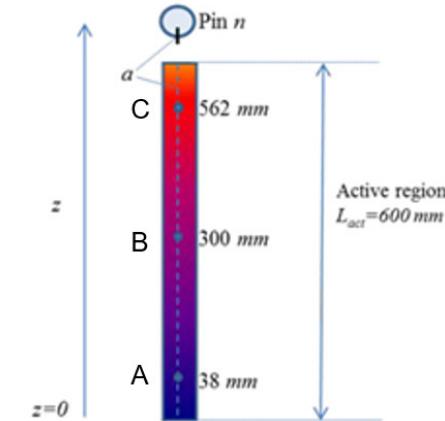
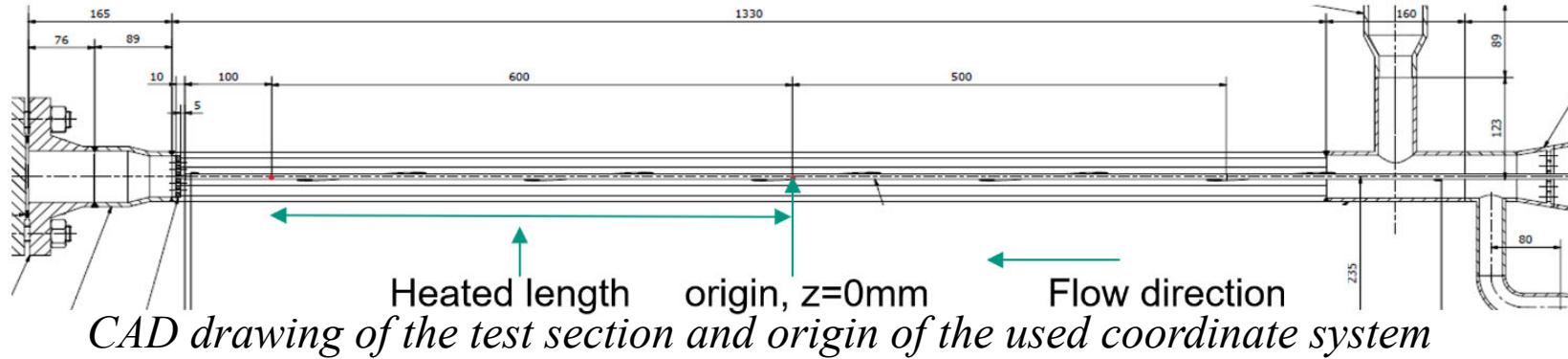
Parameter	Steady state 1			Steady state 2		
	Data	σ	$\sigma[\%]$	Data	σ	$\sigma[\%]$
$\dot{m}_{\text{gas}} (\text{Nl/min})$	10	0.5	5	0	0	0
$\dot{m}_{\text{LBE}} (\text{kg/s})$	2.56	0.28	11	1.31	0.14	11
$T_{\text{IN,FPS}} [\text{°C}]$	231.3	1.5		219.5	1.5	
$\Delta T_{\text{FPS}} [\text{°C}]$	72	0.7	0.9	140.6	0.3	0.2
$Q_{\text{nom}} [\text{w}]$	30000	50	0.2	30000	44	0.1
$Q_{\text{eff}} [\text{w}]$	27000	1053	3.9	27000	1010	3.7
$Q_{\text{pre}} [\text{w}]$	2236	403	18	2339	217	9.3
$Q_{\text{tfm}} [\text{w}]$	1915	3	0.2	1644	4	0.3

Integral parameters of the test ADP10



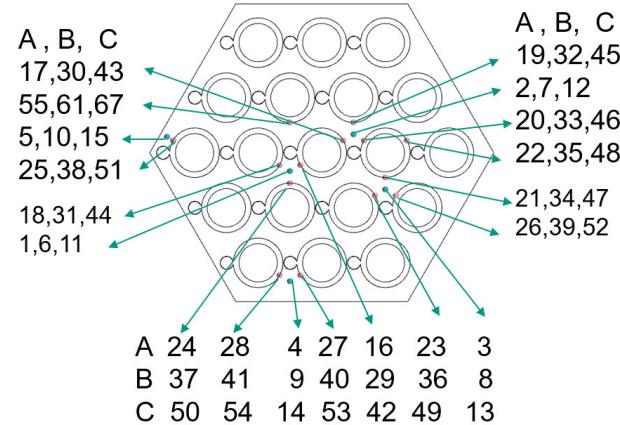
LBE temperature at the outlet of the test section TP102

Benchmark data / TC locations in the test section



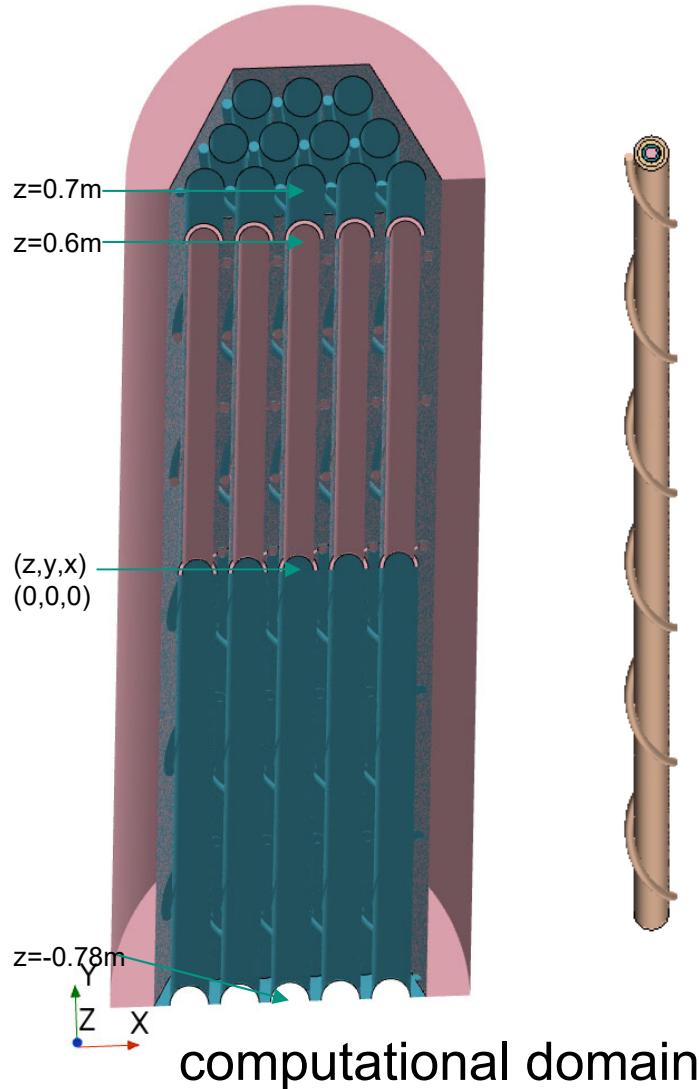
Location of planes for TC measurements in the test section (A at 38mm, B at 300mm and C at 562mm)

The accuracy of the thermocouples according to the rule IEC 60584-3 (2007) are: $\pm 1.5^\circ\text{C}$ from -40°C to $+375^\circ\text{C}$; $\pm 0.004 \text{ T}$ from 375°C to 1000°C .



location and names of thermocouples in measurements sections.

Numerical Model



Model:

- SST turbulence model
- All y^+ wall treatment
- Second order convection schema
- Model default parameters
- Adiabatic condition applied, neglecting heat losses to environment
- Conjugate heat transfer to rods and wrapper

Simulation:

- First run: simplified short heater, heat flux imposed at inner side of cladding
- Second run: full details of heater layers simulated

Thermo-physical properties

LBE physical properties, OECD/NEA Handbook 2015

LBE properties as a function of temperature (T in Kelvin, SI Units)

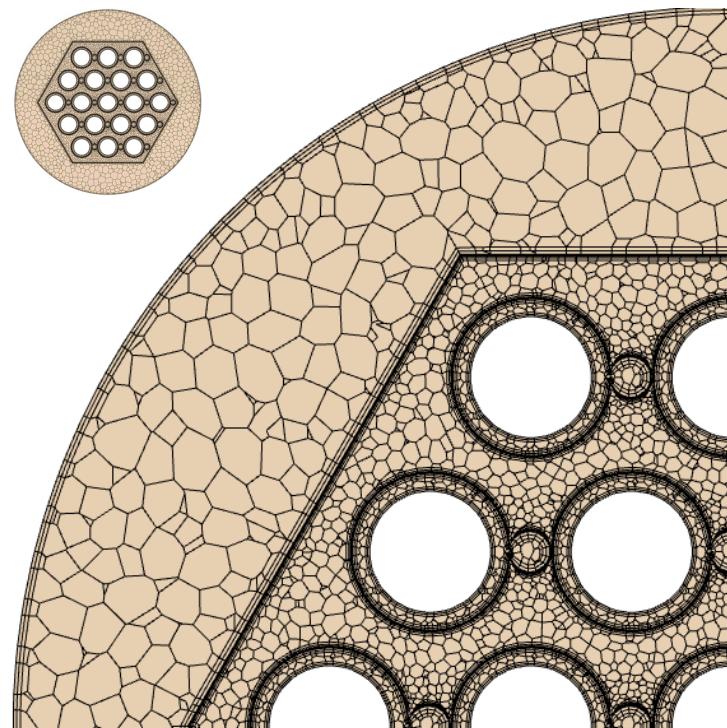
Property	Symbol	Correlation	Maximum Uncertainty	Standard deviation
Density	$\rho(T)$	$11065 - 1.293 \cdot T$	$\leq 0.8\%$	0.58%
Heat capacity	$c_p(T)$	$164.8 - 3.94 \cdot 10^{-2} \cdot T + 1.25 \cdot 10^{-5} \cdot T^2 - 4.56 \cdot 10^5 \cdot T^{-2}$	$\leq 5.0\%$	2.4%
Dynamic viscosity	$\mu(T)$	$4.94 \cdot 10^{-4} \exp\left(\frac{754.1}{T}\right)$	$\leq 6.0\% - 8.0\%$	7.2%
Thermal conductivity	$k(T)$	$3.284 + 1.617 \cdot 10^{-2} \cdot T - 2.305 \cdot 10^{-6} \cdot T^2$	$\leq 10.0\% - 15.0\%$	6.2%

For other materials, benchmark specifications are used

Mesh and turbulent Model

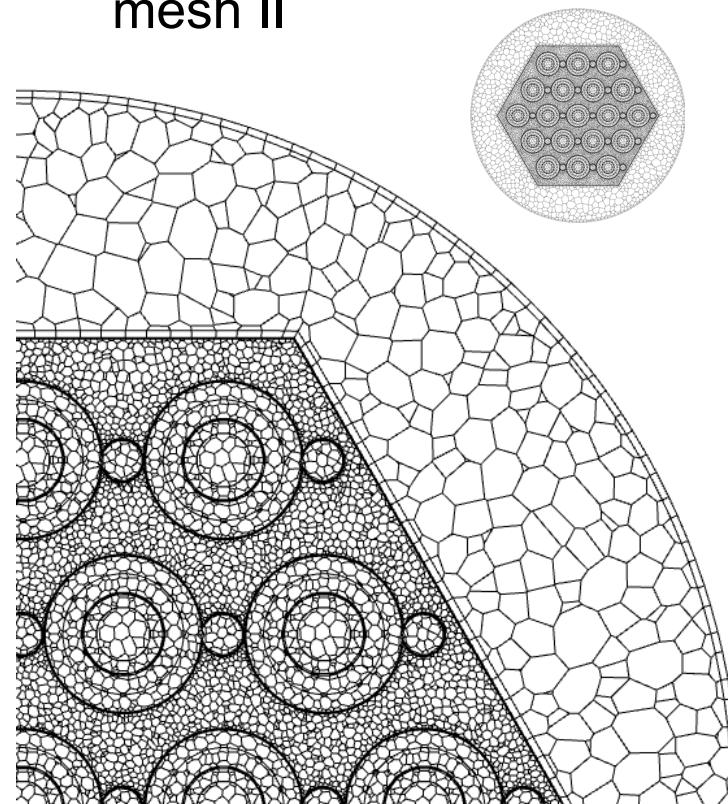
- SST All y+ wall treatment, Star CCM+
- Y+ small resolving buoyancy near heated walls

mesh I



Fluid 49 M cell
Whole 62 M cell

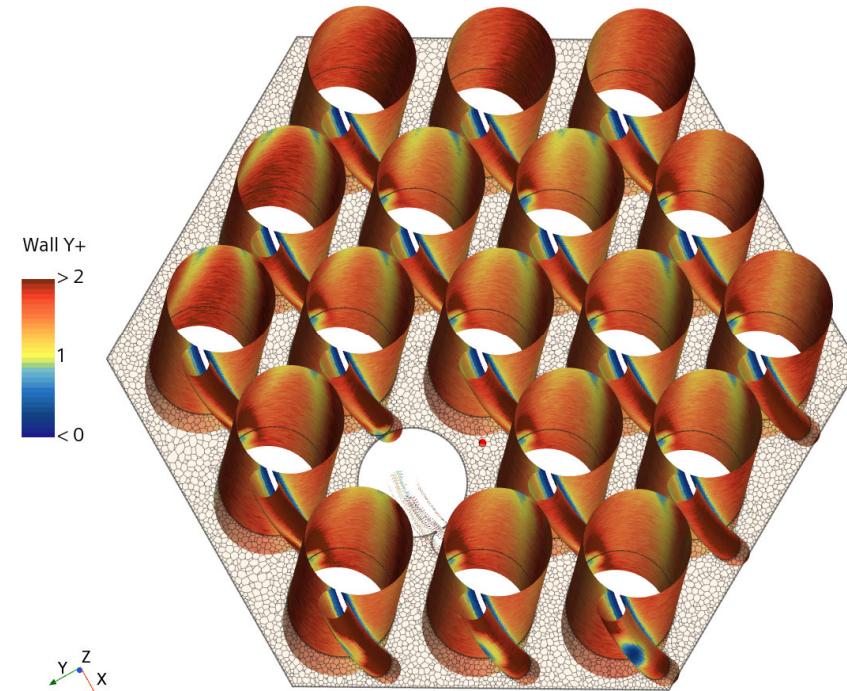
mesh II



Fluid 96 M cell
Whole 125 M cell

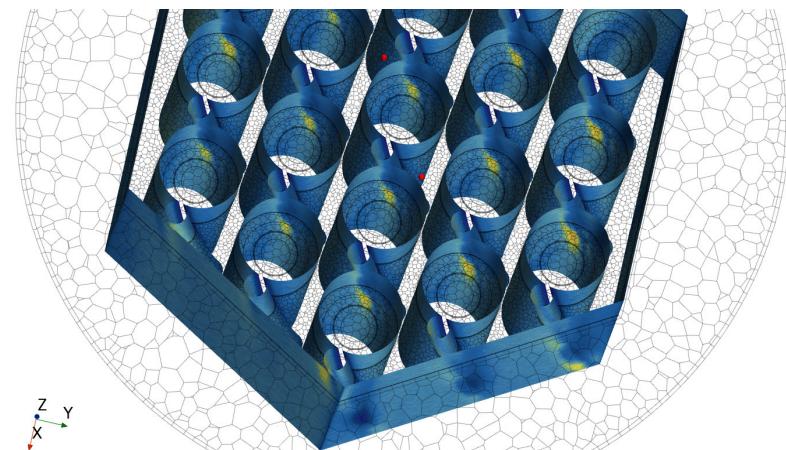
Y+ values, mesh II, ADP 10 case A (Forced Circulation)

Simcenter STAR-CCM+



Y+ range 0 to 2

Wall Y+
5
4
2
1
< 0

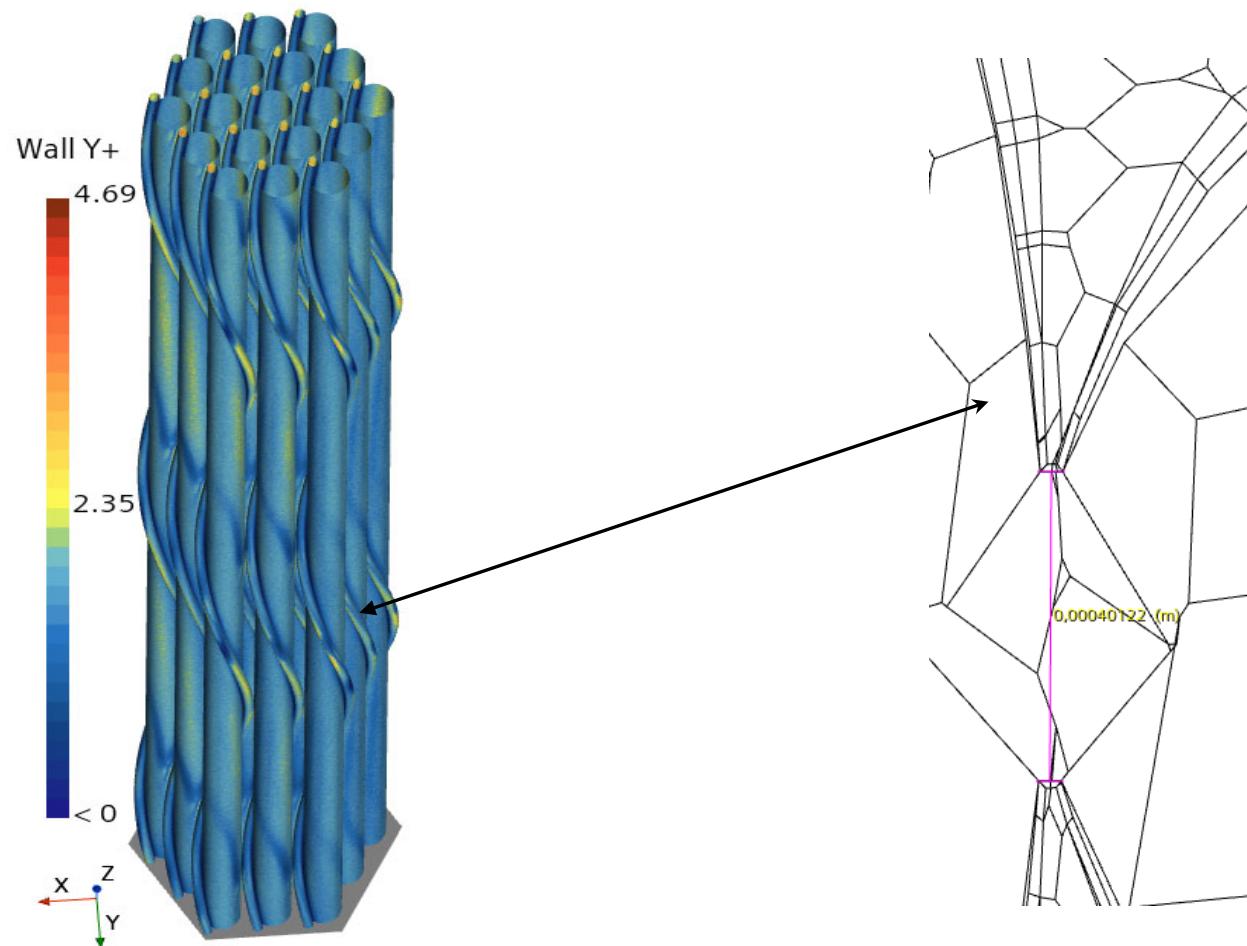


&

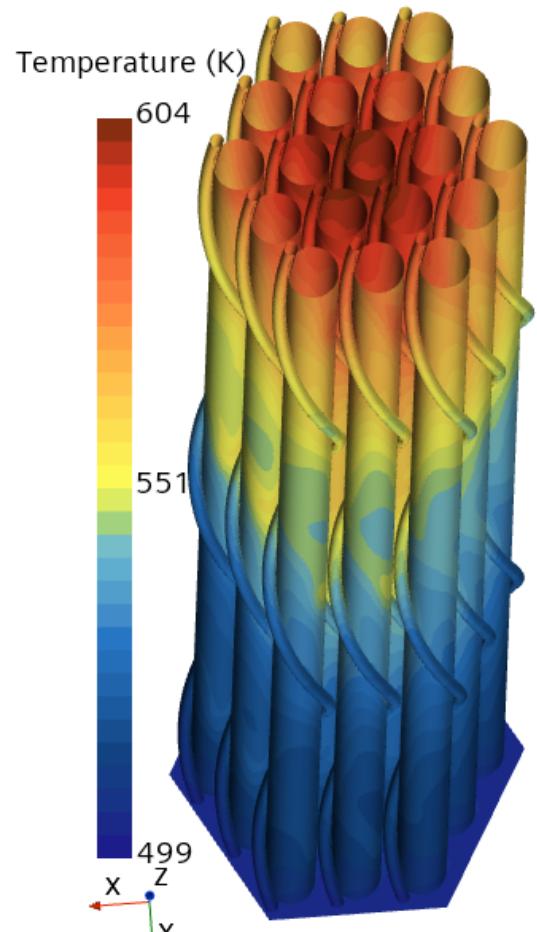
range 0 to 5

Cross-section showing mesh in fluid domain & wrapper near section A

Mesh resolving contact of wire and pin



Results, case ADP10 steady state1, mesh II.

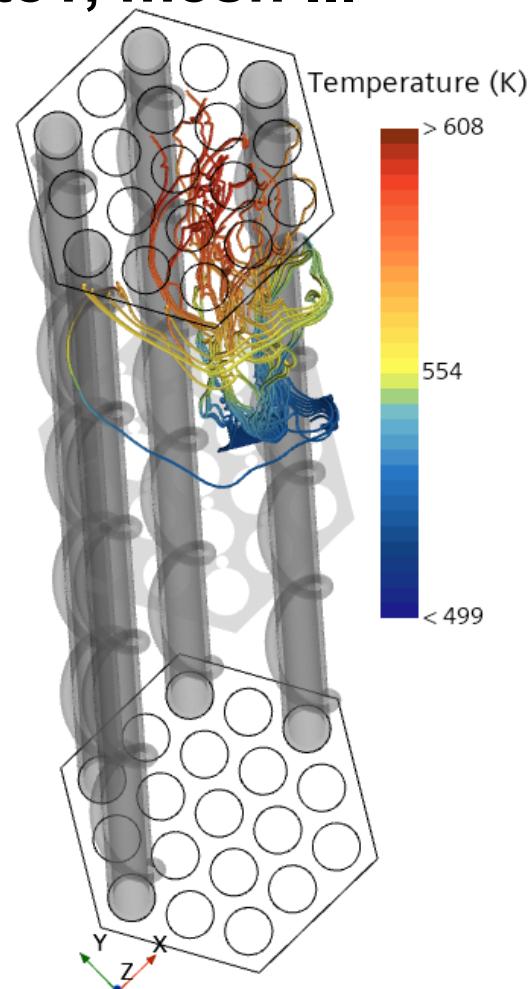


Temperature contours at heater
fluid interface in heated region,
 $z=0.0$ to 0.6 m

heated section
0 to 0.6 m

$z=0.0$ m

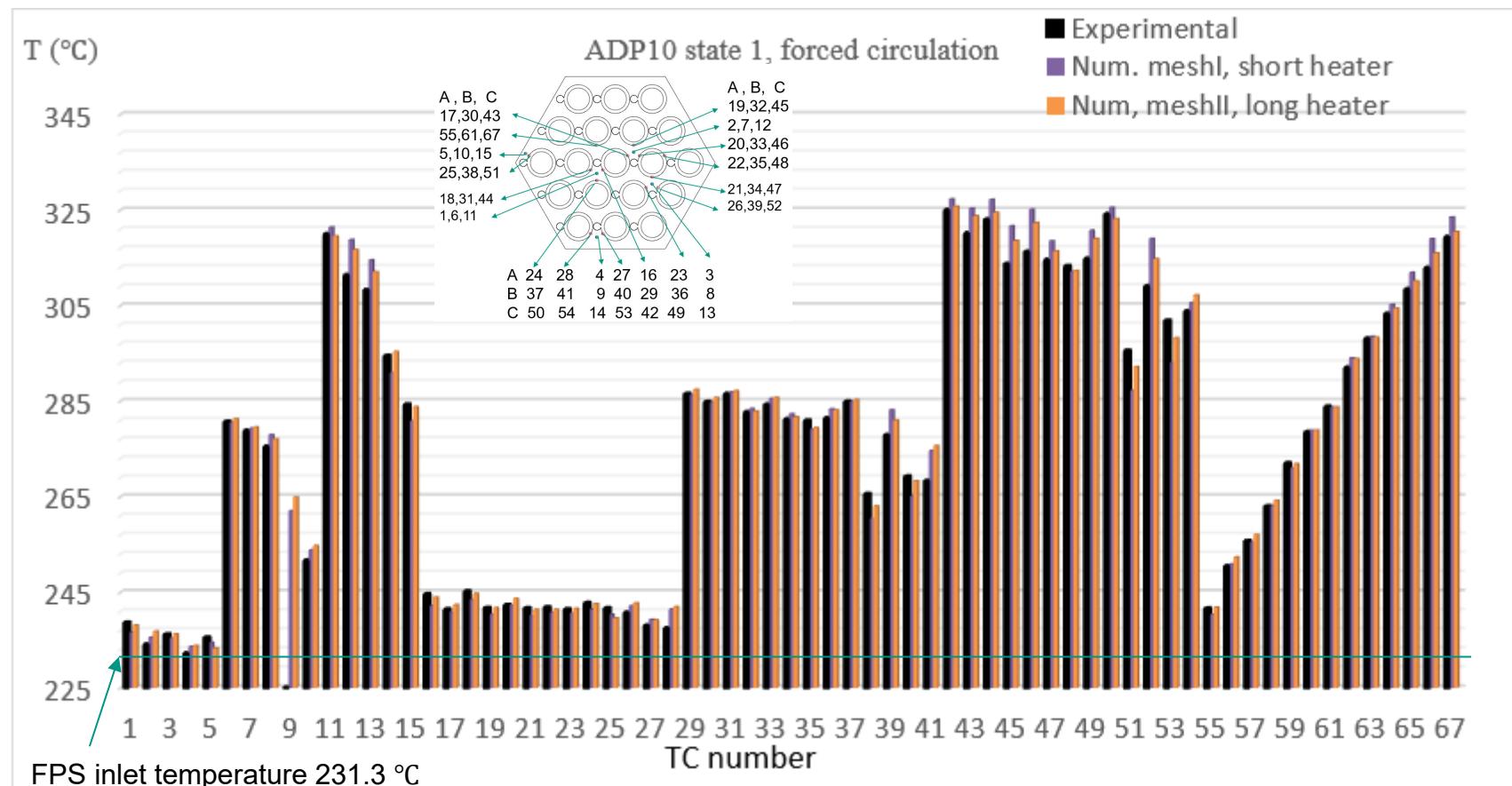
unheated section
-0.78 to 0 m



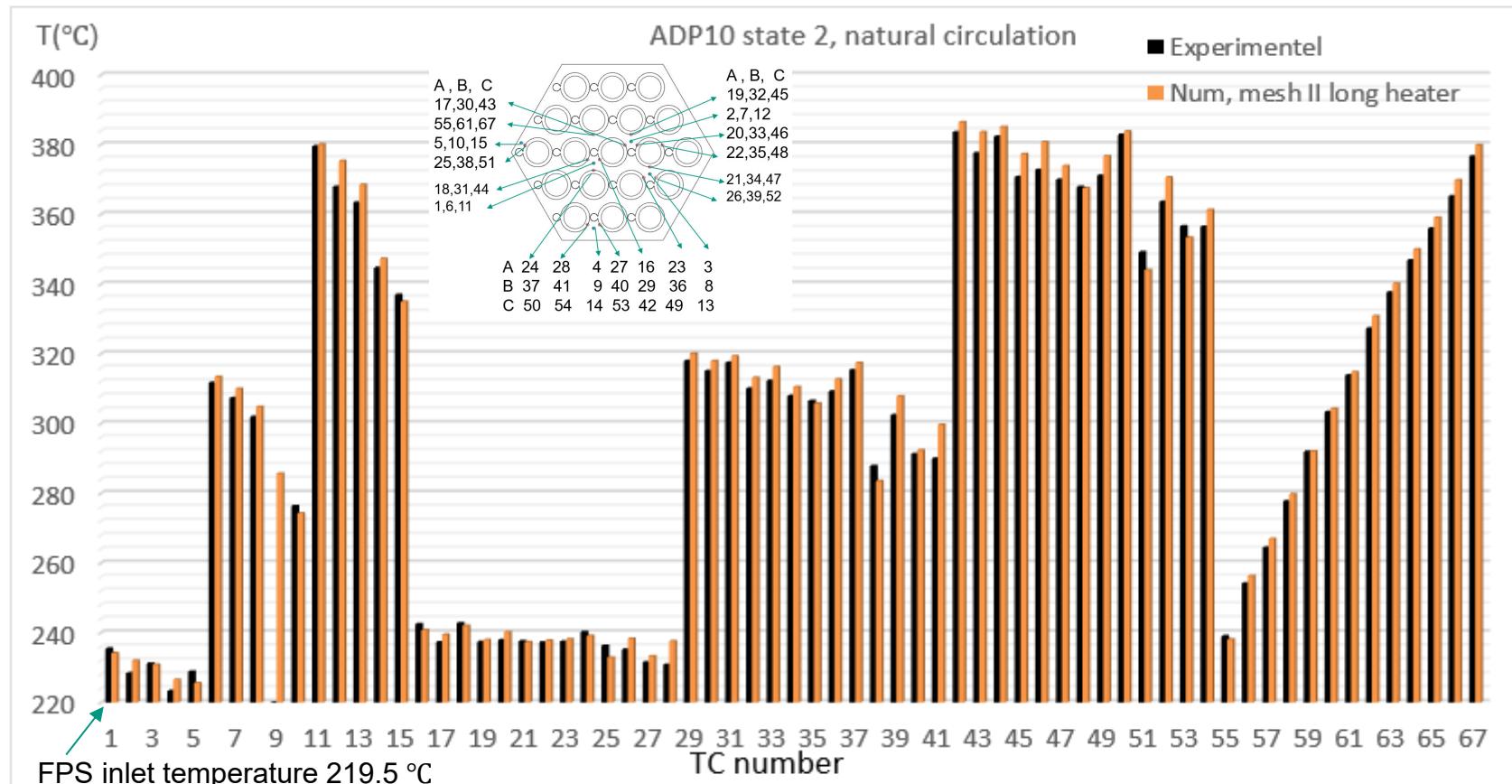
Streamlines from a line in x-y plane just
downstream of start of heated zone,
 $z=0.038$ m

Forced circulation results mesh I with short heater, mesh II with long heater (Forced convection)

Comparison to experiment, averaging 100 measurements in steady period

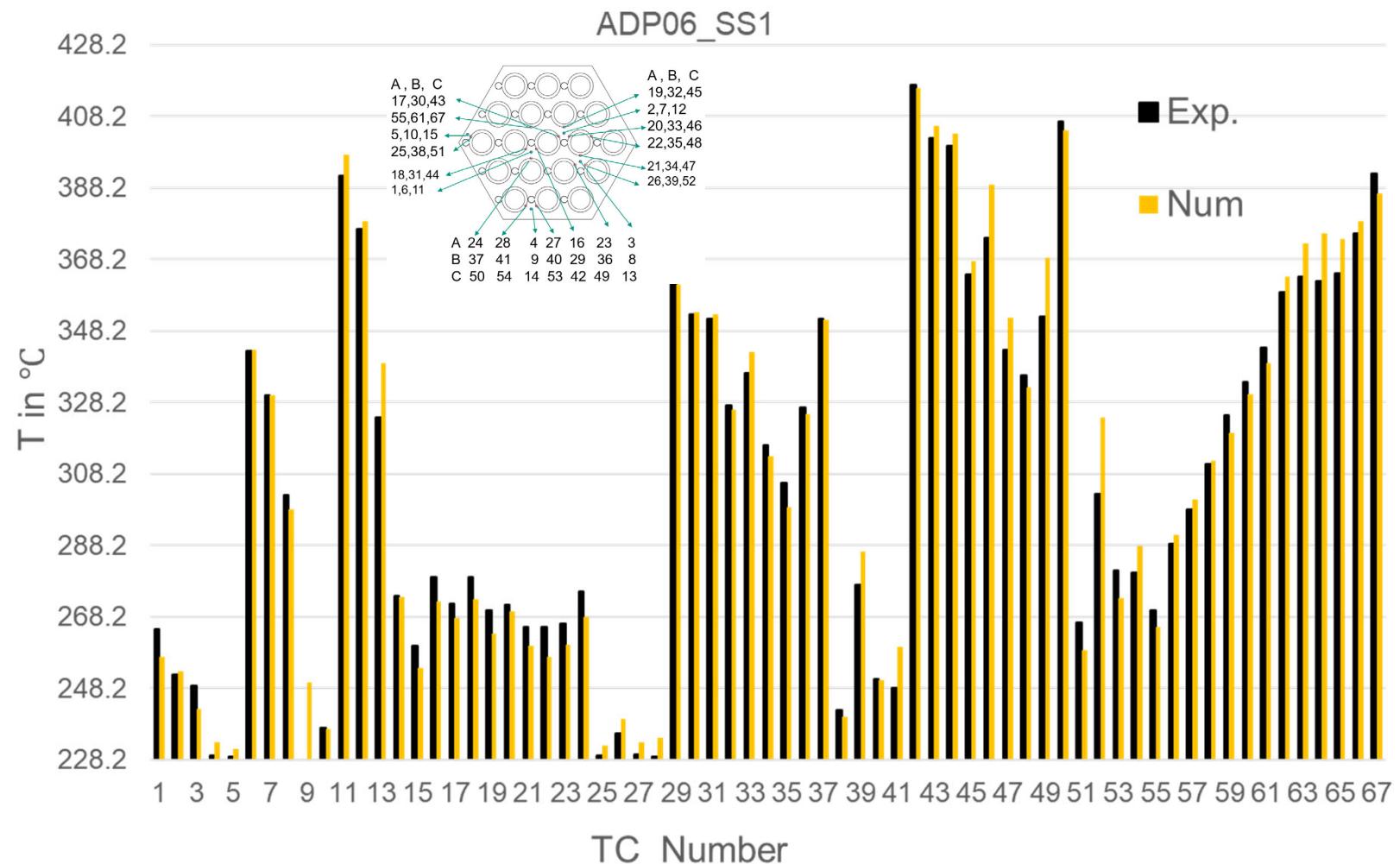


Natural convection results, mesh II with long heater

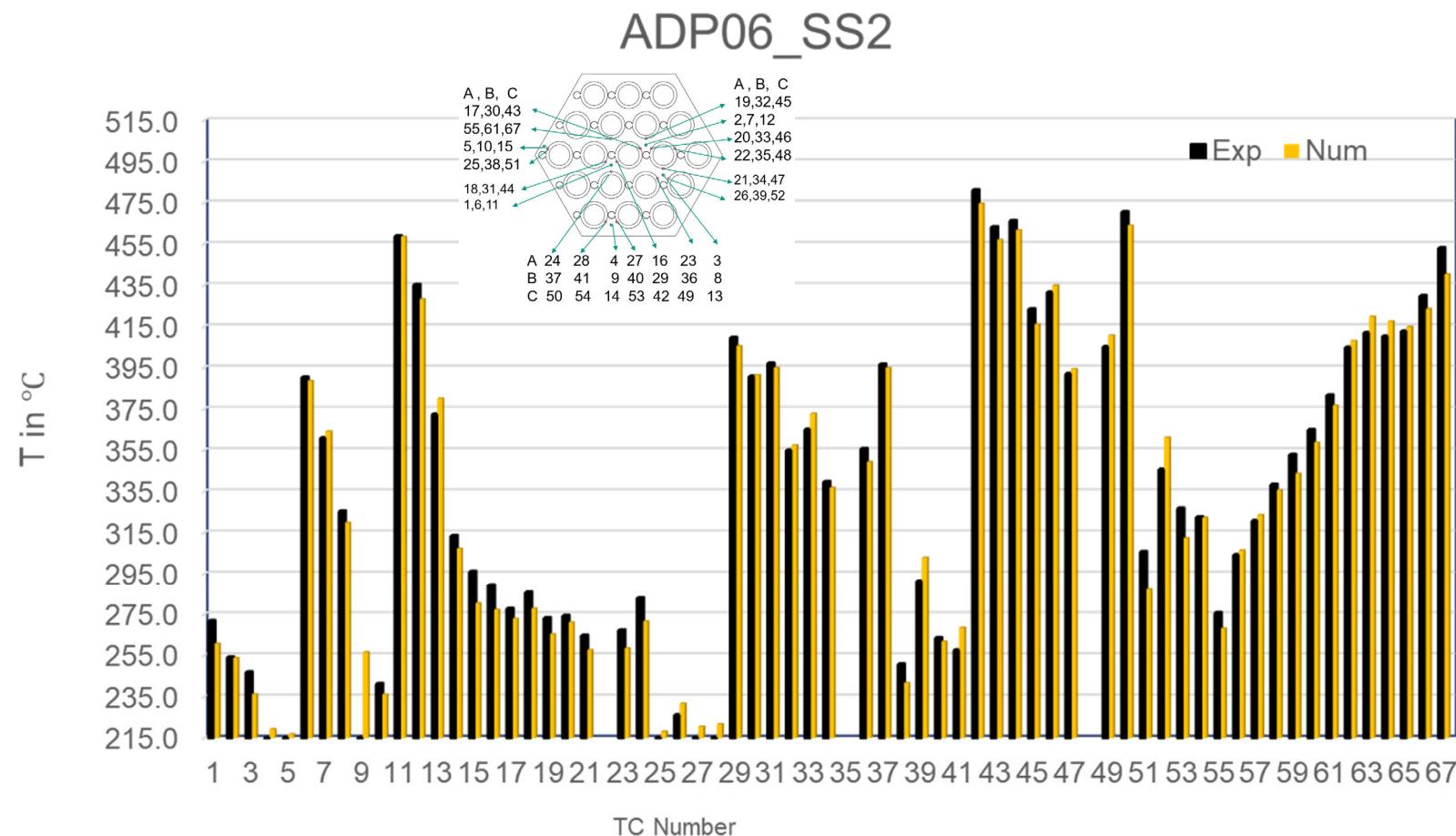


- Error within uncertainty of measurements / material properties / dimensions
- Small absolute error, no systematic trends

Forced circulation results, mesh II with long heater



Natural convection results, mesh II with long heater



- Error within uncertainty of measurements / material properties / dimensions
- Small absolute error, no systematic trends

Conclusions

- Test of various modelling for heater show small influence on results
- Small sensitivity of results to mesh refinement
- Selected SST model & finer mesh & details in heater model are used for blind phase case with asymmetric heating
- Results for the blind phase are submitted.
- In regions with high temperatures, errors can be related to uncertainties in heating power, benchmark specification & modelling (physical parameters, turbulence models)
- Near inlet and boundary regions (low temperatures) yields larger uncertainty due to boundary conditions and TC measurement uncertainty.
- Published results: SCOPE (Saudi International Conference On Nuclear Power Engineering) 13–15 Nov 2023 King Fahd Conference Center, KFUPM, Dhahran, KSA

ACKNOWLEDGMENTS



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