

Experimental activities in support of the EU-DEMO Helium Cooled Pebble Bed Blanket design and qualification

HCPB-related Experiments

Breeder Unit

- BU-cooling: Pin-mockup
 - HTC in the annular gap
- Breeding material interaction with Eurofer
- Pressure loss coefficients for pebble bed (cm/s velocity range)

First Wall

- Prototypical Mock-up Units
 - Oxide Dispersed Strengthen (ODS) Steel-FW Mock-up
 - W-Functional Graded (FG) coated FW Mock-up
 - First Wall PMU

ODS-FW & FG-FW Experiments

Objective

- feasibility study of industrial fabrication and production processes
- demonstration of material high temperature performance at $T=550 \div 650^{\circ}\text{C}$ under high heat loads

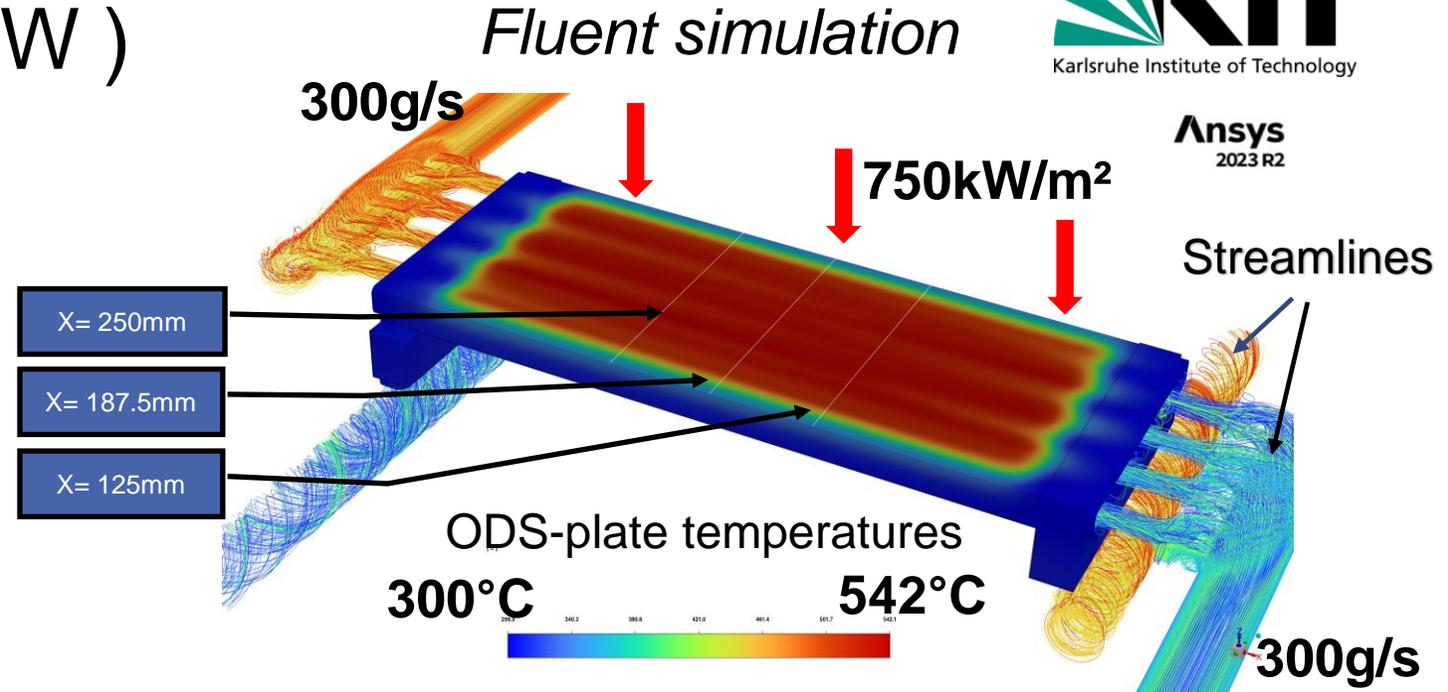
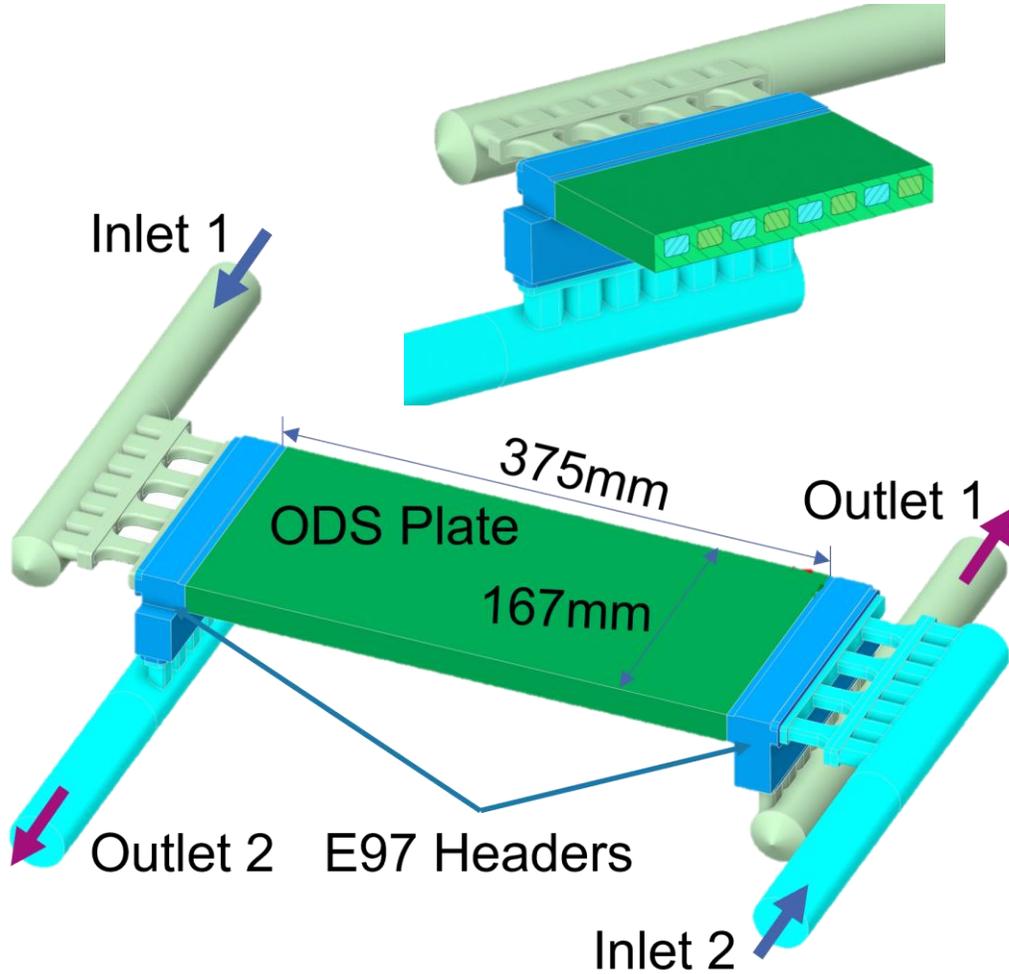
ODS-FW Mock-up

- RAFM 9Cr-ODS Steel Plate (**375x167x20.5** mm³)
- Loading cycles (1000) with high heat flux (750kW/m²) loading on the surface: surface temperature **550°C÷650°C**
- **Mock-up:**
 - 8 channels (15x10mm²) with counter-flow cooling
 - Heated surface 375x130mm²
 - ODS to E97 HIP joining
 - Inconel transition piece from E97 to 316L;

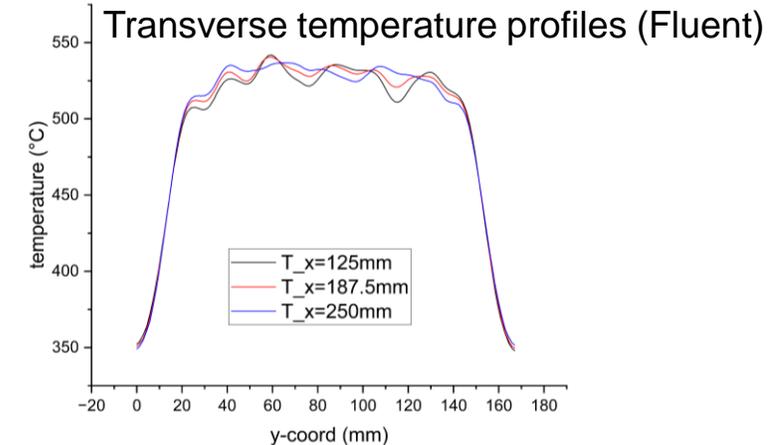
FG-FW Mock-up

- Industrial W FG coating process applied on a **390x247** mm² surface
- Loading cycles (1000) with high heat flux (750kW/m²) loading on the surface: substrate (E97) surface temperature around **550°C**
- **Mock-up:**
 - 12 channels (15x10mm²) with counter-flow cooling
 - Heated surface 350x230mm²
 - Inconel transition piece from E97 to 316L;

Mock-up Design (ODS-FW)

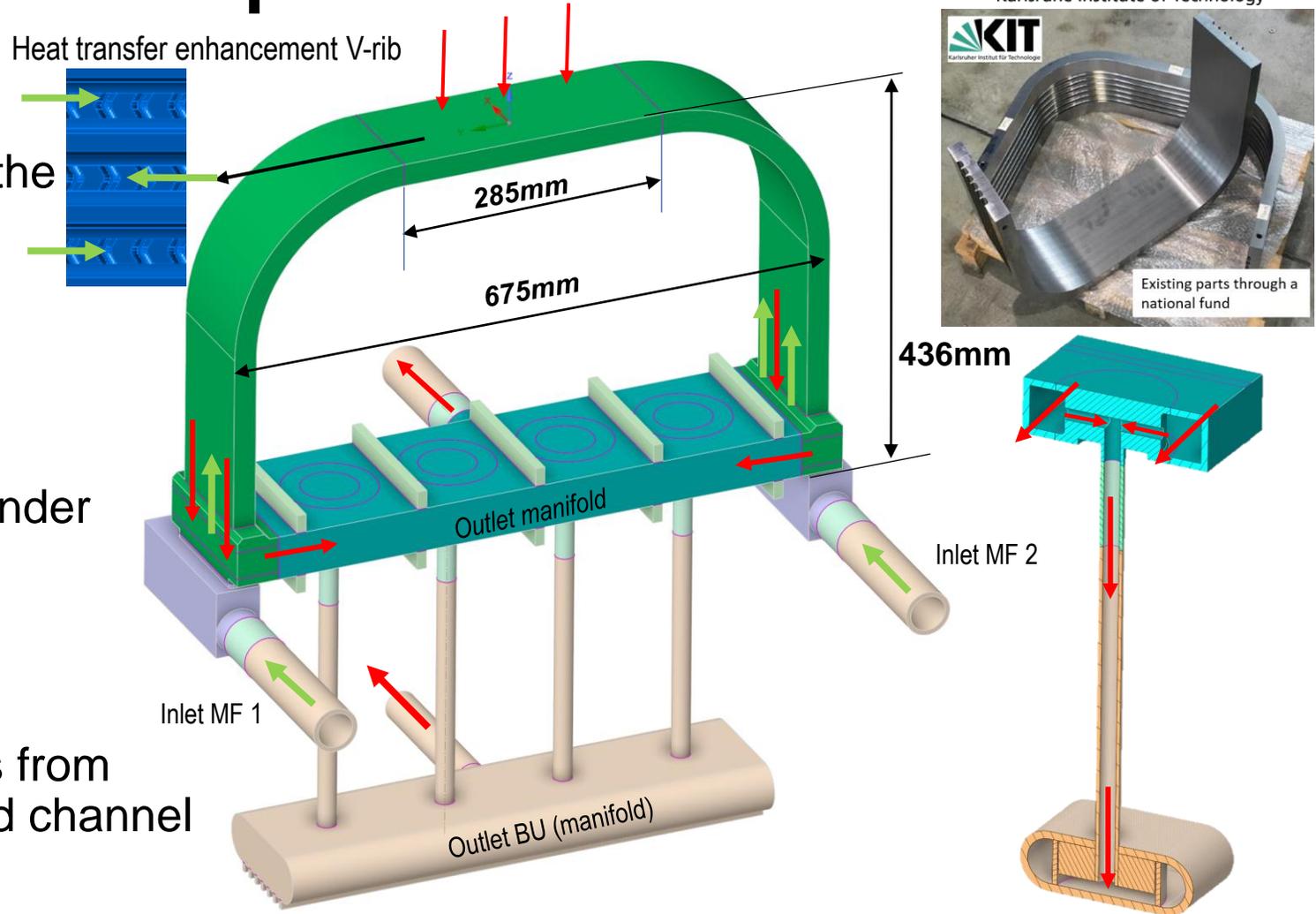


(raw) ODS plate with headers



FW-PMU: Prototypical FW Experiment

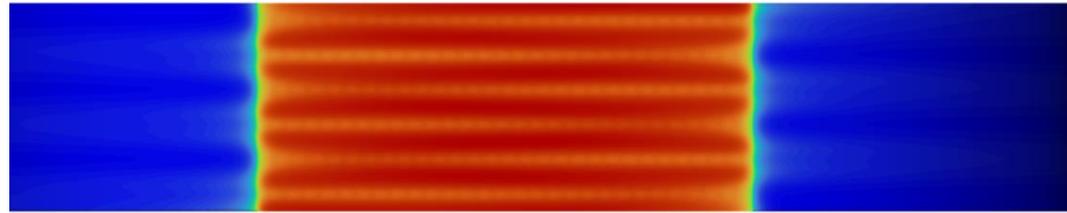
- Manufacturing prototype investigating the use of a new technological path
 - U-shaped FW with 6 channels
 - V-rib heat transfer enhancement on the heat loaded side of the channels
- Primary focus on cooling capabilities under high heat flux loadings
- Outlet manifold includes elements of a HCPB BU cooling path
- Test campaign with surface heat fluxes from 280kW/m^2 up to 1MW/m^2 (or more) and channel flow rates from 30 g/s up to 150 g/s



BU-like cooling path elements

FW-PMU: thermal-hydraulic analysis (CFD)

Surface temperature @ 750kW/m² & 70g/s /channel



[C]

299.4 324.0 348.5 373.1 397.7 422.2 446.8 471.4 495.9 520.5 545.1



[C]

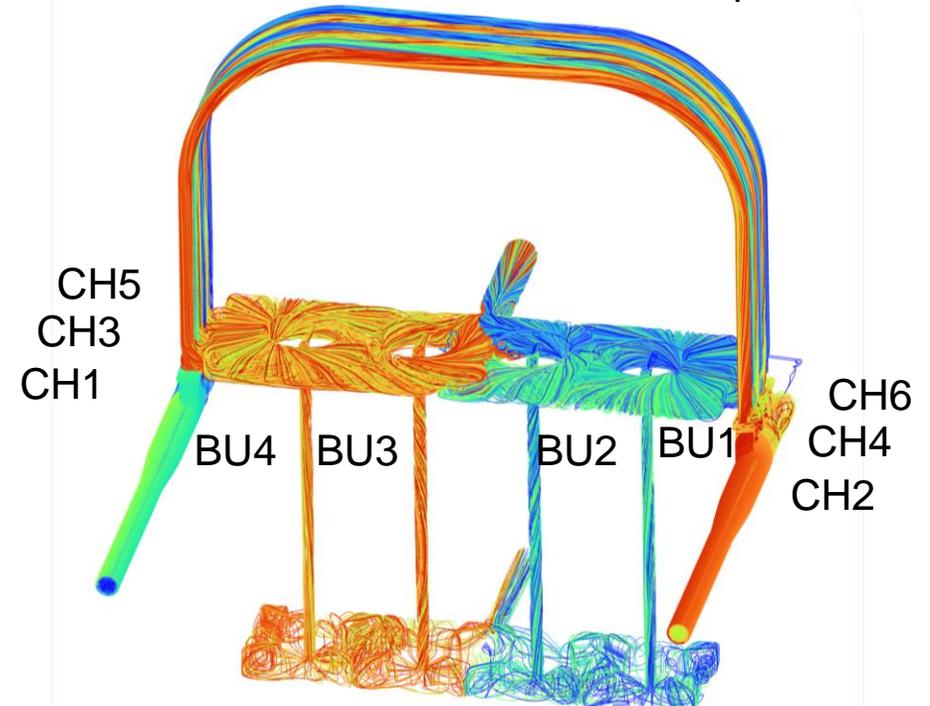
Channel wall temperature

316.0 330.7 345.5 360.2 374.9 389.7 404.4 419.1 433.9 448.6 463.3



| | CH1 | CH2 | CH3 | CH4 | CH5 | CH6 |
|----------------------|------|------|-------|------|-------|------|
| Flow (g/s) | 70.1 | 69.6 | 67.99 | 68.7 | 72.97 | 70.7 |
| Deviation to nominal | 0% | 1% | 3% | 2% | -4% | -1% |

Streamlines: Colours indicate the particle ID



| | BU1 | BU2 | BU3 | BU4 |
|-------------------|------|------|------|-------|
| Flow (g/s) | 15.4 | 15.3 | 14.4 | 14.97 |
| Deviation to mean | -2% | -2% | 4% | 0% |

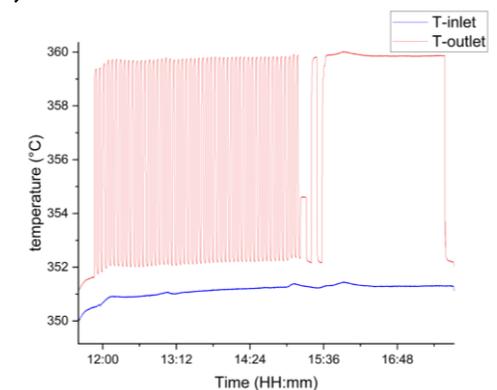
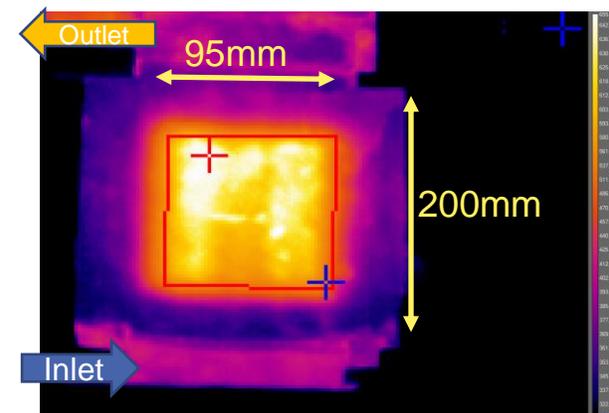
Validation of computed temperature field

Widely used methods

- CFD: fluid-solid coupled simulation (single software tool like CFX or Fluent)
- Fluid simulation coupled with heat transfer in solids (distinct software tools: CFX or Fluent coupled to ANSYS Thermal)
- Heat transfer in solids with (simplified) flow and fluid to solid heat transfer models (ANSYS Thermal with fluid lines, for instance)

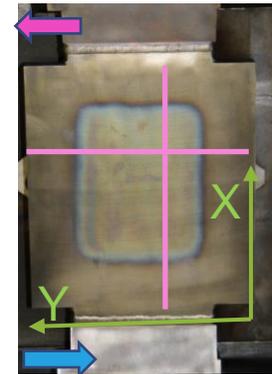
Validation case: 2019 (1st) ODS-FW Exp.

- 5 channels mock-up with co-curent flow
- Long (~2h) pulses
- IR recordings of the surface temperature
- Reference case: run from 07.10.2019
 - $199.9 \pm 0.2\text{g/s}$
 - $351.3 \pm 0.9 \text{ }^\circ\text{C}$ @ inlet;
 - $7.75 \pm 0.9 \text{ }^\circ\text{C}$ @ outlet;

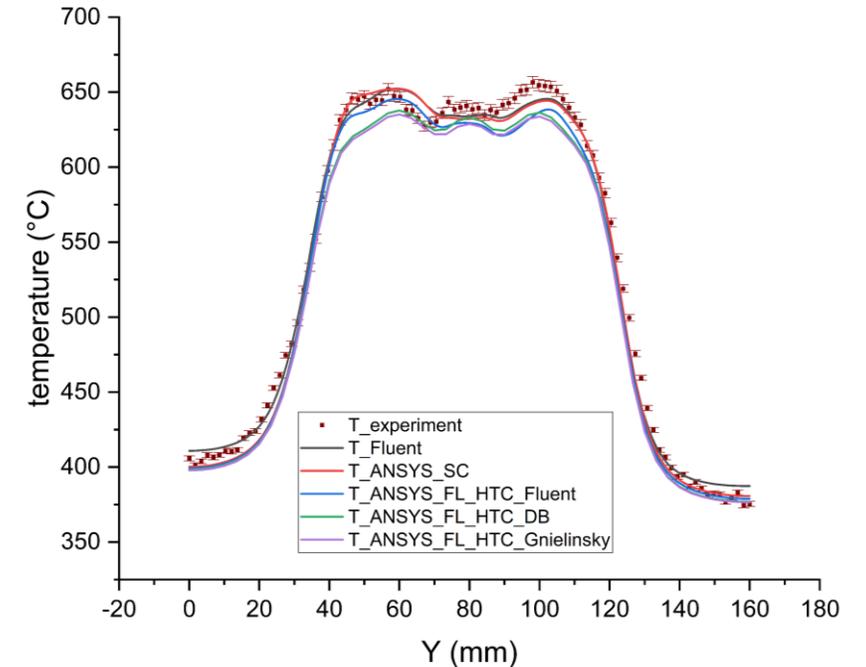
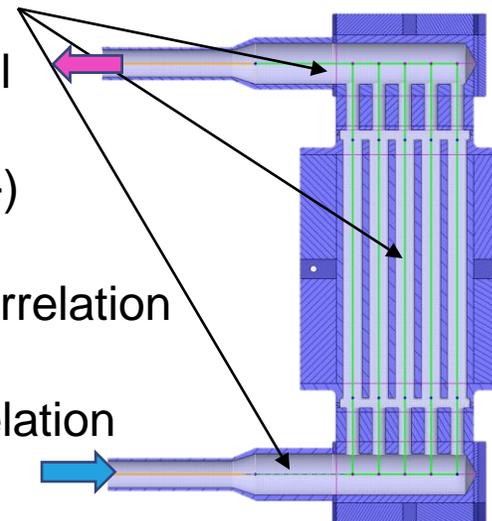


Validation of computed temperature field (cont.)

- Heating profile:
 - Power to match the value in experiment: 8894.6W
 - Width & Length from shaded area on the mock-up
- CFD Model: case 1
 - use Fluent (ANSYS 2023R2)
 - Turbulence model SST
- ANSYS Thermal:
 - Case 2: System Coupling Fluent & Thermal
 - Case 3: Thermal with Fluid Lines, HTC from Fluent (Y+)
 - Case 4: Thermal with Fluid Lines, HTC Dittus-Bölder correlation
 - Case 5: Thermal with Fluid Lines, HTC Gnielinski correlation



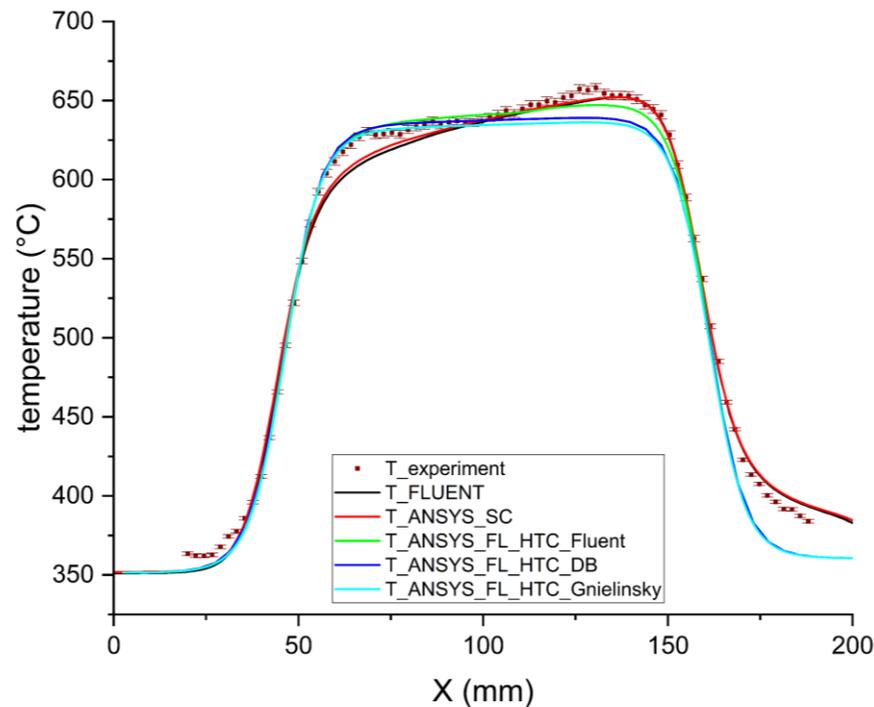
Fluid Lines



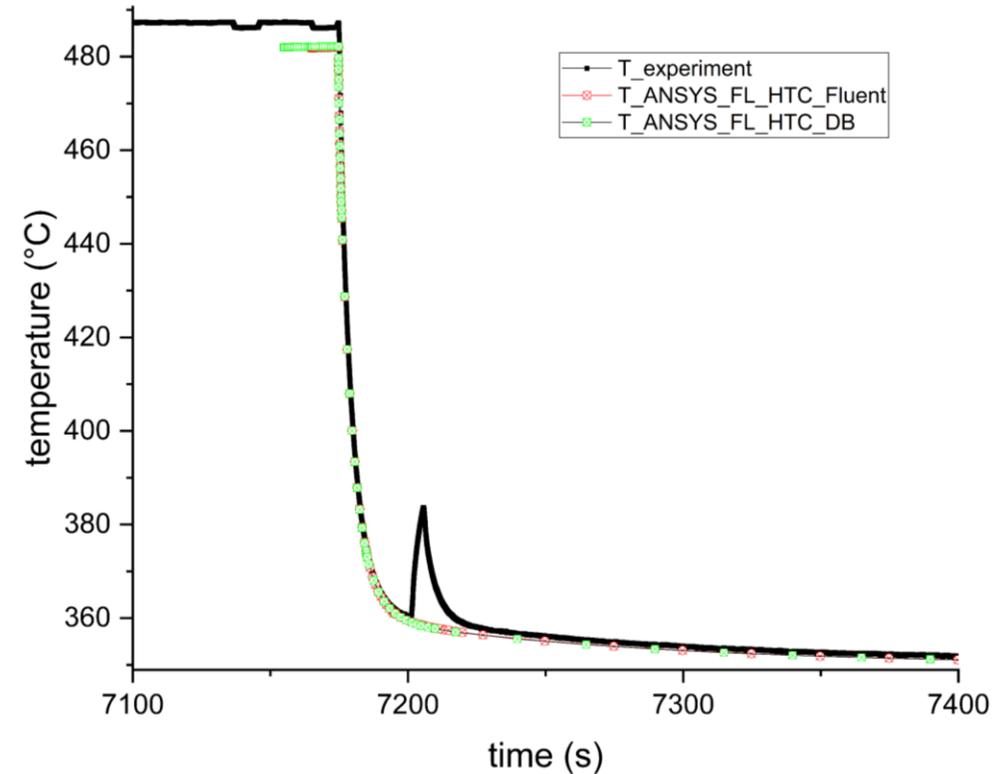
Temperature profiles @ $x=136.5$ mm (steady state calculation)
 Fluent Surface T_{max} @ $x=136.5$ mm; $y=57.4$ mm

* FL mass flow from Fluent
 ** Dittus-Bölder and Gnielinski only for the channel region
 ***Gnielinski from VDI Heat Atlas (2010): G1, Section 4

Validation of computed temperature field (cont.)



Temperature profiles @ $y=57.4$ mm
(steady state calculation)



Time evolution of the surface mean temperature
 HTC-Fluent: 0.9% deviation of the simulated values to experiment
 HTC-DB: 0.8% deviation of the simulated values to experiment

Overview & Conclusions

- Wide range of experiments in support of HCPB-BB design
 - **BU**: single effect experiments concerning thermal hydraulic or materials issues
 - **FW**: increased size mock-ups -> prototypical mock-ups
 - Materials: ODS and FG-FW mock-ups with industrially applies technologies
 - Manufacturing: FW-PMU
 - Thermal-hydraulics with channels and flow configurations of relevant sizes
- Model validation
 - Fluid-solid conjugated simulations show good fit with the experimental data
 - (ANSYS Thermal) Models using Fluid Lines and HTC from CFD (either Fluent or Fluent+ANSYS:SC) slightly overpredict the cooling but they are very close to the experimental data both for steady state and transient analysis
 - HTC correlations (HTC versus temperature):
 - Dittus-Bölder show similar performance as the HTC from CFD for both steady state and transient simulations but the temperature profile in the flow direction tends to be rather flat
 - Gnielinski gives similar results as Dittus-Bölder with a slight overestimation of the cooling; the use of a stream-wise dependent correlation is presently investigated

Thank you for your attention !

Contributors:

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