New Co-Simulation Variants for Emissions and Cost Reduction of Sustainable District Heating Planning

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Abstract-Classical heating of residential areas is very energy-intensive, so alternatives are needed, including renewable energies and advanced heating technologies. Thus, the present paper introduces a new methodology for comprehensive variant analysis for future district heating planning, aiming at optimizing emissions and costs. For this, an extensive Modelica-based modelling study comprising models of heating center, heat grid pipelines and heating interface units to buildings are coupled in co-simulations. These enable a comparative analysis of the economic feasibility and sustainability for various technologies and energy carriers to be carried out. The new modular and highly parameterizable building model serves for validation of the introduced heat grid model. The results show that bio-methane as an energy source reduces carbon equivalent emissions by nearly 70% compared to conventional natural gas heating, and the use of hydrogen as an energy source reduces carbon equivalent emissions by 77% when equipped with a heat pump. In addition, the use of ground source heat pumps has a high economic viability when economic benefits are taken into account. The study findings highlight the importance of strategic planning and flexible design in the early stages of district development in order to achieve improved energy efficiency and a reduced carbon footprint.

Keywords—District heating, Buildings, Modelica, Co-Simulation, Economy, Sustainability.

I. INTRODUCTION

As efforts are made to reduce the impacts of climate change [1], the transition to more sustainable and efficient energy systems is an urgent imperative in the face of growing energy demand. In this context, heating through the heat grid accounts for a large part of the energy consumption in residential areas [2], which highlights the importance of optimizing heating solutions [3]. Quaggiotto et al. [4] used a graph-theoretic model of the existing heat grid in Verona, Italy, to analyze the economic and environmental benefits of the district heating under different control strategies and with or without thermal storage tanks. In [5], the possibility of cost and emission savings of multiple district building setups, installed with different heat pump (HP) and district heating system technologies, are evaluated. However, since the studied districts are already in use, only the existing configurations are taken into account and no further heating configuration possibilities or more energy options are

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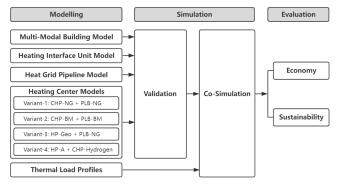


Fig. 1. New methodology of the economic and sustainability analysis with multi-physics model variants of heating centers in a co-simulation setup.

provided. Sterchele et al. [6] show in their study that there is potential for generating heat in a heat grid using a variety of novel measures such as large-scale HPs, peaking boilers, and combined heat and power plants (CHPs) operating with bioor synthetic methane. Based on this, the present study aims to build a heat grid system digital twin of a real district being under construction, and to calculate and analyze multiple aspects of alternative heating options at the heat grid operator level.

The contribution of this paper is a new methodology of a systematic analysis and co-simulation of four variants of a heating center designed for a real residential area, evaluating each configuration through multiple key performance indicators such as economic efficiency and sustainability. The comparative analysis of the advantages and disadvantages of each variant and the horizontal evaluation provide a solid basis for understanding their respective impacts and provide actionable recommendations for the planning and development of new areas. By this research, the authors close the gap between theoretical models and practical, scalable solutions that could significantly improve energy efficiency and the sustainability of urban development, and to provide the best use scenarios for different energy sources and heating technologies.

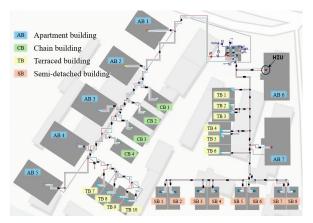


Fig. 2. Residential area heat grid model built in Dymola using the Modelica language.

II. METHODOLOGY

The new methodology of this paper is outlined by the flowchart in Fig. 1. First, by using the Modelica modelling language, the heat grid of the planned reconstruction area is modeled on the basis of the data provided by the builder, whereby scientific assumptions are made about the missing data and are applied to the model design. For an advanced evaluation of the heat grid model properties, the buildings in this area are modelled as modular and highly parameterizable multi-physics models with a heat exchange unit (or heat interface unit, HIU), which serves as the coupling between the building models and the heat grid model. In a second step, the correctness and feasibility of the model is verified. For the analysis of the impact of different fuels and heat-generating equipment on the economic benefits and CO₂ emissions of heat grid, four variants for the heating center are introduced. Their simulation results are compared to provide the best variant solution in several aspects. All model variants are modeled with *Dymola* using *Modelica* and are translated into FMU units for co-simulation of the energy system.

A. Heat Grid in the Study Area

The study area provides a blueprint for exploring and analyzing heating options in close cooperation with municipal utilities, property developers and investors. The area planning and a respective heat grid model are shown in Fig. 2, consisting of two modules connected by interfaces: the pipeline and the heating center model. This split design concept provides high flexibility for the model and facilitates the subsequent design of multiple variants of the heating center.

In the pipeline model, the selection of the pipe manufacturer and the design of the pipes were made by the respective engineering office. The pipe information is provided by the pipe manufacturer isoplus (www.isoplus.org), so that the pipeline model is modeled as realistically as possible considering thermal inertia and heat loss. In the heating center model, a basic load module and a peak load module are added to ensure the heat demand of the area over the entire period. Different configurations of these two modules will produce different variants of the heating center. In the present paper, an existing configuration and three further variants are selected.

B. Modelling of Buildings and Substation

A total of 29 buildings under construction and of various types in this area are shown in Fig. 2. Due to the diversity of

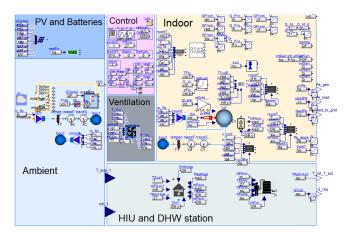


Fig. 3. Modular building model built in *Dymola* using the *Modelica* language.

parameters of various buildings, the manual modelling of all buildings is workload intensive and not viable. Thus, we present a modular building model with universal properties, see Fig. 3, in which not only each module (such as HP and PV module, etc.) can be activated or deactivated by control signals, but also their parameters can be modified through parameter tables to easily generate building models representing different buildings types.

The HIU in the building, which exchanges heat with the heat grid, and the domestic hot water station (DHW) can be seen in the lower right corner of the model view in Fig. 3. The hot water from the heat grid transfers heat to the heat buffer tank in the HIU via a heat exchanger. The heated water in the buffer tank is used for room- and domestic water heating. The temperature of the return water from the building to the heat grid is set to 55°C under the regulation of the controller to reduce the heat loss during the transmission of hot water in the grid. In addition, radiators, ventilation systems, and electrical loads are included in the building model to simulate the actual indoor temperature under heating provided by the heat grid as accurately as possible.

C. Load Profile of Buildings

In addition to the *Modelica* based building model, in order to calculate the building heat demand, the heat grid model also supports the import of heat load demand data files from external sources. Since only an estimated overall annual duration curve and the annual trends of heat demand are provided in the area construction plan, and the specific heat demand of each house is not provided in advance, the Fraunhofer-SynPro tool [7] is used to estimate the building heating and hot water demand, based on user types and building configurations.

D. Co-Simulation via Model Coupling

As mentioned in section II, the modules are translated into FMUs and coupled for simulation. Co-simulation [8] through the FMI interface standard allows multiple subsystems to run in the same simulation environment, which greatly reduces the divide between different subject models and provides great convenience for large-scale multi-subject energy system co-simulation as demonstrated in our previous studies [9].

III. MODEL VARIANTS OF HEATING

Analysis of future district heating is performed for four variants of the heating center, ranging from conventional heating with natural gas up to the utilization of renewables considering HPs and hydrogen (H₂). See Table I for all variants.

TABLE I. OVERVIEW OF HEATING CENTER VARIANTS.

Variants	Basic Load Module	Peak Load Module					
Variant-1	CHP - Natural Gas	Peak Load Boiler - Natural Gas					
CHP-NG + PLB-NG	2g – g-box 50 plus	Viessmann – Vitocrossal 200 CM2					
Variant-2	CHP - Biomethane	Peak Load Boiler - Biomethane					
CHP-BM + PLB-BM	2g – g-box 50 plus	Viessmann – Vitocrossal 200 CM2					
Variant-3	HP - Geothermal heat	Peak Load Boiler - Natural Gas					
HP-Geo + PLB-NG	Viessmann – Vitocal 350-HT Pro	Viessmann – Vitocrossal 200 CM2					
Variant-4 HP-A + CHP-H2	HP - Air 2g – afilia air M-AEK-sI-4	CHP - Hydrogen 2g –agenitor 404 pure H2					

A. Variant-1 (Existing Status): CHP-NG + PLB-NG

The combination of natural gas CHP and natural gas peak load boiler corresponds to the current construction plan. Natural gas is a widely used energy source, however, to achieve environmental targets, emissions must be reduced [1].

The existing heating center was designed according to the plans of the respective engineering office as well. The left diagram in Fig. 4 shows the schematic structure of the center, which consists of two buffer storage tanks of seven cubic meters each, a CHP unit with a maximum heat output of 108kW for base load, and a gas boiler of 400kW for peak load. Based on the requirements of the area design, an outflow temperature from the heating center of approx. 80°C is assumed.

In addition, the model of the generated heat energy by the heating center is simplified to the difference between the actual heat demand of the area and the heat output of the storage tank, where the storage tank output is calculated as follows:

$$\dot{Q}_{buffer} = \dot{m} \cdot c_p \cdot (T_2 - T_1) \tag{1}$$

where \dot{m} is the mass flow rate of water through storage buffer tanks in kg/s, c_P is the specific heat capacity of the water, T_1 and T_2 are the temperatures on the return side and the supply side of the tank group, respectively. When the area heat demand is less than half of the CHP's maximum load, the CHP unit is only used to maintain the temperature of the heat storage tank. Conversely, if the heat demand is higher than the heat that the fully loaded CHP can provide, the CHP and peak load boiler are used together to provide heat.

B. Variant-2: CHP-BM + PLB-BM

Bio-methane is seen as a sustainable domestic energy source that can be used where natural gas was previously used, also, it is considered to be in secure supply [10]. This variant explores the possibility of reducing emissions without displacing existing energy producers. The heating center remains as in Variant-1, the energy source is changed to biomethane.

C. Variant-3: HP-Geo + PLB-NG

In this variant, the CHP unit is replaced by a ground source heat pump (GSHP). The general structure of the heating center is shown in the middle of Fig. 4. Like the existing heating system, Variant-3 consists of two buffer storage tanks and a peaking boiler with a heat output of 400kW, which, again, uses natural gas as an energy source. On the source side of the HP, the underground probe used is selected as a double U-shaped probe. Due to the limited space in the study area, it is assumed that the HP is equipped with 30 probes, and the distance between the probes is 10m [11]. In addition, the maximum output power of the HP has been matched to the

maximum power of the CHP units in the other variants to ensure comparability.

D. Variant-4: $HP-A + CHP-H_2$

Hydrogen is seen as a promising energy source of the future, especially for heating, and as a game changer in the energy transition in Germany [12]. In Variant-4, the dimensions of the heat storage tank are the same as in Variant-3. The heating center is equipped with an air source heat pump (ASHP) and the CHP unit uses H₂ as energy source (see Fig. 4). In winter, the return water from the area is first preheated by the HP to 65°C and then continues to be heated in the CHP unit to 80°C. In summer, since the heat demand of the area is lower compared to winter, only the HP is used to meet the area heat demand, i.e., the HP heats the return water to 70°C, while the CHP unit is used to satisfy the thermal peak loads only. To simplify the calculation, this present paper will not consider the role of CHP in the electrical grid.

IV. EVALUATION

The heat grid model is validated using three *Modelica* building models. The economic and sustainability analyses are performed based on heat load data from SynPro [7].

A. Model Validation

According to the data provided by the property developer, the heating power per square meter q_H of the buildings in the area should be around 35W/m^2 , the simultaneity factor g is 80%, and the full utilization hours F is 1500h/a. For a building with a conditioned area (A_C) of 186.8m^2 , the total heating energy required in a year is calculated as follows:

$$Q_H = \frac{q_H \cdot A_C}{1000} \cdot g \cdot F = 7845.6 \, kWh \, / \, a \tag{2}$$

which corresponds to a heating demand per square meter of 42kWh/(m²a), which corresponds to the heating demand of the reference building *DE.N.SFH.12.Gen.ReEx.001.003* in the TABULA [13] project. For this reason, the envelope parameters of the three *Modelica* building envelope models are chosen to have the same values as this reference building.

In order to complete the model validation, the heat grid model with the heating center in its existing status (Variant-1) is co-simulated with the three building models. The indoor temperature is controlled at about 20°C from 0 to 6 o'clock and at about 22°C from 7 o'clock to 23 o'clock in January. The simulation results of the indoor temperature deviation of the three buildings (b_1 , b_2 and b_3) and the sensible heat output \dot{Q}_{rad} from the radiator to the indoor air are shown in Fig. 5. The temperature deviation occurs due to the change of external solar light and the delay of the heat grid response. It can be seen that the heat grid can supply the building with adequate heat according to the temperature requirements.

B. Seasonal Analysis of Energy Balance and Emissions

For the economic and sustainability analysis of the heating center, the thermal loads of all 29 buildings are generated by SynPro. This is accomplished to ensure consistency in building load behavior and to better observe changes between heating center variants. In addition, January, April, and August are selected as the observation time window for calculation and analysis.

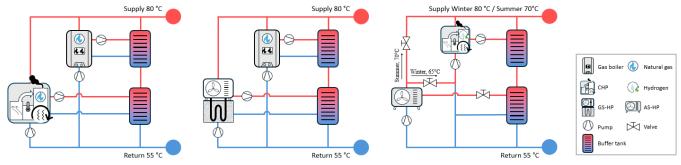


Fig. 4. Schematic diagram of heating center variants f.l.t.r. Variant-1, Variant-3 and Variant-4.

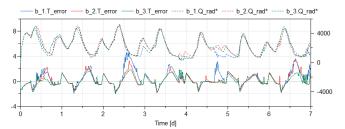


Fig. 5. The deviation between the actual indoor temperature and the set temperature of the three *Modelica* buildings within one week in K (bottom) and the sensible heat output from the radiator to the indoor in W (top).

Regarding CO₂ emissions, the emission values per MWh (heat and electricity) of various energy sources in 2021 are based on the reference values in [14]. Due to space limitations, the present paper concentrates on the calculation results of the existing status (Variant-1) and Variant-4 (see Fig. 6). Other results will be presented and summarized in detail in section IV-C.

By comparing the power output in three different months, the total heat supply of Variant-1 in January is calculated to be 86MWh, and in April it is 47MWh, of which CHP accounted for 76% and 89% of the heat output respectively; in August the heat supply is 29MWh, and the peak load boilers remain closed during this period. The area marked by striped shadows is the electricity output of the heating center, accounting for around a quarter of the total output energy. Fig. 6-right shows the emissions in tonnes of CO₂ equivalent, which are 22.94t, 12.04t and 7t for Variant-1 respectively.

In Variant-4, it is assumed that the electricity produced by the CHP will be supplied to the HP first, and the excess electricity will be exported to the grid. If the electricity required by the HP is higher than the electrical output of the CHP, the grid will serve as a supplement to make up for the energy consumption of the HP. The simulation results show that the heat energy produced by the heating center in the three months is 86MWh, 46.9MWh and 29MWh respectively, which is consistent with the results in existing status. The power consumption of the HP is defined as a negative value, while the power generation of the CHP is defined as positive. Therefore, it can be seen that in January and April, the electricity consumption of the HP can be completely covered by the electricity generated by the CHP. However, since the CHP is not used to supply base load, it is always shut down in August when heat demand is low, resulting that the HP's consumption is only obtained from the external power grid.

According to KEA's technical catalog [14], a HP will release an average of 0.028kg of carbon dioxide for every 1kWh of heat it produces, therefore, in the CO₂ result

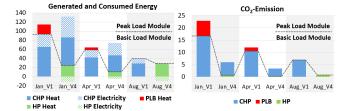


Fig. 6. Seasonal comparison of energy production and CO₂ emissions from the heating center in Variant-1 and Variant-4.

comparison diagram, CO₂ emissions of Variant-4 are converted by the corresponding heat production. However, since in January and April the electricity consumed by the HP was entirely generated by the CHP unit, the emission from the HP can be set to zero in these two time periods if both the CHP and the HP are running simultaneously, i.e. the CHP unit accounts for 100% of the CO₂ emission, i.e. 5.34t and 3.12t respectively. In August, the CO₂ emission is 0.84t, all caused by HP operation.

C. Overall Comparison and Rating of Heating Variants

Table II shows a detailed comparison of multiple aspects of each variant throughout the year and scores them in each aspect to make the comparison more intuitive. In each variant, the results are divided according to whether the excess electricity was sold to the area, and assume that the electricity is sold at the local basic supply tariff, which is 39.86ct/kWh in April 2024. In addition, based on the calculation results in Section IV-B, the results are scaled up to analyze and calculate for the whole year. The calculation results for January represent the two coldest months of the year, January and December, while the results for April account for the four transitional months February, March, April, and November. Lastly, the results for August reflect the six warmest months, from May to October.

1) CO₂ emissions: One of the evaluation key indicators of heating center variants is the reduction in emissions compared to the current system, measured in tons of CO₂ equivalent. The emissions of the existing status are calculated to be 136t/a. Compared to the existing status, Variant-2 reduces CO₂ emissions by nearly 70% due to the lower CO₂ equivalent of bio-methane in production, transport, and use compared to natural gas with the same produced energy. Variants-3 and 4 emit even less CO₂ due to the presence of an HP and, in Variant-4, H₂ as the energy source enables a 77.2% reduction.

2) Heat production costs: In the present paper, the lifetime of the heating center device is chosen as 15 years, and takes into account the investment cost, operation and

TABLE II. DATA SUMMARY AND COMPARISON OF ALL VARIANTS BASED ON KEY INDICATORS.

	Variant-1 Variant-2				Varia	ant-3	Variant-4					
	CHP-NG	+ PL-NG	CHP-BM + PL-BM			HP-Geo	+ PL-NG	HP-A + CHP-H2				
Electricity sales	N Y		N Y		Y	N		N		Y		
Heat generation - Base load	CHP		CHP			HP-Geothermal		HP-Air				
Heat generation - Peak load	Peak load boiler		Peak load boiler			Peak load boiler		CHP				
Energy source(s)	Natur	al gas	Biomethan			Electricity + Natural gas		Electricity + Hydrogen				
CO ₂ -Emissions [t]	136		41	+	41	+	33	+	31	++	31	++
Emission reduction compared to existing	0%		70%	+	70%	+	76%	++	77%	++	77%	++
Heat production costs k [ct/kWh]	14.86	1.83	21.55	-	8.53	-	12.23	++	26.33		16.51	-
Fuel energy consumed [MWh]	673		673	0	673	0	264	++	625	+	625	+
Fuel costs [€]	51817		87484		87484		49242	+	65187	-	65187	-
Fixed costs (investment and maintenance) [€]	27320		27320	0	27320	0	16269	++	75248		75248	
Flexibility and scalability	+		/	+	/	+	1	++	/	+	/	+
Security of supply	++		1	++	1	++	1	++	1	+	1	+

maintenance cost, and energy cost with the discount factor 1.0303 per year and the imputed interest rate 3.03% per year to calculate the annuity A_n [15]. To keep the power generation system costs comparable and understandable, fuel and equipment costs are also taken from the KEA technical catalog and the corresponding manufacturers. The heat production cost of the heating center is calculated as $k = |A_n| / Q_{used}$ [7].

Obviously, after selling the excess electricity to the area, the heating cost is greatly reduced. Due to the high price of bio-methane and the high investment and maintenance costs of H₂ energy equipment, Variant-3 becomes the most affordable solution if there is no electricity sold. This is because Variant-3 consumes only 264MWh of fuel (natural gas) to generate the same amount of electricity and heat as the other variants. It shows that the configuration of HPs can effectively reduce the consumption of fossil energy. However, the electricity required for the operation of the HP comes from the external power grid. If the required electricity is generated on-site by a photovoltaic system, the heating costs and CO₂ emissions can be further reduced. At the same time, the data in the table also shows the huge potential of H₂ and biomethane for emission reduction, and it provides diverse possibilities for regions with different resource reserves.

V. CONCLUSIONS

In this paper, the digital twin models of a heating system and buildings of a new residential area are developed and evaluated with alternative heating options considering various technologies and energy sources comprising renewables as H₂ driven heating pumps. For this purpose, simulation models of the heat grid pipelines, heating centers, heating transfer stations HIU, and modular and highly parameterizable building models are developed. Each heating center has an individually tailored control system and model. The cosimulation-based evaluation of energy consumption and production, costs, and carbon emissions from the perspective of the heat grid shows enormous economic savings potential and reduction of CO₂ emissions. Thus, the present study contributes to the planning and implementation of future sustainable and cost-efficient urban energy infrastructures.

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