



Forschungszentrum Karlsruhe
in der Helmholtz-Gemeinschaft

Institut für Reaktorsicherheit

Analysis of liquid phase turbulent kinetic energy balance for bubble-train flow

Milica Ilić, Martin Wörner, Dan Gabriel Cacuci

Sitzung der GVC-Fachausschüsse ,‘Mehrphasenströmungen‘, März 2003



Contents

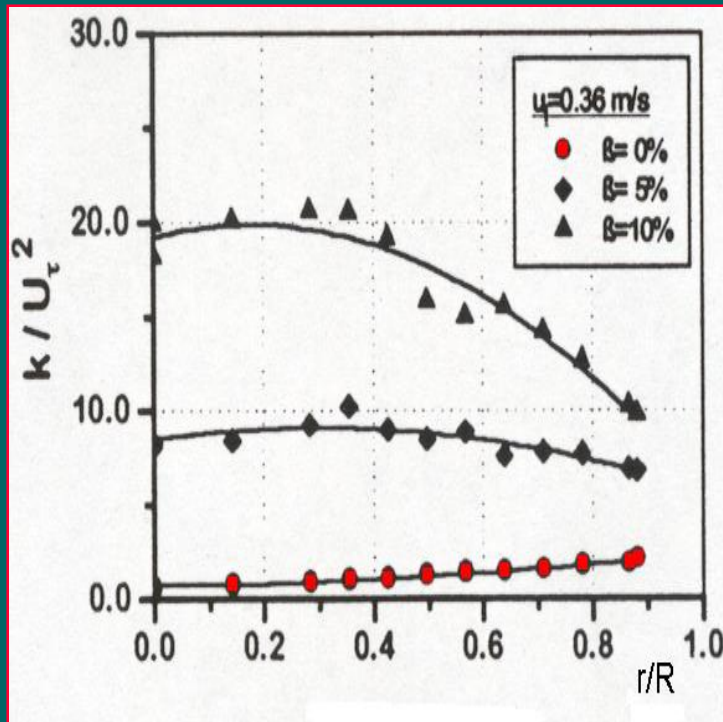
- Introduction
 - Phenomenon of bubble-induced turbulence (BIT)
 - Conservation equation of liquid turbulence kinetic energy (k_L equation)
- Analysis of BIT using DNS data
 - TURBIT-VoF computer code
 - Bubble-train flow numerical experiment
 - Methodologies to evaluate balance terms in k_L equation*
 - Budget of k_L equation for bubble-train flow
 - Scrutiny and validation of closure assumptions in modelling BIT
- Conclusions and further steps

Bubble-induced turbulence

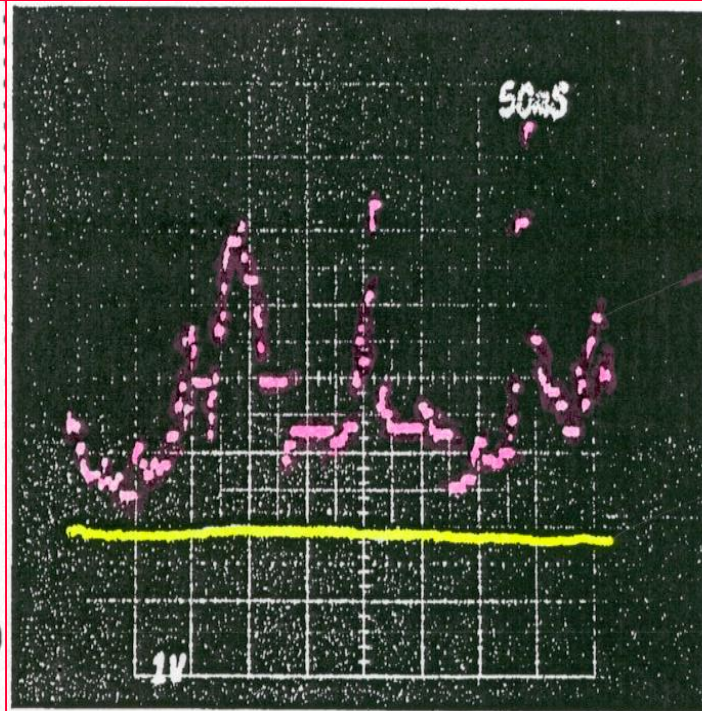
DEFINITION OF BUBBLE-INDUCED TURBULENCE:

Nonlinearity of the flow; Discrete buoyancy distribution
Bubble wakes; Deformation of interfaces

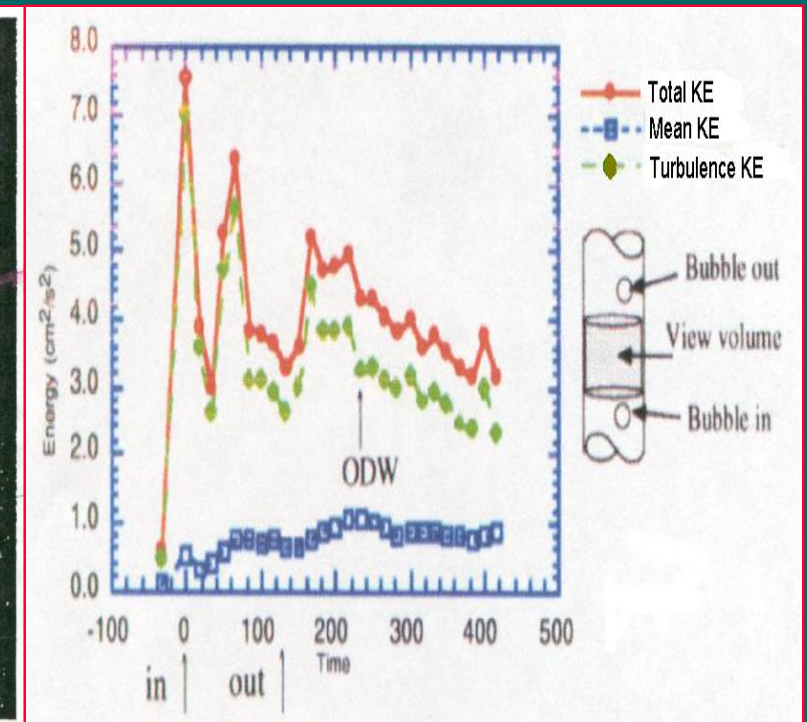
ILLUSTRATIVE EXAMPLES:



High Re liquid flow



Low Re liquid flow



Originally stagnant liquid

Liquid phase turbulence kinetic energy

1. Definition: $k_L = \overline{u_{L\alpha}'^2} / 2$
2. Conservation equation (Kataoka and Serizawa, 1989):

$$\frac{D}{Dt} (\overline{\Phi_L k_L}) = \underbrace{-\frac{\partial}{\partial x_\alpha} (\overline{\Phi_L \overline{p_L' u_{L\alpha}'}}) - \frac{\partial}{\partial x_\beta} \left(\frac{1}{2} \overline{\Phi_L \overline{u_{L\alpha}'^2 u_{L\beta}'}} \right)}_{\text{DIFFUSION}} + \frac{1}{Re_{ref}} \frac{\partial}{\partial x_\beta} \left(\overline{\Phi_L \frac{\partial k_L}{\partial x_\beta}} \right)$$

$\underbrace{-\overline{\Phi_L \overline{u_{L\alpha}' u_{L\beta}'}} \frac{\partial \overline{u_{L\alpha}'}}{\partial x_\beta}}_{\text{PRODUCTION}}$	$\underbrace{-\frac{1}{Re_{ref}} \overline{\Phi_L \frac{\partial u_{L\alpha}'}{\partial x_\beta} \frac{\partial u_{L\alpha}'}{\partial x_\beta}}}_{\text{DISSIPATION}}$
$\underbrace{-\overline{p_{Lin}' u_{Lin\alpha}'} n_{Lin\alpha} a_{in} + \frac{1}{Re_{ref}} \overline{u_{Lin\alpha}'} \frac{\partial u_{Lin\alpha}'}{\partial x_\beta} n_{Lin\beta} a_{in}}_{\text{INTERFACIAL TERMS}}$	

$\overline{A_L}$ averaging

$\overline{\overline{A_L}} = \overline{A_L \Phi_L} / \overline{\Phi_L}$ phase averaging

$A_L' = A_L - \overline{A_L}$ $A_{Lin}' = A_{Lin} - \overline{A_{Lin}}$ fluctuation

$\overline{\Phi_L}$ liquid volumetric fraction

a_{in} interfacial area concentration

Data on local liquid flow field and interface topology evaluated by **DNS** !!!

DNS of bubbly flows

TURBIT-VoF computer code developed at IRS, FZK

$$(1) \quad \frac{\partial u_\alpha}{\partial x_\alpha} = 0$$

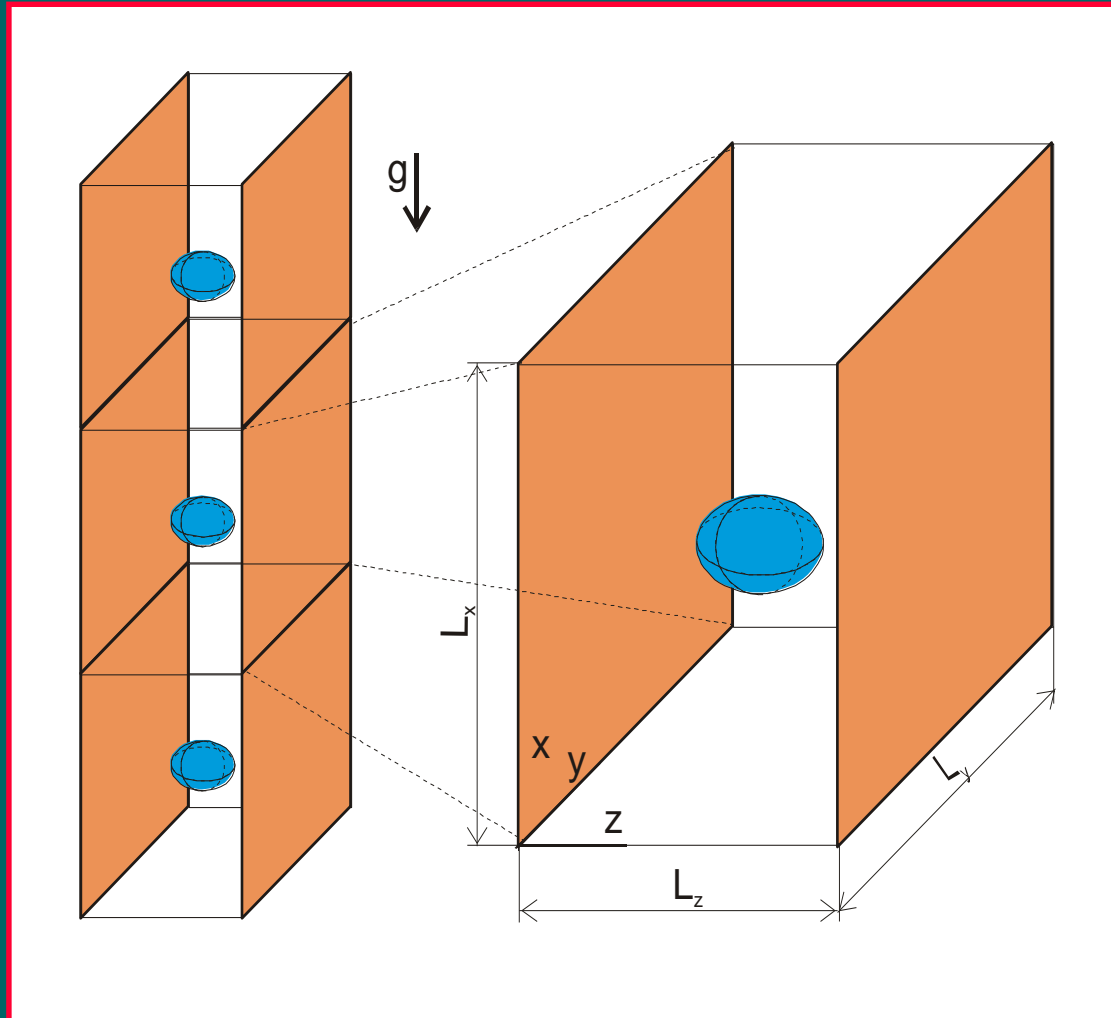
$$(2) \quad \frac{\partial(\rho u_\alpha)}{\partial t} + \frac{\partial(\rho u_\alpha u_\beta)}{\partial x_\beta} = -\frac{\partial p}{\partial x_\alpha} + \frac{1}{Re_{ref}} \frac{\partial \tau_{\alpha\beta}}{\partial x_\beta} - \frac{(1-f)E\ddot{o}_{ref}}{We_{ref}} \frac{g_\alpha^*}{g^*} + \frac{\kappa n_\alpha a_{in}}{We_{ref}}$$

$$(3) \quad \frac{\partial f}{\partial t} + \frac{\partial(u_\alpha f)}{\partial x_\alpha} = 0$$

Homogeneous mixture model in interfacial cells ($0 < f < 1$): $u_R = 0$, $p_L = p_G$

Interface evolution tracked using Volume-of-Fluid procedure

TURBIT-VoF numerical experiment



Definition of bubble-train flow

Computational domain: $1 \times 1 \times 1$
Grid: $64 \times 64 \times 64$

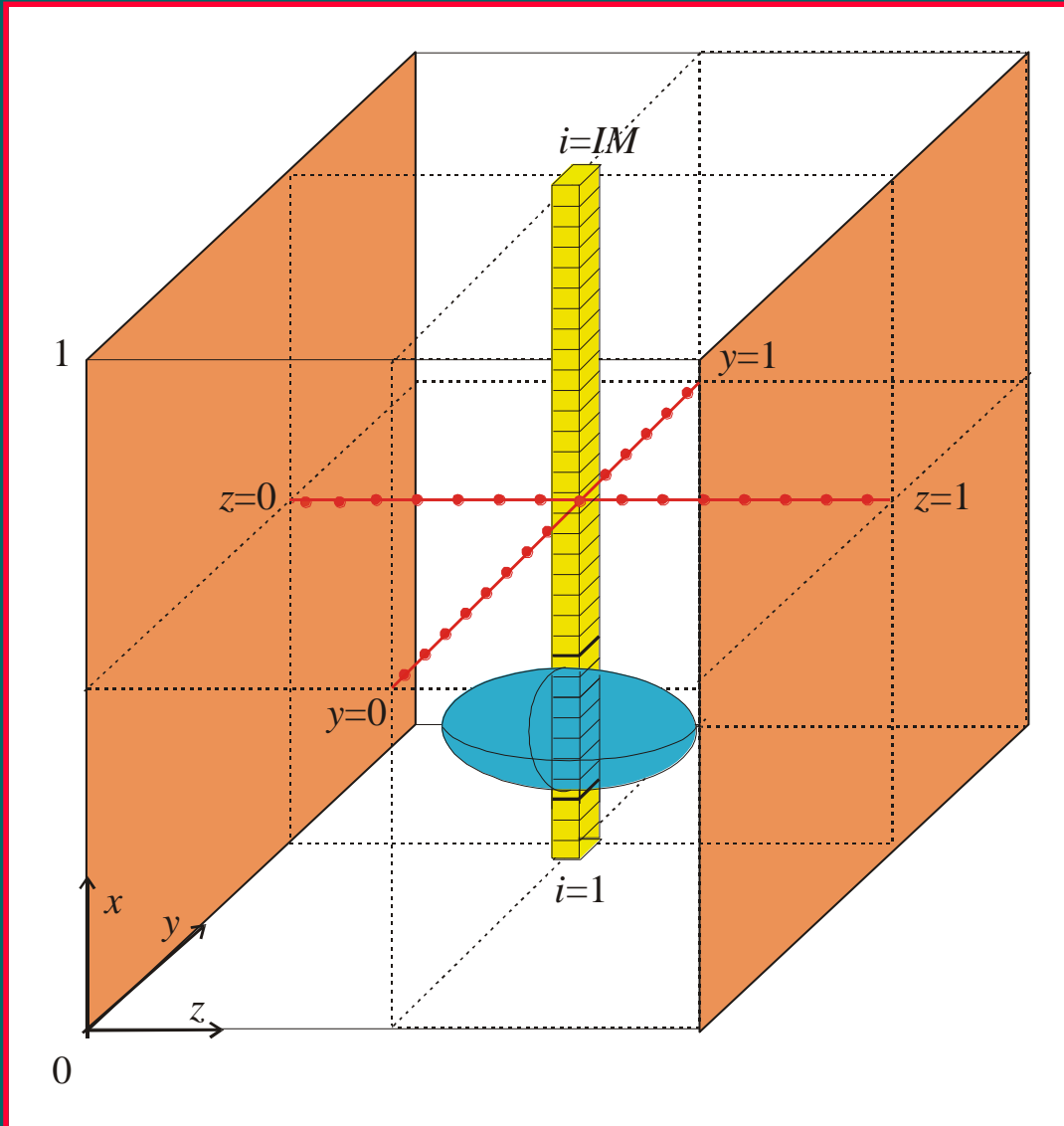
Bubble diameter: 0.25
Gas volumetric fraction: 0.818%

Phase density ratio: 0.5
Bubble Eötvös number: 3.065
Morton number: $3.06 \cdot 10^{-6}$

Time step: $1 \cdot 10^{-4}$
Number of time steps: 65 000
Bubble-path: approx. rectilinear
Bubble Reynolds number: ~ 125
Quasi-steady flow for time > 5.5

Data on [instantaneous](#) 3D flow field and interface topology are available !!!

Averaging procedure



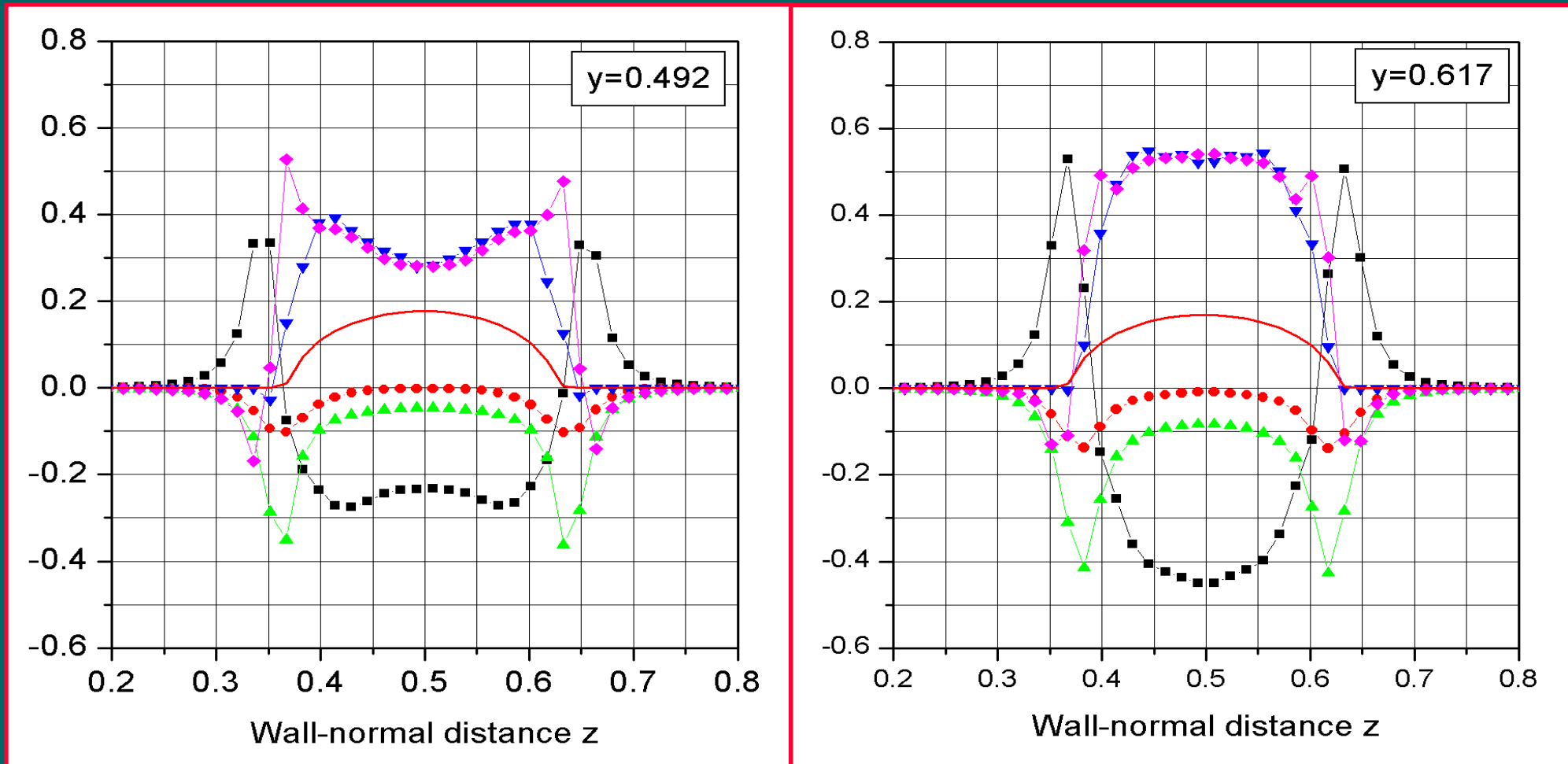
Ergodic theorem valid in x direction
 → Time averaging
 replaced by spatial-line averaging:

$$\bar{A}_L = \bar{A}_L^i(j,k) = \frac{\sum_{i=1}^{IM} A_{(i,j,k)}}{IM} \quad (1)$$

$$\bar{A}_L = \bar{A}_L^i(j,k) = \frac{\sum_{i=1}^{IM} f_{(i,j,k)} A_{(i,j,k)}}{\sum_{i=1}^{IM} f_{(i,j,k)}} \quad (2)$$

Balance terms in exact k_L equation

■ DIFFUSION (D)
 ● PRODUCTION (P)
 ▲ DISSIPATION (ε)
 ▼ INTERFACIAL TERMS
 ◆ $IFT^B = -(D+P+\varepsilon)$



Turbulence in bubbly flow is gained by **interfacial terms** and lost through **dissipation and production term !!!**

Modelling of turbulence in bubbly flows

Use of exact balance terms: validation of closure terms in modelled k_L equation

Models:

1. Laminar
2. Algebraic
3. One-equation ($k-l$)
4. Two-equation ($k-\varepsilon$)
5. Reynolds-stress ($\tau-\varepsilon$)

Modelled k_L equation:

$$\frac{D(\alpha_L k_L)}{Dt} = \underbrace{\frac{\partial}{\partial x_\beta} \left[\alpha_L v_L^{eff} \frac{\partial k_L}{\partial x_\beta} \right]}_{\text{DIFFUSION}} + \underbrace{\alpha_L \tau_{L\alpha\beta}^t \frac{\partial \overline{u_{L\alpha}}}{\partial x_\beta}}_{\text{PRODUCTION}} - \underbrace{\alpha_L \varepsilon_L}_{\text{DISSIPATION}} + \underbrace{IFT^M}_{\text{INTERFACIAL TERMS}}$$

Modelled versus exact production term

Modelling approaches:

- ◆ Pflieger and Becker, 2001
Grienberger and Hofmann, 1992
Svedsen et al., 1992

$$P = \alpha_L \left[2 \left(\frac{1}{Re_{ref}} + \nu_L^t \right) \bar{S}_{L\alpha\beta} \right] \frac{\partial \bar{u}_{L\alpha}}{\partial x_\beta}$$

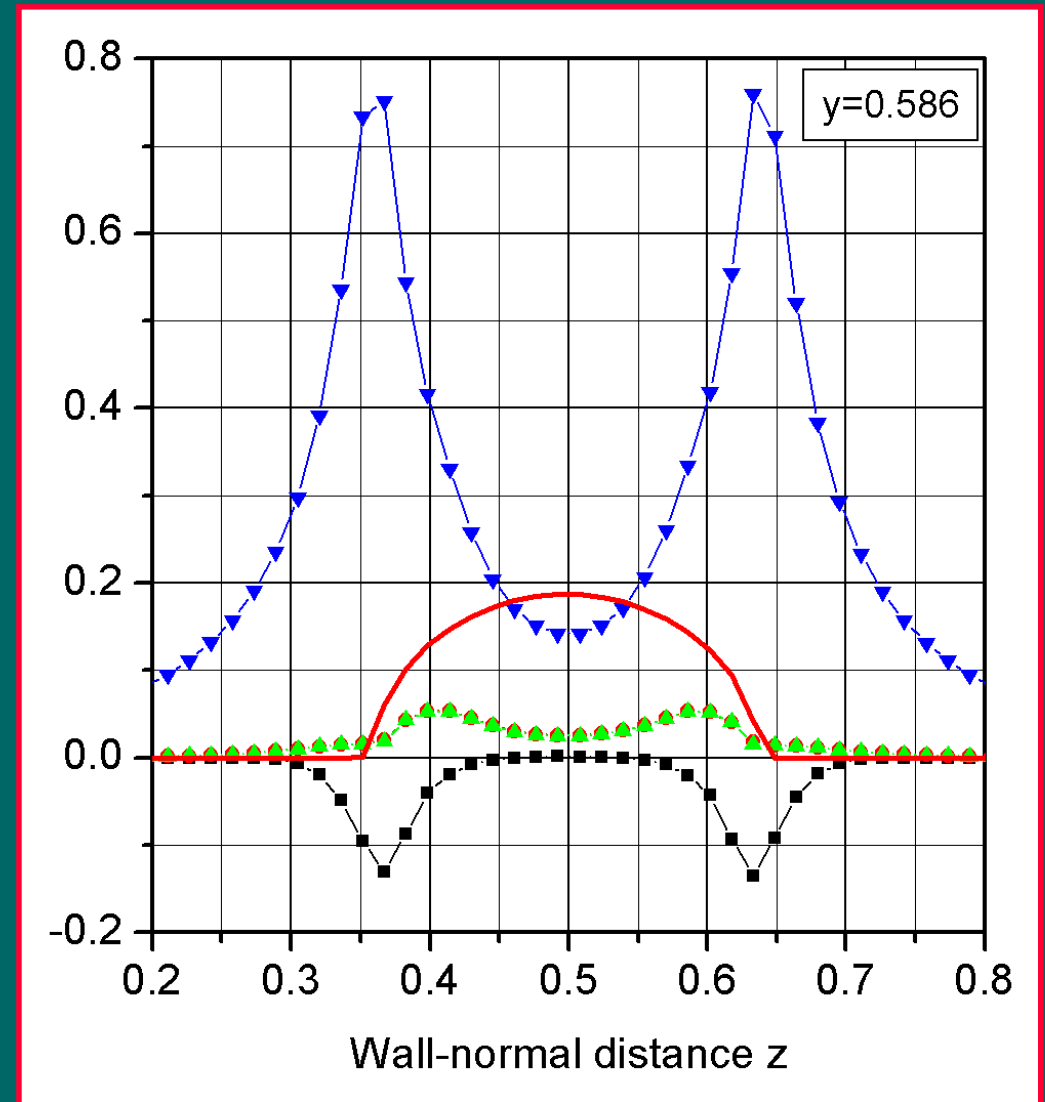
- De Bertodano et al., 1994
Boisson and Malin, 1996; Lain et al., 2001

$$P = \alpha_L \left[2 \nu_L^t \bar{S}_{L\alpha\beta} \right] \frac{\partial \bar{u}_{L\alpha}}{\partial x_\beta}$$

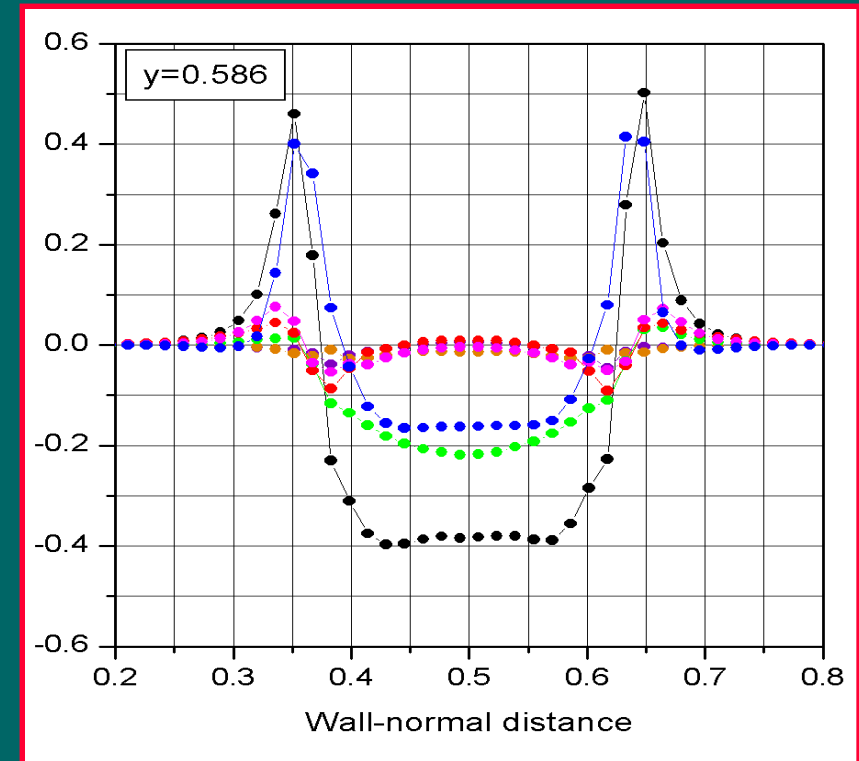
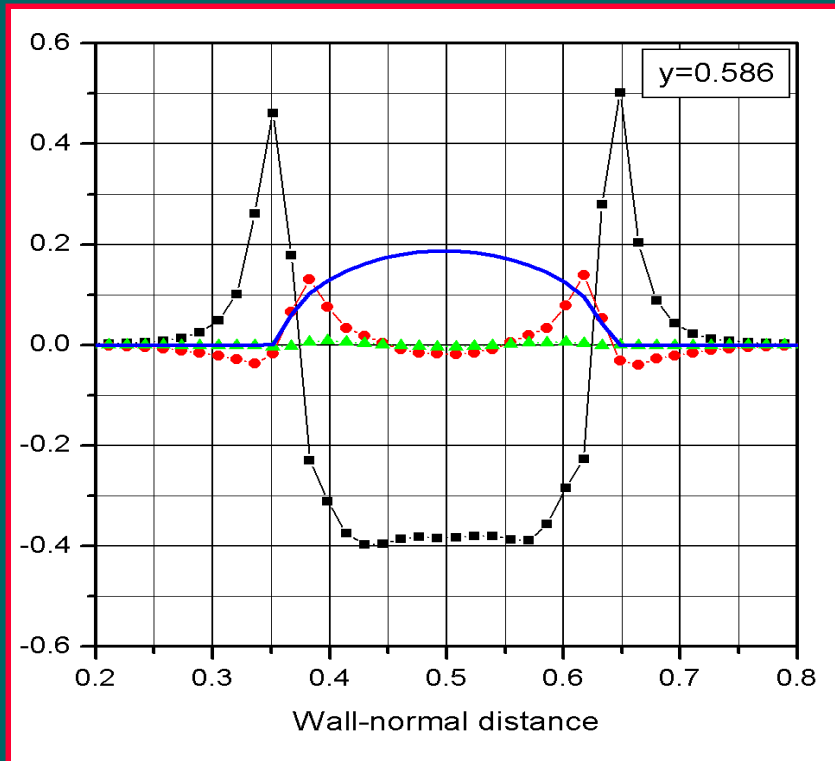
- 🕒 Troshko and Hassan, 2001
Morel, 1997; Hill et al., 1995

$$P = \alpha_L \left[2 \nu_L^t \bar{S}_{L\alpha\beta} k_L - \frac{2}{3} \left(k_L + \nu_L^t \frac{\partial \bar{u}_{L\alpha}}{\partial x_\beta} \right) \mathbf{I} \right] \frac{\partial \bar{u}_{L\alpha}}{\partial x_\beta}$$

Exact production term



Modelled versus exact diffusion term



- Troshko and Hassan, 2001
- De Bertodano et al., 1994
- Boisson and Malin, 1996;
- Morel, 1997;

$$\frac{\partial}{\partial x_\beta} \left[\alpha_L \left(\frac{1}{Re_{ref}} + \frac{v_L^t}{\sigma_k} \right) \frac{\partial k_L}{\partial x_\beta} \right]$$

- ▲ Pflieger and Becker, 2001

- Grienberger and Hofmann, 1992
- Svedsen et al., 1992; Hill et al., 1995

$$\frac{\partial}{\partial x_\beta} \left(\alpha_L \frac{v_L^t}{\sigma_k} \frac{\partial k_L}{\partial x_\beta} \right)$$

- Exact diffusion term

Pressure correlation sub-terms ● ●
 dominant in exact diffusion term ●

→ Closure for pressure correlation

$$\overline{\overline{p'_L u'_{L\beta}}} \propto -v_L^t \frac{\partial k_L}{\partial x_\beta}$$

not appropriate for bubble-driven flows

Interfacial terms modelling

Interfacial terms \propto work of interfacial forces

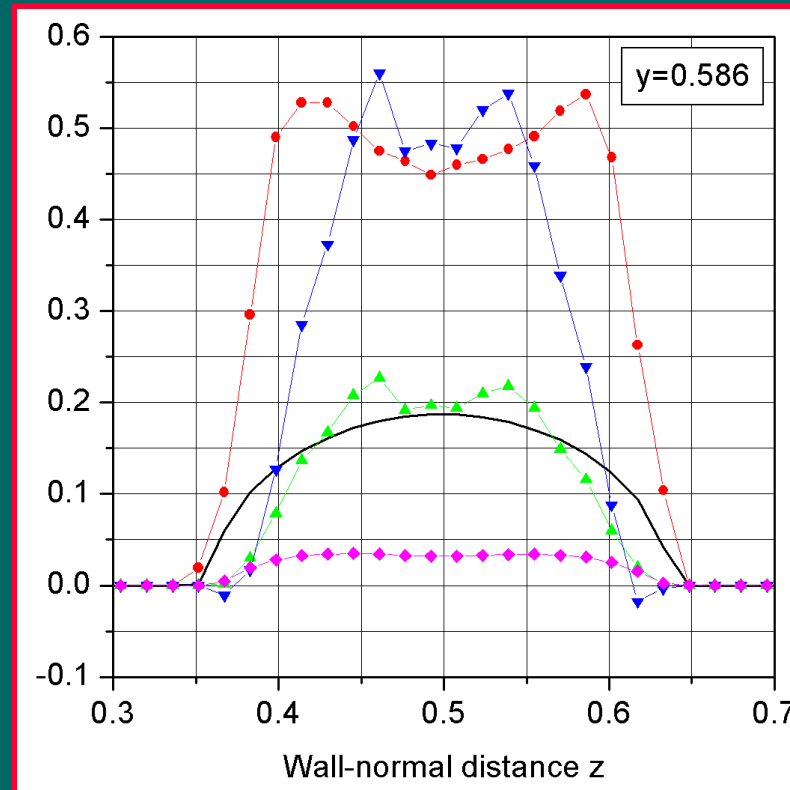
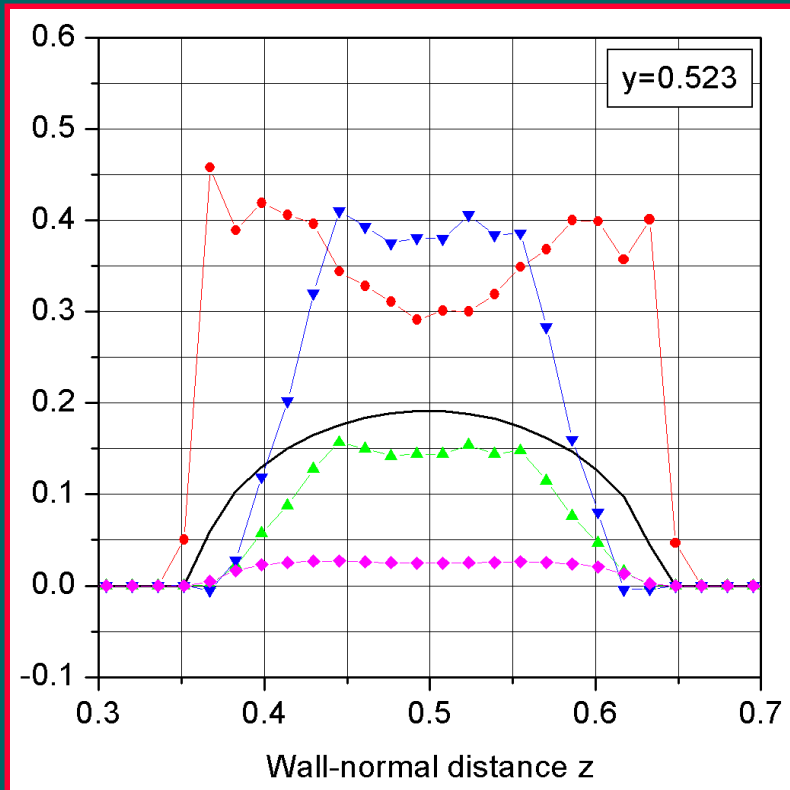
$$IFT = W^D + W^{ND}$$

Reference	<u>Work of drag force:</u> W^D	$M_{L\alpha}^D$	C_D
Morel, 1997	$M_{L\alpha}^D \bar{u}_{R\alpha}$	$\frac{3}{4} \alpha_G \frac{C_D}{D_b} \bar{U}_R \bar{u}_{R\alpha}$	$\frac{2}{3} \sqrt{E\ddot{o}_B} \cdot f(\alpha_L)$
de Bertodano et al., 1994			not given
Pfleger and Becker, 2001			0.44
Svendsen et al., 1995	$0.75 M_{L\alpha}^D \bar{u}_{R\alpha}$	not given	
Hill et al. 1995	$\frac{3}{4} \frac{\alpha_G C_D}{D_b} \bar{U}_R \left(\frac{\bar{u}_{R\alpha} \partial \alpha_G / \partial x_\alpha}{0.3 Re_{ref} \alpha_L \alpha_G} + 2k_L (C_t - 1) \right)$	not contained explicitly in W^D	$\frac{2}{3} \sqrt{E\ddot{o}_B} \cdot f(\alpha_L)$

Work of added-mass force:
(Morel, 1997)

$$W^{AM} = \frac{1}{2} \left(\bar{u}_{G\alpha} - \bar{u}_{L\alpha} \right) \frac{1 + 2\alpha_G}{\alpha_L} \alpha_G \left(\frac{D_G \bar{u}_{G\alpha}}{Dt} - \frac{D_L \bar{u}_{L\alpha}}{Dt} \right)$$

Modelled versus exact interfacial terms



Modelled IFT:

- ◆ Morel, 1997
- 🕒 Pfleger&Becker, 2001
- 📄 Hill et al. 1995
- Exact IFT

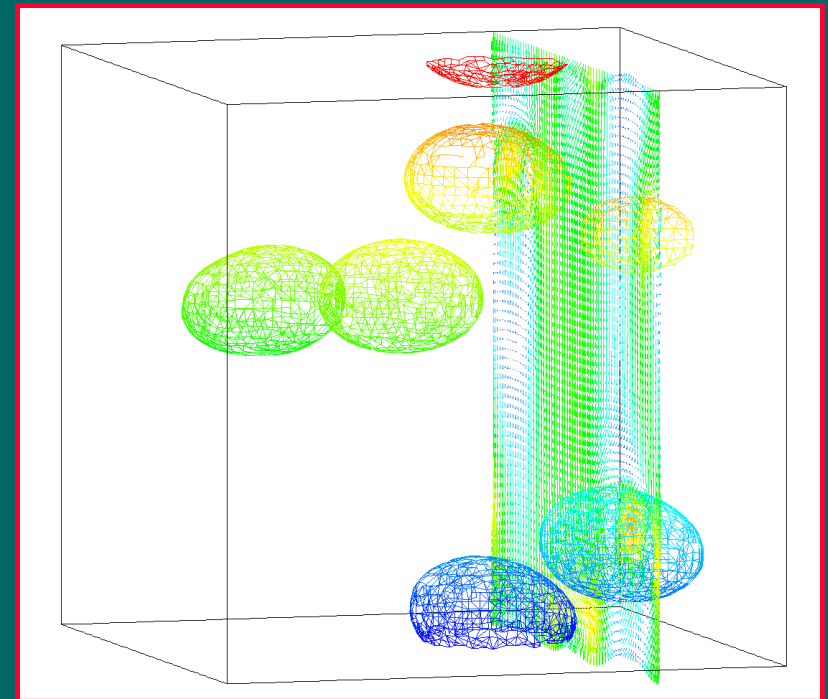
◆ Morel	$M_{L\alpha}^D = \frac{3}{4} \alpha_G \frac{C_D}{D_b} \bar{U}_R \bar{u}_{R\alpha}$	$W^D = M_{L\alpha}^D \bar{u}_{R\alpha}$	$C_D = \frac{2}{3} \sqrt{E\ddot{o}_B} \cdot f(\alpha_L)$
🕒 Pfleger&Becker		$W^D = 1.44 \alpha_L M_{L\alpha}^D \bar{u}_{R\alpha}$	$C_D = 0.44$

Importance of including local flow details in model assumptions for IFT !!!



Conclusions and future steps

- **Use of bubble-train flow DNS data to study BIT**
 - Production term is negative
 - Interfacial terms are only source terms
- **Closure assumptions**
 - Models for production and diffusion perform poor
 - Morels' model for interfacial terms performs well
- **Future steps**
 - DNS data for bubble swarm
 - Improvement of BIT models





Evaluation of interfacial terms

Definition:
$$IFT^E = -\overline{p'_{Lin} u'_{Lin\alpha} n_{Lin\alpha} a_{in}} + \frac{1}{Re_{ref}} \overline{u'_{Lin\alpha} \frac{\partial u'_{Lin\alpha}}{\partial x_\beta} n_{Lin\beta} a_{in}}$$

Fluctuation of interfacial quantity:
$$A'_{Lin} = A_{Lin} - \overline{\overline{A_L}}$$

Homogeneous mixture model in interfacial cells: $u_G = u_L = u$ and $p_G = p_L = p$

Liquid phase interfacial pressure $p_{Lin} = ?$

Assumption: $p_{Lin} \cong p_{(i,j,k)}$ where (i,j,k) is a neighbouring cell with $f_{(i,j,k)} = 1$

Liquid phase interfacial velocity $\vec{u}_{Lin} = ?$

No assumptions. Methodology for evaluation of \vec{u}_{Lin} is developed.

Evaluation of interfacial velocity

No phase change: $\vec{u}_{Lin} = \vec{u}_{Gin} = \vec{u}_{in}$

$$\vec{u}_{in} = \vec{u}_{int} + \vec{u}_{inn}$$

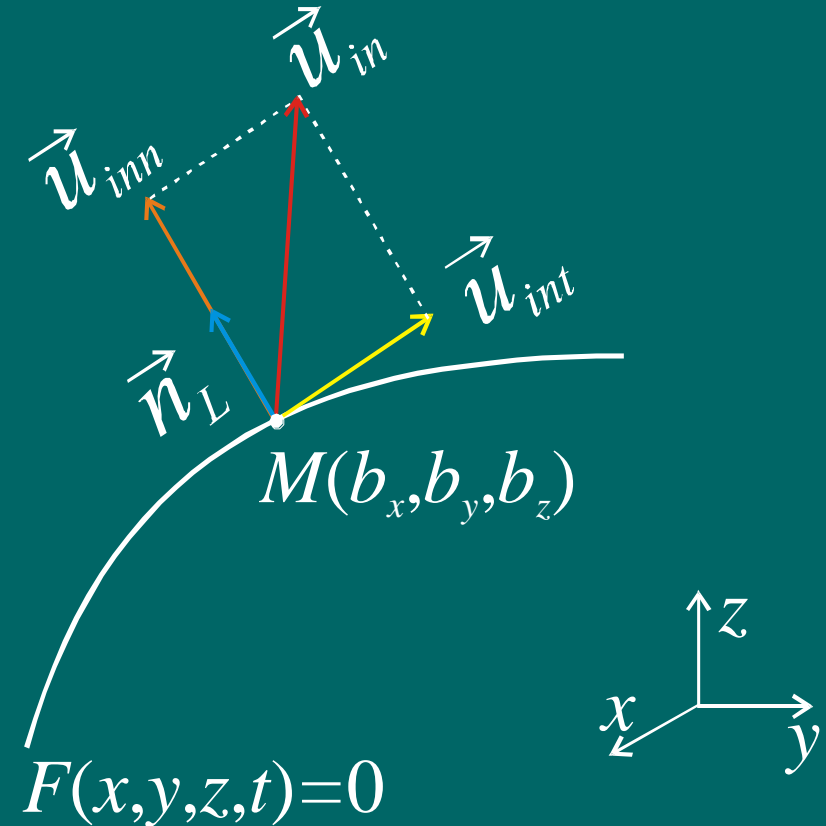
Tangential component (Ishii 1975):

$$\vec{u}_{int} = \vec{u}_t = \vec{u} - (\vec{u} \cdot \vec{n}_L) \cdot \vec{n}_L$$

Normal component (Kataoka et al., 1986):

$$\vec{u}_{inn} = (\vec{u}_{in} \cdot \vec{n}_L) \cdot \vec{n}_L$$

$$\vec{u}_{in} \cdot \vec{n}_L = \frac{\partial F / \partial t}{\sqrt{(\partial F / \partial x)^2 + (\partial F / \partial y)^2 + (\partial F / \partial z)^2}}$$



TURBIT-VoF definition: $F(x, y, z, t) = (b_x - x) \cdot n_{Lx} + (b_y - y) \cdot n_{Ly} + (b_z - z) \cdot n_{Lz} = 0$

$F(x, y, z, t)$ is not explicit function of $t \rightarrow \frac{\partial F}{\partial t} = ?$



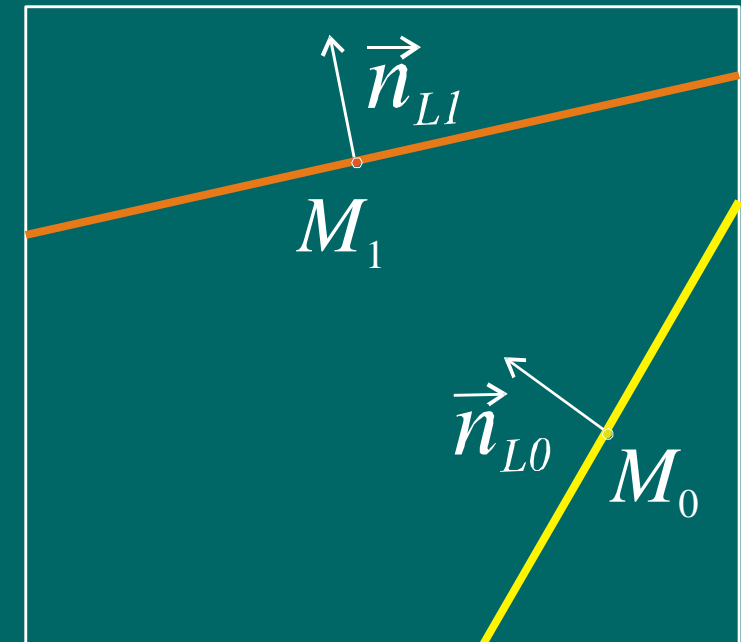
$$\frac{\partial F}{\partial t} = ?$$

$$(I) \quad F(b_{x0}, b_{y0}, b_{z0}, t_0) = 0$$

$$(II) \quad F(b_{x0} + \delta x, b_{y0} + \delta y, b_{z0} + \delta z, t_0 + \Delta t) = 0$$

1. Expand (I) in Taylor series
2. Neglect HOT
3. Subtract (II) from expanded (I)
4. Rearrange the difference

$$\rightarrow \quad \frac{\partial F}{\partial t} = -\frac{1}{\Delta t} (n_{Lx0} \delta x + n_{Ly0} \delta y + n_{Lz0} \delta z)$$



Idea comes from **experimental** determination of interfacial velocity. There M_0 and M_1 are fixed (sensor positions) and Δt is variable (Kataoka et al. 1986).