

Quantitative analysis of liquid phase turbulence kinetic energy equation using DNS data of bubble-train flow

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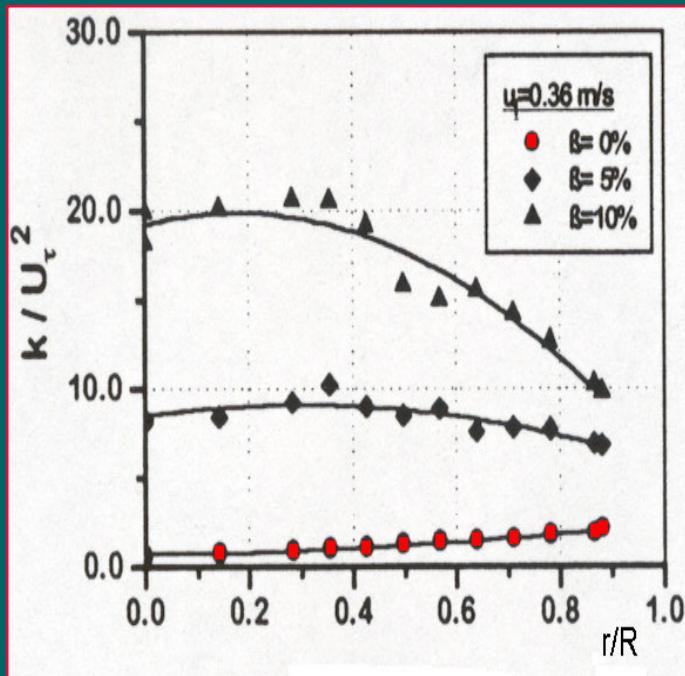
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Bubble-induced turbulence

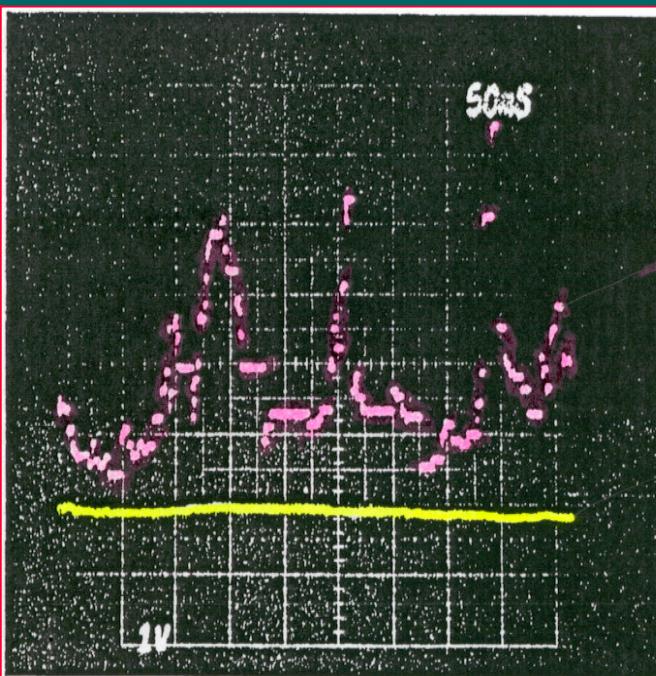
DEFINITION OF BUBBLE-INDUCED TURBULENCE:

Nonlinearity of the flow; Discrete buoyancy distribution
Bubble wakes; Deformation of interfaces

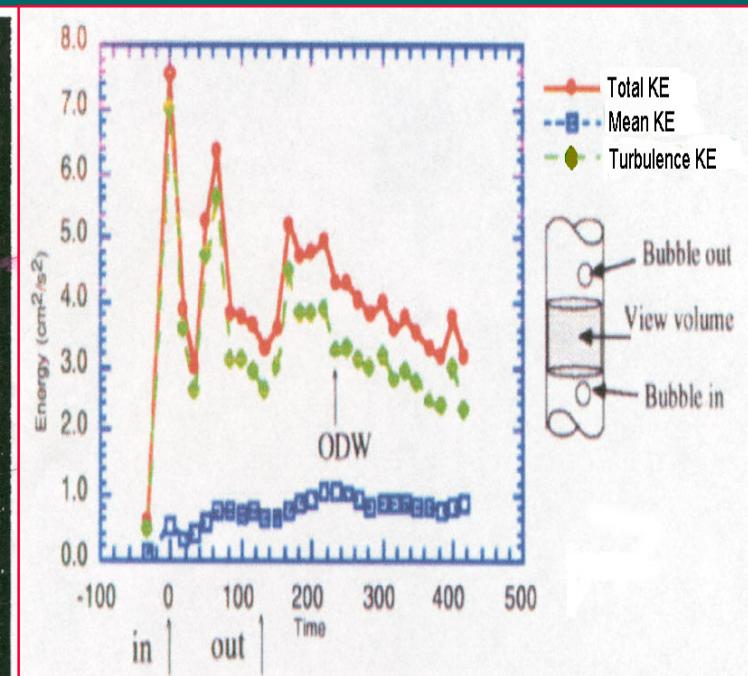
ILLUSTRATIVE EXAMPLES:



High Re liquid flow



Low Re liquid flow



Originally stagnant liquid

Conservation of liquid turbulence kinetic energy

1. Exact k_L equation (Kataoka and Serizawa, 1989):

$$\frac{D}{Dt}(\alpha_L k_L) = \underbrace{-\frac{\partial}{\partial x_\alpha} \left(\alpha_L \overline{\overline{p_L \dot{u}_{L\alpha}}} \right) - \frac{\partial}{\partial x_\beta} \left(\frac{1}{2} \alpha_L \overline{\overline{u_{L\alpha}^2 \dot{u}_{L\beta}}} \right)}_{\text{DIFFUSION}} + \frac{1}{Re_{ref}} \frac{\partial}{\partial x_\beta} \left(\alpha_L \frac{\partial k_L}{\partial x_\beta} \right) - \underbrace{\alpha_L \overline{\overline{\dot{u}_{L\alpha} \dot{u}_{L\beta}}} \frac{\partial \overline{\overline{u_{L\alpha}}}}{\partial x_\beta}}_{\text{PRODUCTION}}$$

$$\overline{\overline{A_L}} = \overline{A_L \Phi_L} / \Phi_L$$

$$\overline{\overline{A_L}} = A_L - \overline{\overline{A_L}}$$

$$\overline{\overline{A_{Lin}}} = A_{Lin} - \overline{\overline{A_L}}$$

$$\underbrace{-\frac{1}{Re_{ref}} \alpha_L \frac{\partial \overline{\overline{\dot{u}_{L\alpha}}}}{\partial x_\beta} \frac{\partial \overline{\overline{\dot{u}_{L\alpha}}}}{\partial x_\beta}}_{\text{DISSIPATION}} - \underbrace{\overline{\overline{p_{Lin} \dot{u}_{Lin\alpha} n_{Lin\alpha} a_{in}}} + \frac{1}{Re_{ref}} \overline{\overline{u_{Lin\alpha} \frac{\partial \dot{u}_{Lin\alpha}}{\partial x_\beta} n_{Lin\beta} a_{in}}}}_{\text{INTERFACIAL TERMS}}$$

α_L liquid volumetric fraction a_{in} interfacial area concentration

2. Modelled k_L equation (two-phase $k-\varepsilon$ model):

$$\frac{D(\alpha_L k_L)}{Dt} = \underbrace{\frac{\partial}{\partial x_\beta} \left[\alpha_L \nu_L^{eff} \frac{\partial k_L}{\partial x_\beta} \right]}_{\text{DIFFUSION}} + \underbrace{\alpha_L \tau_{L\alpha\beta}^t \frac{\partial \overline{\overline{u_{L\alpha}}}}{\partial x_\beta}}_{\text{PRODUCTION}} - \underbrace{\alpha_L \varepsilon_L}_{\text{DISSIPATION}} + \underbrace{IFT^M}_{\text{INTERFACIAL TERMS}}$$

DNS of bubbly flows

TURBIT-VoF computer code developed at IRS, FZK

(1)

$$\frac{\partial u_\alpha}{\partial x_\alpha} = 0$$

(2)

$$\frac{\partial(\rho u_\alpha)}{\partial t} + \frac{\partial(\rho u_\alpha u_\beta)}{\partial x_\beta} = -\frac{\partial p}{\partial x_\alpha} + \frac{1}{Re_{ref}} \frac{\partial \tau_{\alpha\beta}}{\partial x_\beta} - \frac{(1-f)E\ddot{o}_{ref}}{We_{ref}} \frac{g^*_\alpha}{g^*} + \frac{\kappa n_\alpha a_{in}}{We_{ref}}$$

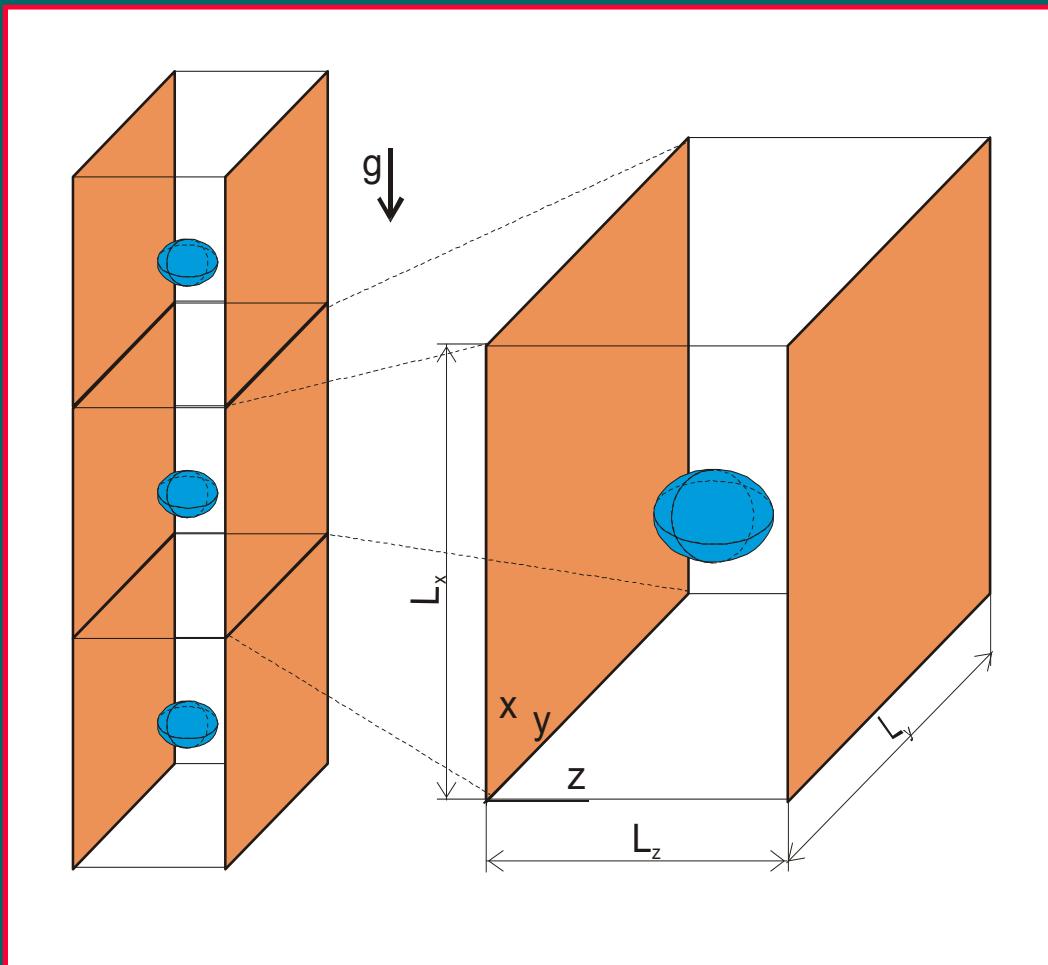
(3)

$$\frac{\partial f}{\partial t} + \frac{\partial(u_\alpha f)}{\partial x_\alpha} = 0$$

Homogeneous mixture model in interfacial cells ($0 < f < 1$): $u_R = 0$, $p_L = p_G$

Interface evolution tracked using Volume-of-Fluid procedure

Bubble-train flow numerical experiment



Computational domain: 1x1x1

Grid: 64x64x64

Bubble diameter: 0.25

Gas volumetric fraction: 0.818%

Phase density ratio: 0.5

Bubble Eötvös number: 3.065

Morton number: 3.06·10⁻⁶

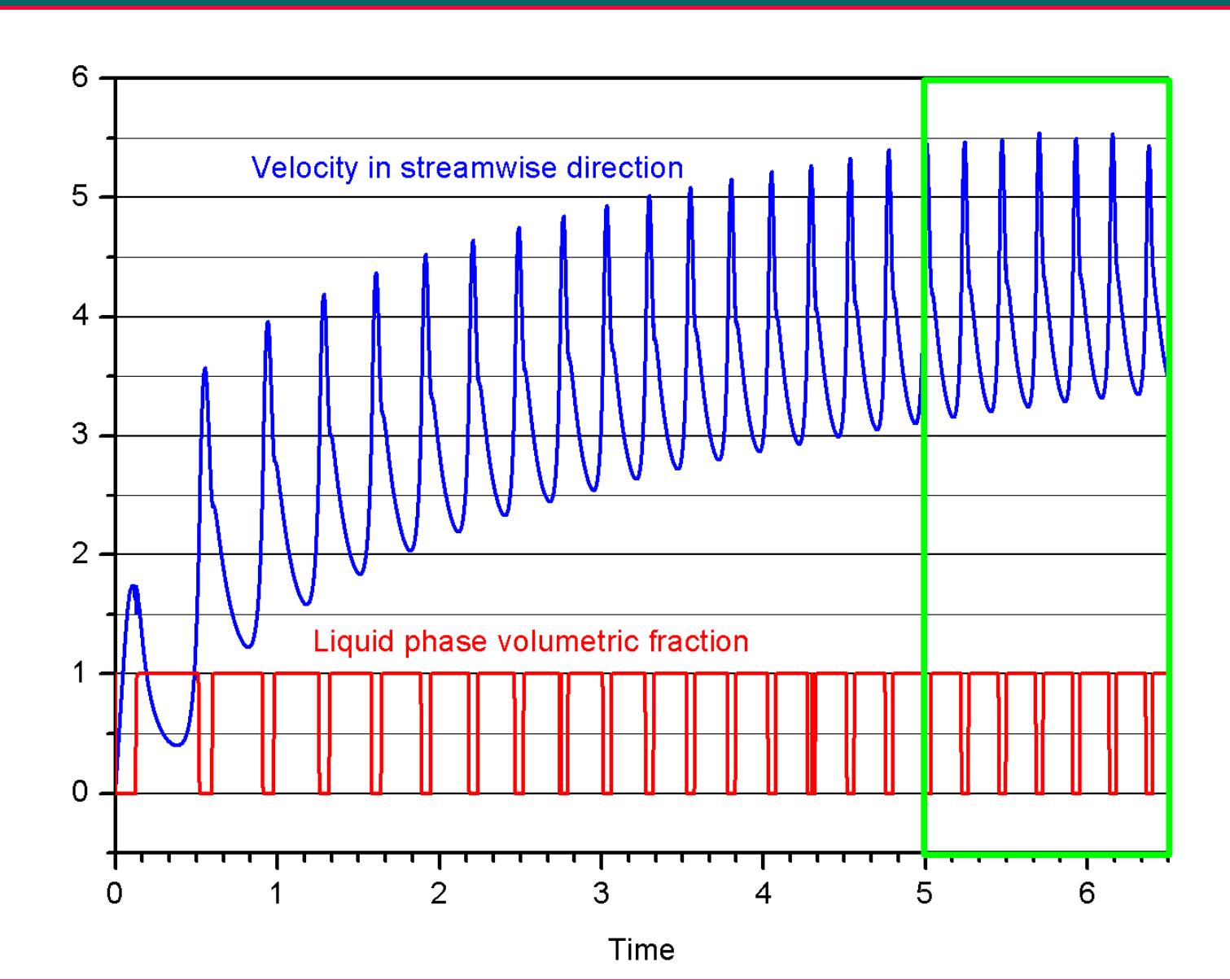
Time step: 1·10⁻⁴

Number of time steps: 65 000

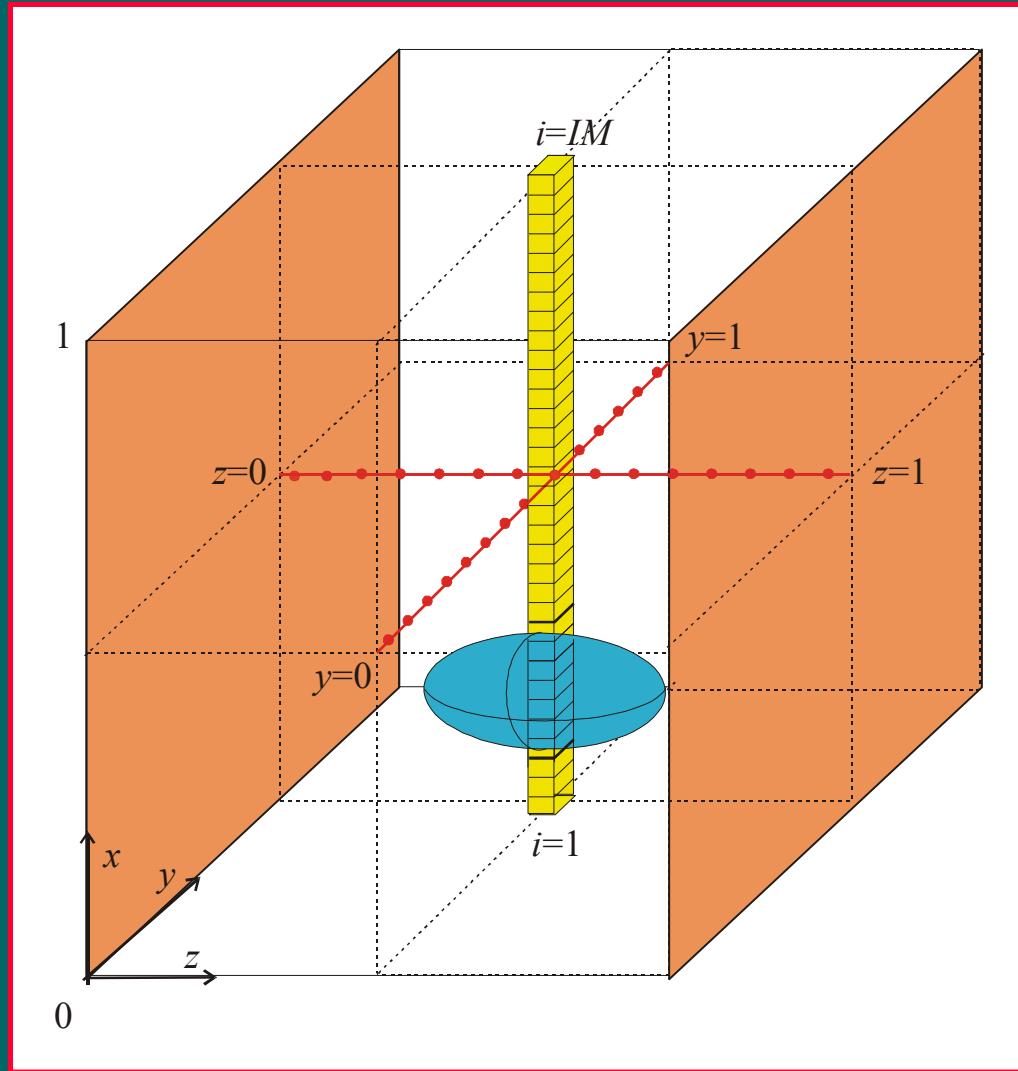
Bubble-path: approx. rectilinear

Bubble Reynolds number: ~125

TURBIT-VoF time signals



Averaging procedure



Ergodic theorem valid in x direction
 → Time averaging
 replaced by spatial-line averaging:

$$\bar{A}_L = \bar{A}_{L(j,k)}^i = \frac{\sum_{i=1}^{IM} A_{(i,j,k)}}{IM} \quad (1)$$

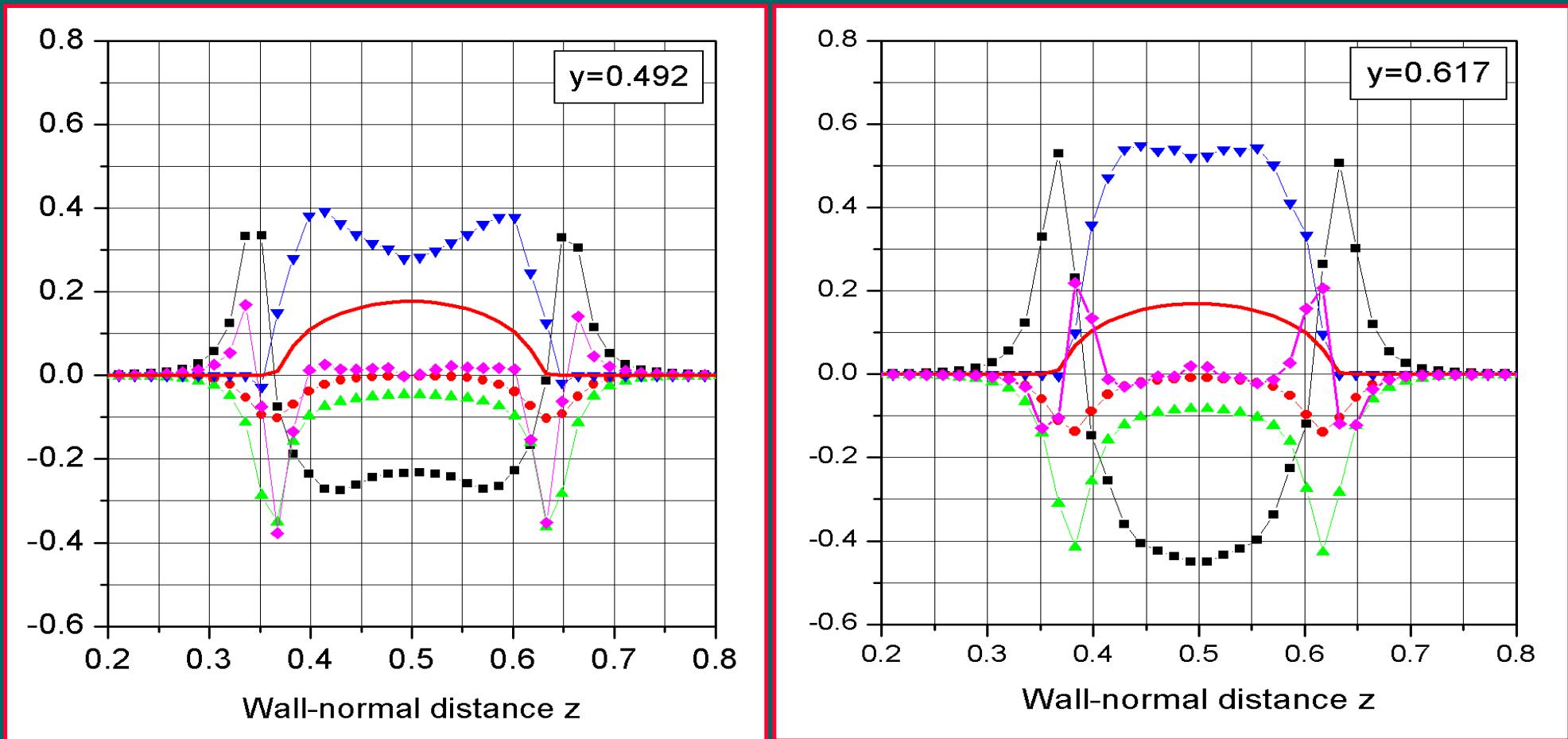
$$\bar{A}_L = \bar{A}_{L(j,k)} = \frac{\sum_{i=1}^{IM} f_{(i,j,k)} A_{(i,j,k)}}{\sum_{i=1}^{IM} f_{(i,j,k)}} \quad (2)$$

$$A'_L = A_L - \bar{A}_L \quad (3)$$

$$A'_{Lin} = A_{Lin} - \bar{A}_L \quad (4)$$

Balance terms in exact k_L equation

DIFFUSION PRODUCTION DISSIPATION INTERFACIAL TERMS out of balance — α_G



Turbulence in bubble-driven flows: gained by INTERFACIAL TERMS
lost through DISSIPATION and PRODUCTION

Modelled versus exact production term

Modelling approaches:

- ◆ Pfleger and Becker, 2001
Grienberger and Hofmann, 1992
Svedsen et al., 1992

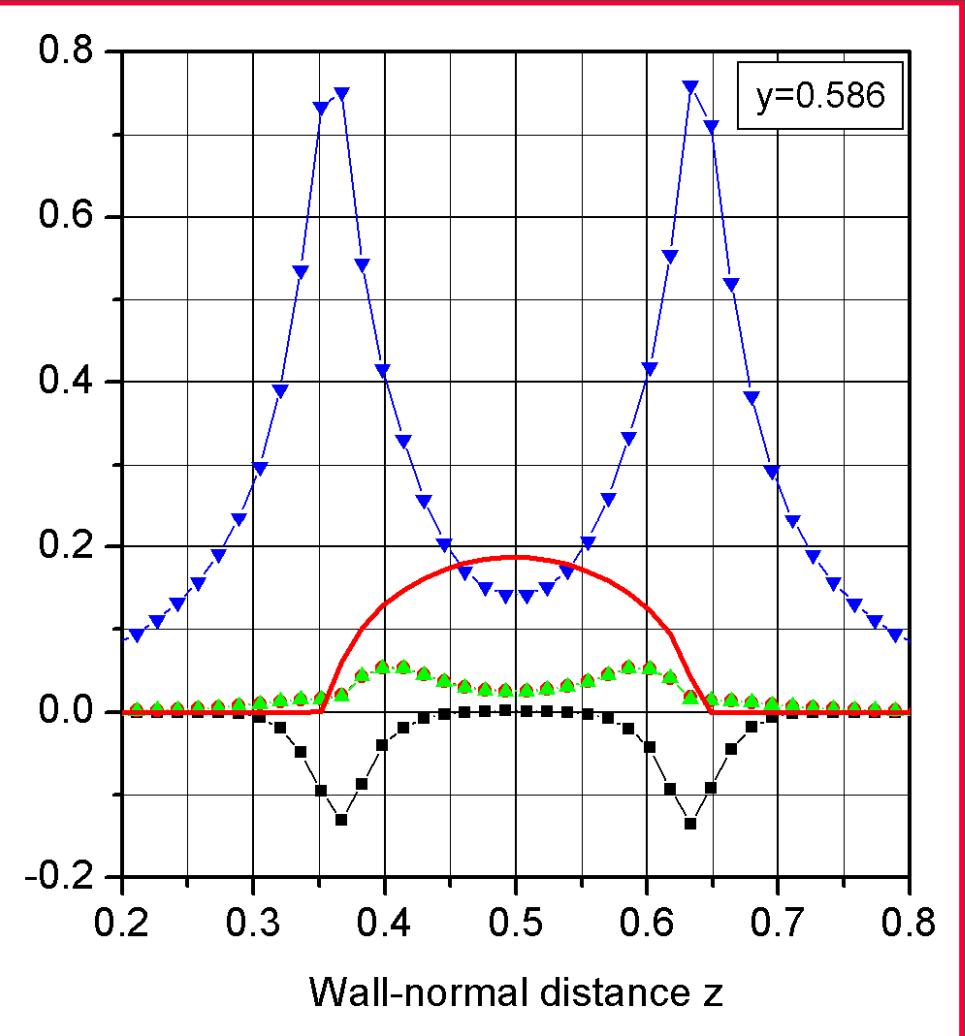
$$P = \alpha_L \left[2 \left(\frac{1}{Re_{ref}} + \nu_L^t \right) \bar{S}_{L\alpha\beta} \right] \frac{\partial \bar{u}_{L\alpha}}{\partial x_\beta}$$

- De Bertodano et al., 1994
Boisson and Malin, 1996; Lain et al., 2001

$$P = \alpha_L \left[2\nu_L^t \bar{S}_{L\alpha\beta} \right] \frac{\partial \bar{u}_{L\alpha}}{\partial x_\beta}$$

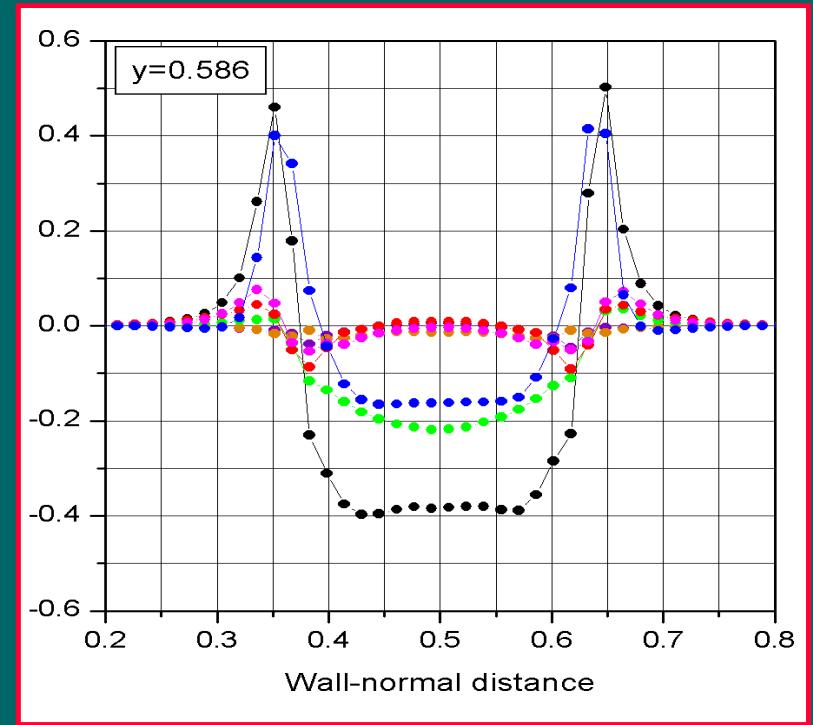
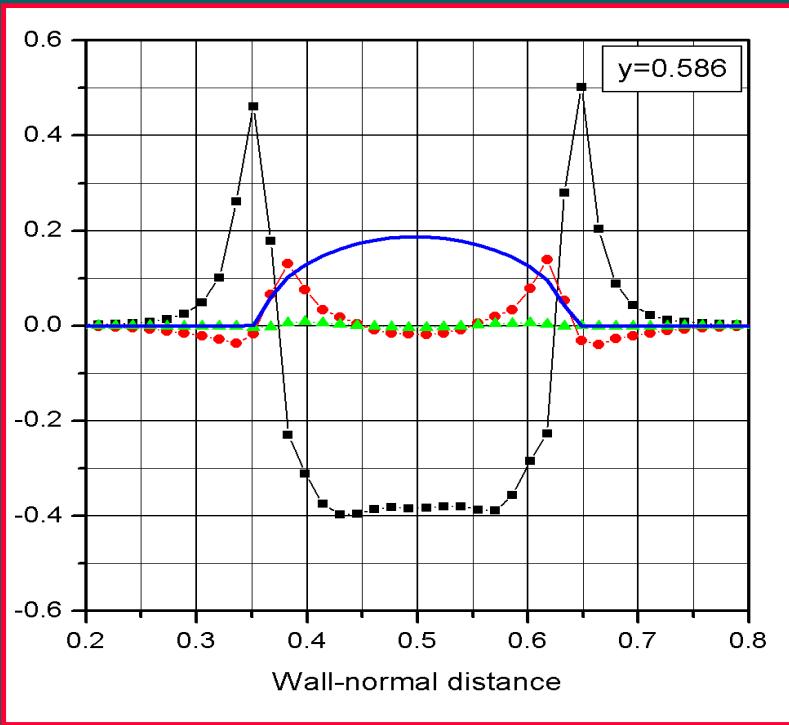
- ⌚ Troshko and Hassan, 2001
Morel, 1997; Hill et al., 1995

$$P = \alpha_L \left[2\nu_L^t \bar{S}_{L\alpha\beta} k_L - \frac{2}{3} \left(k_L + \nu_L^t \frac{\partial \bar{u}_{L\alpha}}{\partial x_\beta} \right) I \right] \frac{\partial \bar{u}_{L\alpha}}{\partial x_\beta}$$



Exact production term

Modelled versus exact diffusion term



- Troshko and Hassan, 2001
De Bertodano et al., 1994
Boisson and Malin, 1996;
 - Exact diffusion term
- $$\frac{\partial}{\partial x_\beta} \left[\alpha_L \left(\frac{1}{Re_{ref}} + \frac{v_L^t}{\sigma_k} \right) \frac{\partial k_L}{\partial x_\beta} \right]$$

- ▲ Pfleger and Becker, 2001
Grienberger and Hofmann, 1992
Svedsen et al., 1992; Hill et al., 1995
- $$\frac{\partial}{\partial x_\beta} \left(\alpha_L \frac{v_L^t}{\sigma_k} \frac{\partial k_L}{\partial x_\beta} \right)$$

Pressure corelation sub-terms ● ●
dominant in exact diffusion term ●
 → Closure for pressure correlation

$$\frac{1}{2} \overline{\overline{u_{L\alpha}^2 u_{L\beta}}} + \overline{\overline{p_L u_{L\beta}}} \propto -v_L^t \frac{\partial k_L}{\partial x_\beta}$$
 not appropriate for bubble-driven flows

Interfacial terms modelling

Interfacial terms \propto work of interfacial forces

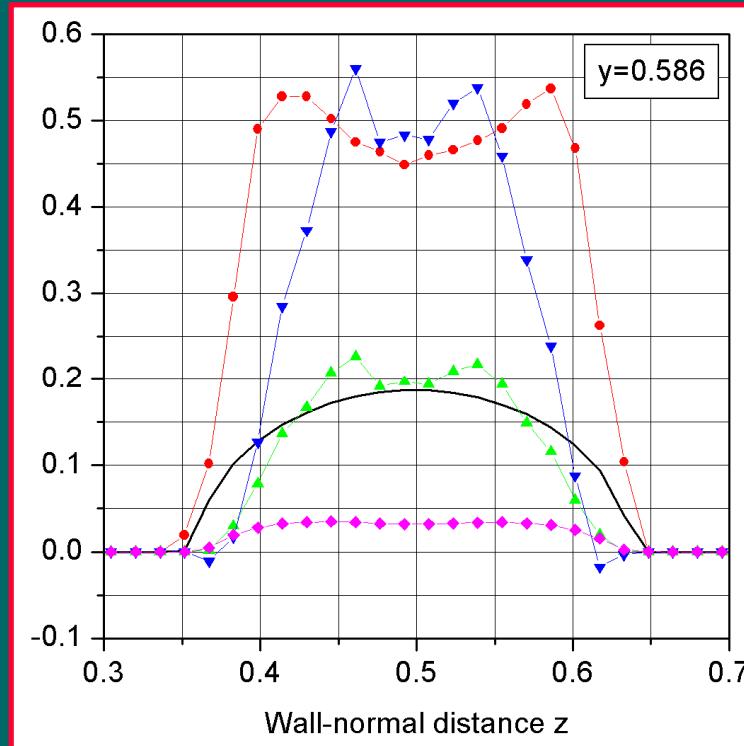
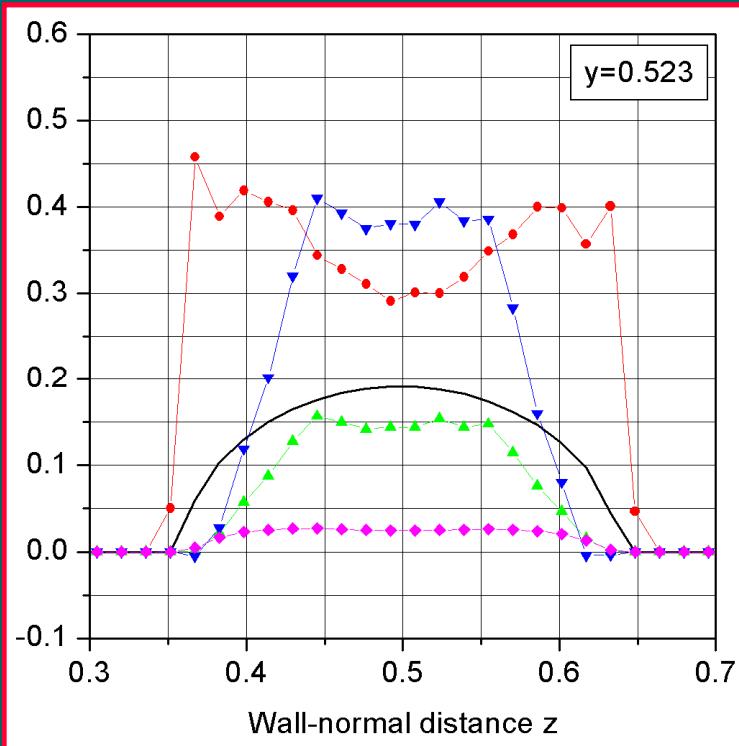
$$IFT = W^D + W^{ND}$$

Reference	<u>Work of drag force:</u> W^D	$M_{L\alpha}^D$	C_D
Morel, 1997	$M_{L\alpha}^D \bar{u}_{R\alpha}$		$\frac{2}{3} \sqrt{Eö_B} \cdot f(\alpha_L)$
de Bertodano et al., 1994		$\frac{3}{4} \alpha_G \frac{C_D}{D_b} \bar{U}_R \bar{u}_{R\alpha}$	not given
Pfleger and Becker, 2001	$1.44 \alpha_L M_{L\alpha}^D \bar{u}_{R\alpha}$		0.44
Svendsen et al., 1995	$0.75 M_{L\alpha}^D \bar{u}_{R\alpha}$		not given
Hill et al. 1995	$\frac{3}{4} \frac{\alpha_G C_D}{D_b} \bar{U}_R \left(\frac{\bar{u}_{R\alpha} \partial \alpha_G / \partial x_\alpha}{0.3 Re_{ref} \alpha_L \alpha_G} + 2k_L (C_t - 1) \right)$	not contained explicitly in W^D	$\frac{2}{3} \sqrt{Eö_B} \cdot f(\alpha_L)$

Work of added-mass force:
(Morel, 1997)

$$W^{AM} = \frac{1}{2} (\bar{u}_{G\alpha} - \bar{u}_{L\alpha}) \frac{1 + 2\alpha_G}{\alpha_L} \alpha_G \left(\frac{D_G \bar{u}_{G\alpha}}{Dt} - \frac{D_L \bar{u}_{L\alpha}}{Dt} \right)$$

Modelled versus exact interfacial terms



Modelled IFT:

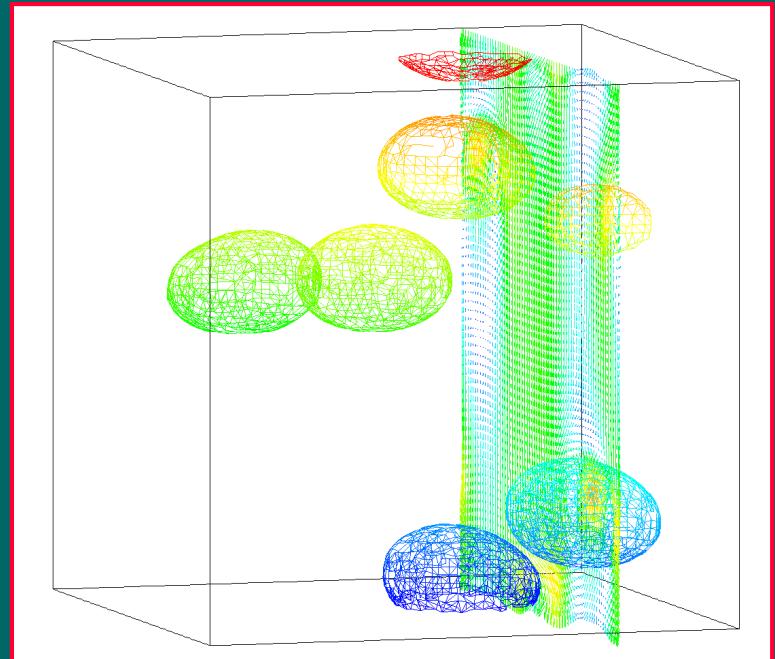
- Morel, 1997
- Pfleger&Becker, 2001
- Hill et al. 1995
- Exact IFT

Morel	$M_{L\alpha}^D = \frac{3}{4} \alpha_G \frac{C_D}{D_b} \bar{U}_R \bar{u}_{R\alpha}$	$W^D = M_{L\alpha}^D \bar{u}_{R\alpha}$	$C_D = \frac{2}{3} \sqrt{E\ddot{o}_B} \cdot f(\alpha_L)$
Pfleger&Becker		$W^D = 1.44 \alpha_L M_{L\alpha}^D \bar{u}_{R\alpha}$	$C_D = 0.44$

Importance of including local flow details in model assumptions for IFT !!!

F Conclusions and future steps

- **Use of bubble-train flow DNS data to study BIT**
 - Production term is negative
 - Interfacial terms are only source terms
- **Closure assumptions**
 - Models for production and diffusion perform poor
 - Morels' model for interfacial terms performs well
- **Future steps**
 - DNS data for bubble swarm
 - Improvement of BIT models



Evaluation of interfacial terms

Definition: $IFT^E = -\overline{\dot{p}_{Lin} \dot{u}_{Lin\alpha} n_{Lin\alpha} a_{in}} + \frac{1}{Re_{ref}} \overline{\dot{u}_{Lin\alpha} \frac{\partial \dot{u}_{Lin\alpha}}{\partial x_\beta} n_{Lin\beta} a_{in}}$

Fluctuation of interfacial quantity: $\dot{A}_{Lin} = A_{Lin} - \overline{\overline{A_L}}$

Homogeneous mixture model in interfacial cells: $u_G = u_L = u$ and $p_G = p_L = p$

Liquid phase interfacial pressure $p_{Lin} = ?$

Assumption: $p_{Lin} \cong p_{(i,j,k)}$ where (i,j,k) is a neighbouring cell with $f_{(i,j,k)} = 1$

Liquid phase interfacial velocity $\vec{u}_{Lin} = ?$

No assumptions. Methodology for evaluation of \vec{u}_{Lin} is developed.



Evaluation of interfacial velocity

No phase change: $\vec{u}_{Lin} = \vec{u}_{Gin} = \vec{u}_{in}$

$$\vec{u}_{in} = \vec{u}_{int} + \vec{u}_{inn}$$

Tangential component (Ishii 1975):

$$\vec{u}_{int} = \vec{u}_t = \vec{u} - (\vec{u} \cdot \vec{n}_L) \cdot \vec{n}_L$$

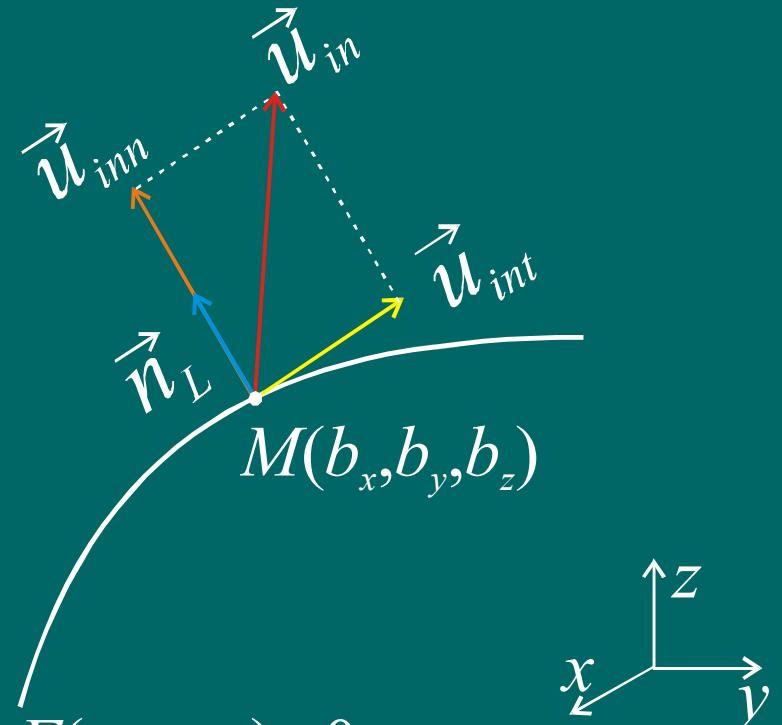
Normal component (Kataoka et al., 1986):

$$\vec{u}_{inn} = (\vec{u}_{in} \cdot \vec{n}_L) \cdot \vec{n}_L$$

$$\vec{u}_{in} \cdot \vec{n}_L = \frac{\partial F / \partial t}{\sqrt{(\partial F / \partial x)^2 + (\partial F / \partial y)^2 + (\partial F / \partial z)^2}}$$

TURBIT-VoF definition: $F(x, y, z, t) = (b_x - x) \cdot n_{Lx} + (b_y - y) \cdot n_{Ly} + (b_z - z) \cdot n_{Lz} = 0$

$F(x, y, z, t)$ is not explicit function of $t \rightarrow \frac{\partial F}{\partial t} = ?$



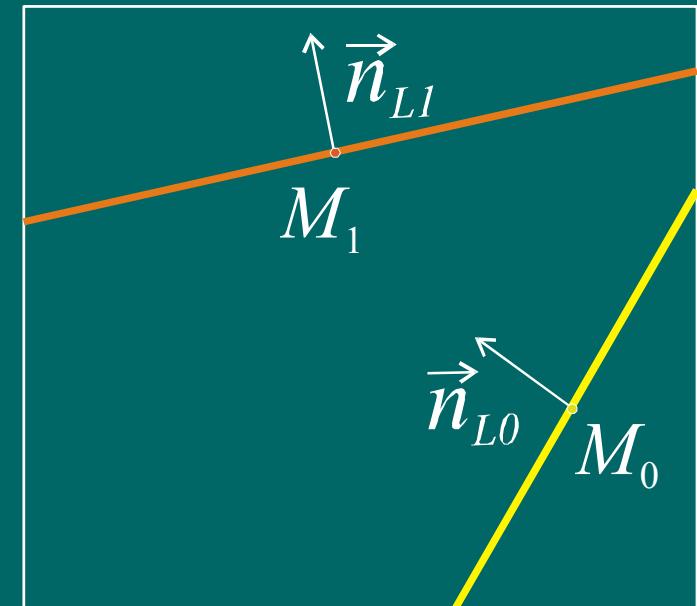
$$\frac{\partial F}{\partial t} = ?$$

$$(I) \quad F(b_{x0}, b_{y0}, b_{z0}, t_0) = 0$$

$$(II) \quad F(b_{x0} + \delta x, b_{y0} + \delta y, b_{z0} + \delta z, t_0 + \Delta t) = 0$$

1. Expand (I) in Taylor series
2. Neglect HOT
3. Subtract (II) from expanded (I)
4. Rearrange the difference

$$\rightarrow \frac{\partial F}{\partial t} = -\frac{1}{\Delta t} (n_{Lx0}\delta x + n_{Ly0}\delta y + n_{Lz0}\delta z)$$



Idea comes from experimental determination of interfacial velocity. There M_0 and M_1 are fixed (sensor positions) and Δt is variable (Kataoka et al. 1986).