

# **Volume-of-fluid method based numerical simulations of gas-liquid flow in confined geometries**

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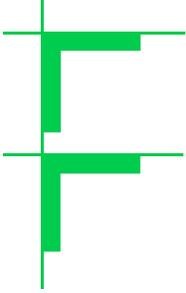
Research Center Karlsruhe, Institute for Reactor Safety, Germany

*IAHR/SHF Workshop on*

*“Advances in the Modelling Methodologies of Two-Phase Flows”*

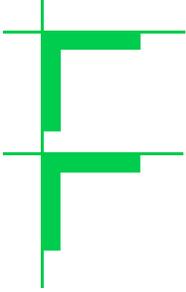
*Lyons, France, November 24 – 26, 2004*

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# Merits of DNS of gas-liquid flow

- Allows to get deeper insight into flow mechanisms and thus fosters physical understanding
  - Here: bubble train flow in square mini-channel
- Provides a complete database of the 3D velocity and pressure field and phase distribution with high spatial and temporal resolution
  - Here: analysis of liquid phase turbulence kinetic energy equation for bubble swarm flow



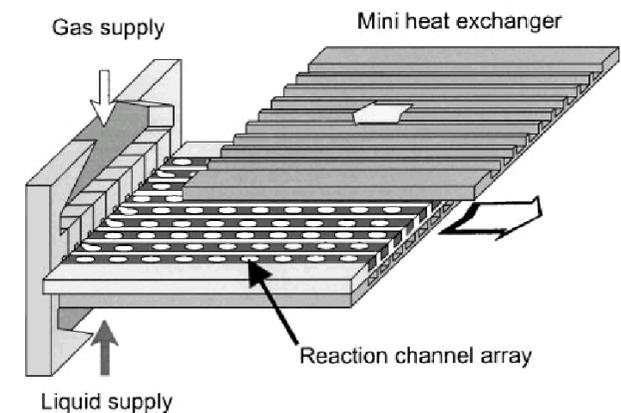
# In-house code TURBIT-VOF

- Volume-of fluid method for interface tracking
  - Interface is locally approximated by plane (PLIC method)
- Governing equations for two incompressible fluids
  - Single field momentum equation with surface tension term
  - Zero divergence condition for center-of-mass velocity
  - Advection equation for liquid volumetric fraction  $f$
- Solution strategy
  - Projection method resulting in pressure Poisson equation
  - Explicit third order Runge-Kutta time integration scheme
- Discretization in space
  - Finite volume formulation for regular staggered grid
  - Second order central difference approximations

# Multi-phase micro process engineering

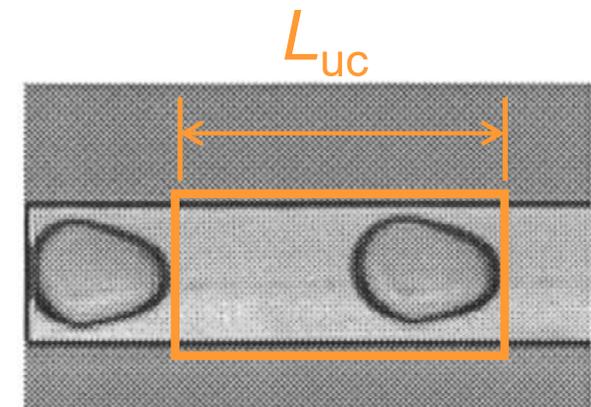
- Miniaturized devices offer certain advantages

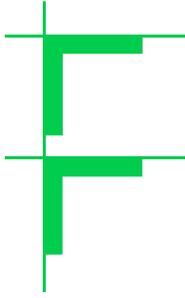
- High interfacial area per unit volume  
⇒ Enhanced heat and mass transfer
- Defined interface geometry  
⇒ Numbering up instead of scaling up
- Example: Micro bubble column\*



- Flow pattern in single channel: bubble train flow

- Bubbles of identical shape move with same velocity
- Flow is fully characterized by a single “unit cell” of length  $L_{uc}$





# Numerical set up

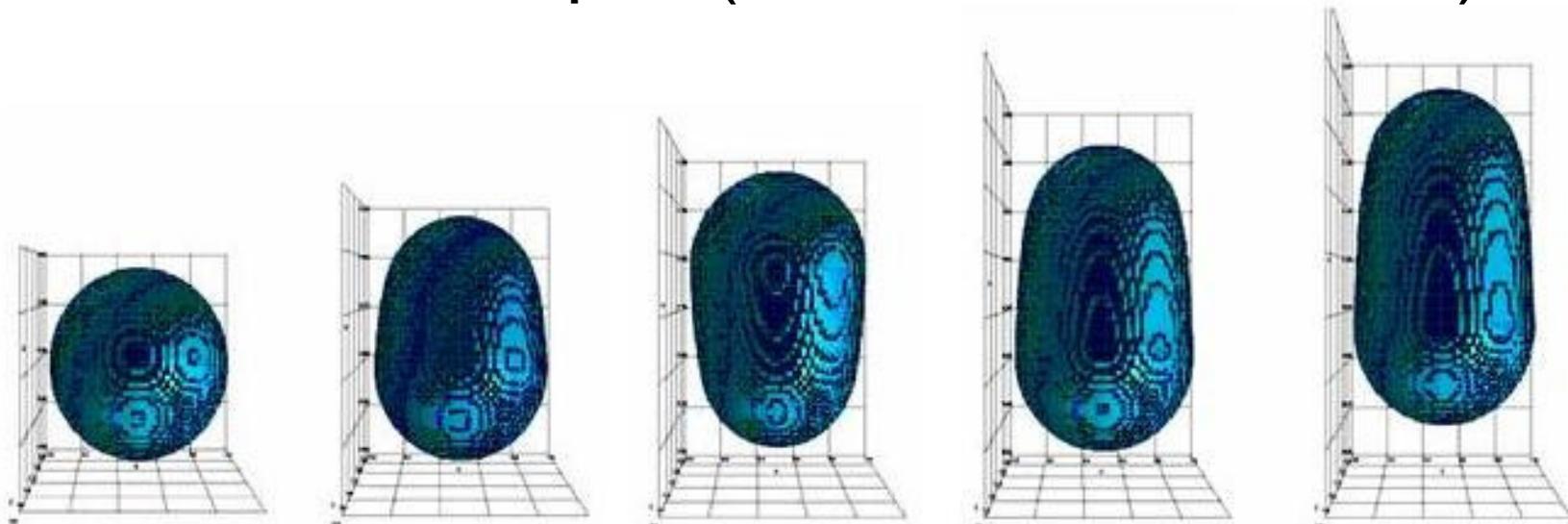
- Square vertical channel with cross section  $2 \text{ mm} \times 2 \text{ mm}$
- Consideration of one flow unit cell only
- Account for influence of trailing/leading unit cells by periodic boundary conditions in axial direction
- Length of flow unit cell,  $L_{uc}$ , is input parameter
- Flow is driven in vertical direction ( $y$ ) by specified axial pressure gradient and buoyancy

# Physical parameters

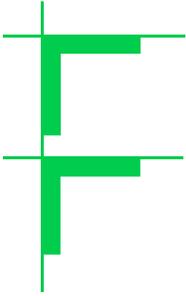
- Fluid properties Factor 10 higher than  $\rho$  and  $\mu$  of air

| $\rho_l$              | $\rho_g$               | $\mu_l$    | $\mu_g$                    | $\sigma$  |
|-----------------------|------------------------|------------|----------------------------|-----------|
| 957 kg/m <sup>3</sup> | 11.7 kg/m <sup>3</sup> | 0.048 Pa s | $1.84 \times 10^{-4}$ Pa s | 0.022 N/m |

- Initial bubble shapes (void fraction  $\varepsilon = 33\%$ )



- Simulations are started from gas and liquid at rest

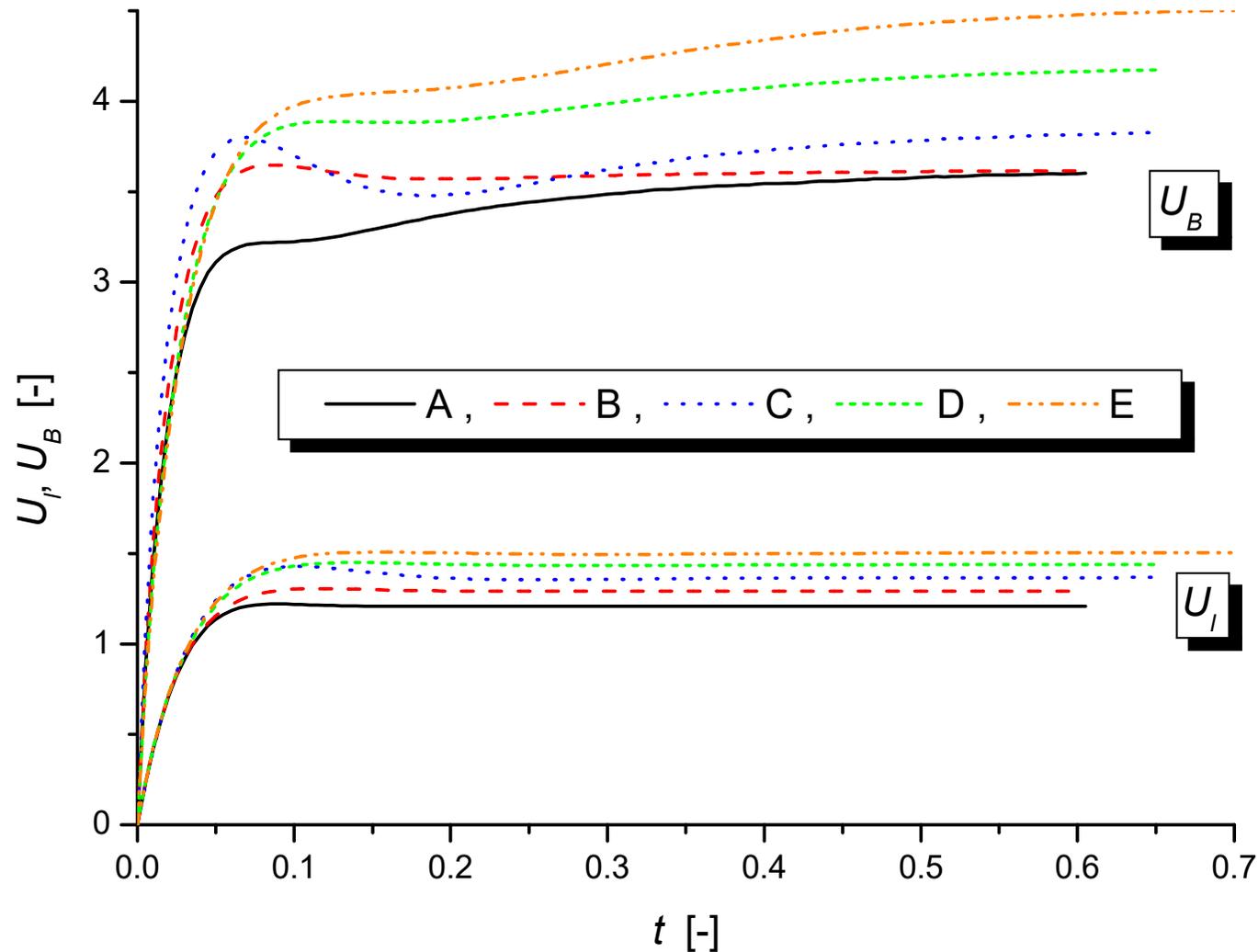


# Computational parameters

| Case | $L_{uc} / W$ | Domain                   | Grid                     | Time steps |
|------|--------------|--------------------------|--------------------------|------------|
| A1   | 1            | $1 \times 1 \times 1$    | $48 \times 48 \times 48$ | 24,000     |
| A2   | 1            | $1 \times 1 \times 1$    | $64 \times 64 \times 64$ | 60,000     |
| B    | 1.25         | $1 \times 1.25 \times 1$ | $48 \times 60 \times 48$ | 24,000     |
| C    | 1.5          | $1 \times 1.5 \times 1$  | $48 \times 72 \times 48$ | 26,000     |
| D    | 1.75         | $1 \times 1.75 \times 1$ | $48 \times 84 \times 48$ | 26,000     |
| E    | 2            | $1 \times 2 \times 1$    | $48 \times 96 \times 48$ | 28,000     |

Results on both grids show only slight differences

# Time history of mean velocities



Steady state values of bubble velocity  $U_B$  and mean liquid velocity  $U_l$  increase with increasing length of the flow unit cell

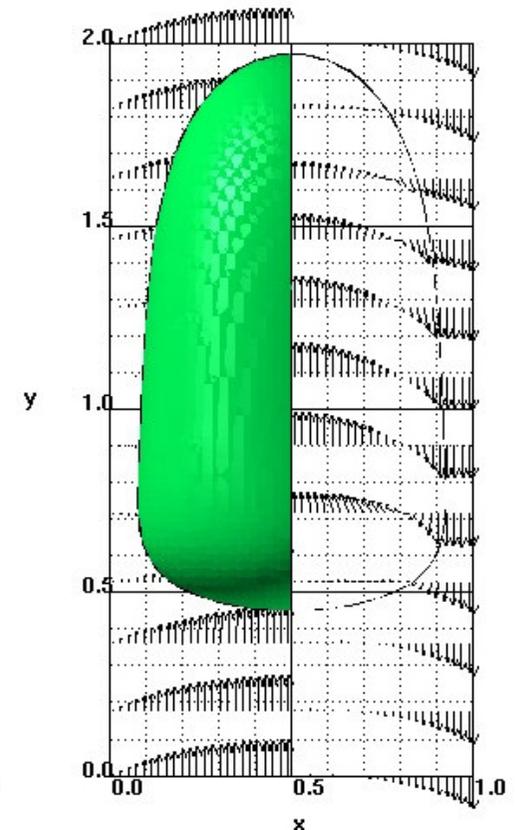
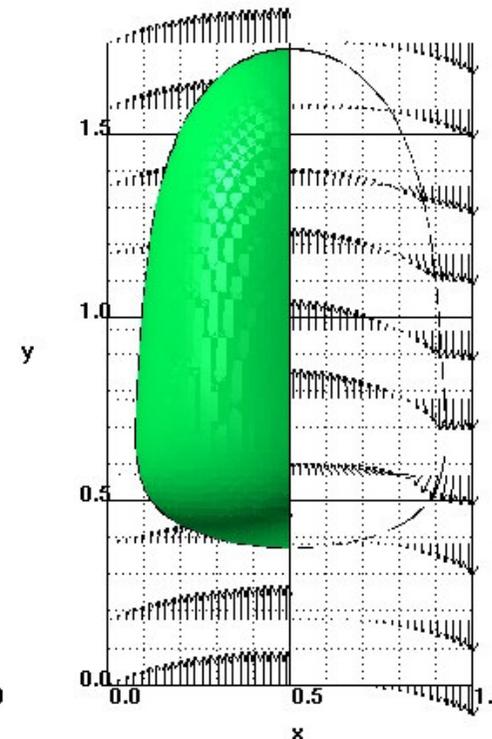
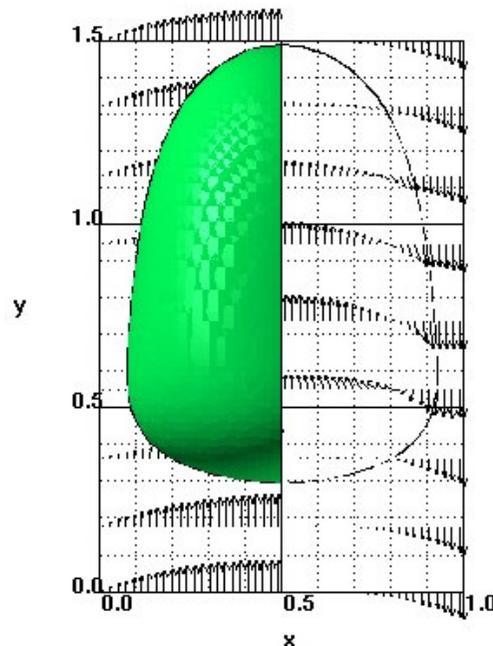
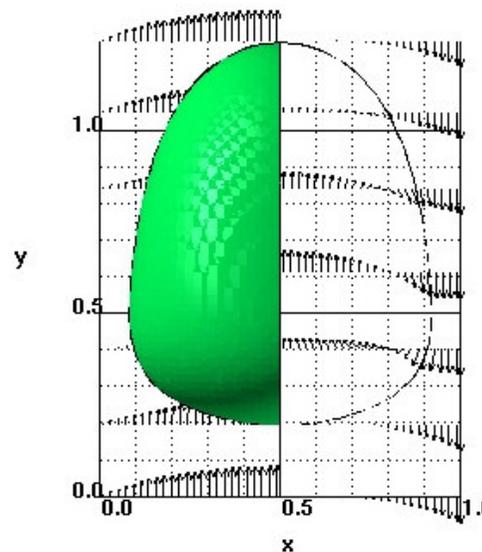
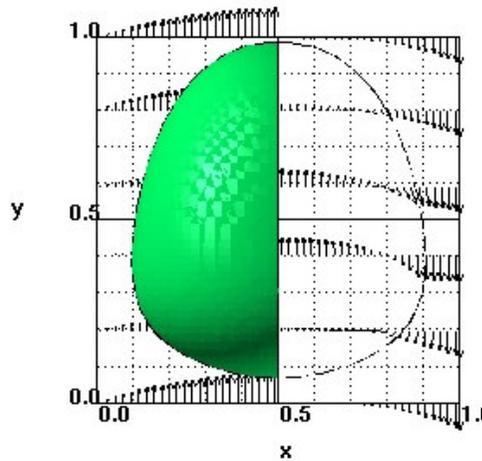


# Bubble shape and velocity field

## Velocity field in vertical mid-plane

Right half: frame of reference moving with bubble

Left half: fixed frame of reference



# Comparison with experiment

Non-dimensional bubble diameter

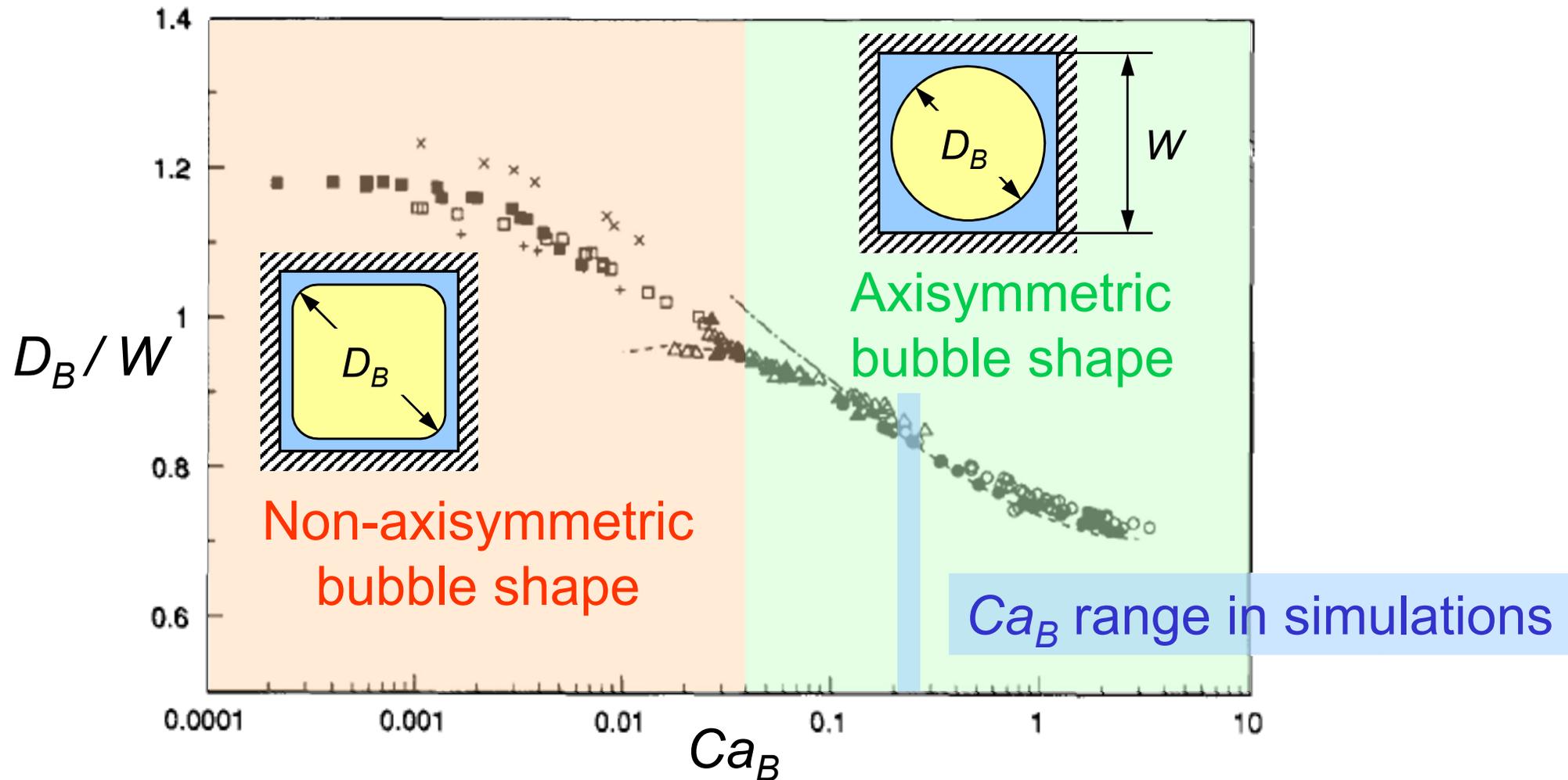
Relative velocity

Non-dimensional  $U_B$

| Case               | $L_{uc} / W$ | $Ca_B$   | $D_B / W$   | $(U_B - J_{total}) / U_B$ | $U_B / J_{total}$ |
|--------------------|--------------|--|-------------|---------------------------|-------------------|
| A                  | 1            | 0.204  | 0.81        | 1.80                      | 0.445             |
| B                  | 1.25         | 0.207  | 0.84        | 1.75                      | 0.430             |
| C                  | 1.5          | 0.215  | 0.85        | 1.75                      | 0.430             |
| D                  | 1.75         | 0.238  | 0.85        | 1.78                      | 0.438             |
| E                  | 2            | 0.253  | 0.85        | 1.8                       | 0.445             |
| Experimental data* |              | correlated in terms of capillary number $Ca_B \equiv \mu_l U_B / \sigma$ |             |                           |                   |
|                    |              | 0.2 – 0.25   | 0.82 – 0.86 | 1.68 – 1.84               | 0.435 – 0.475     |

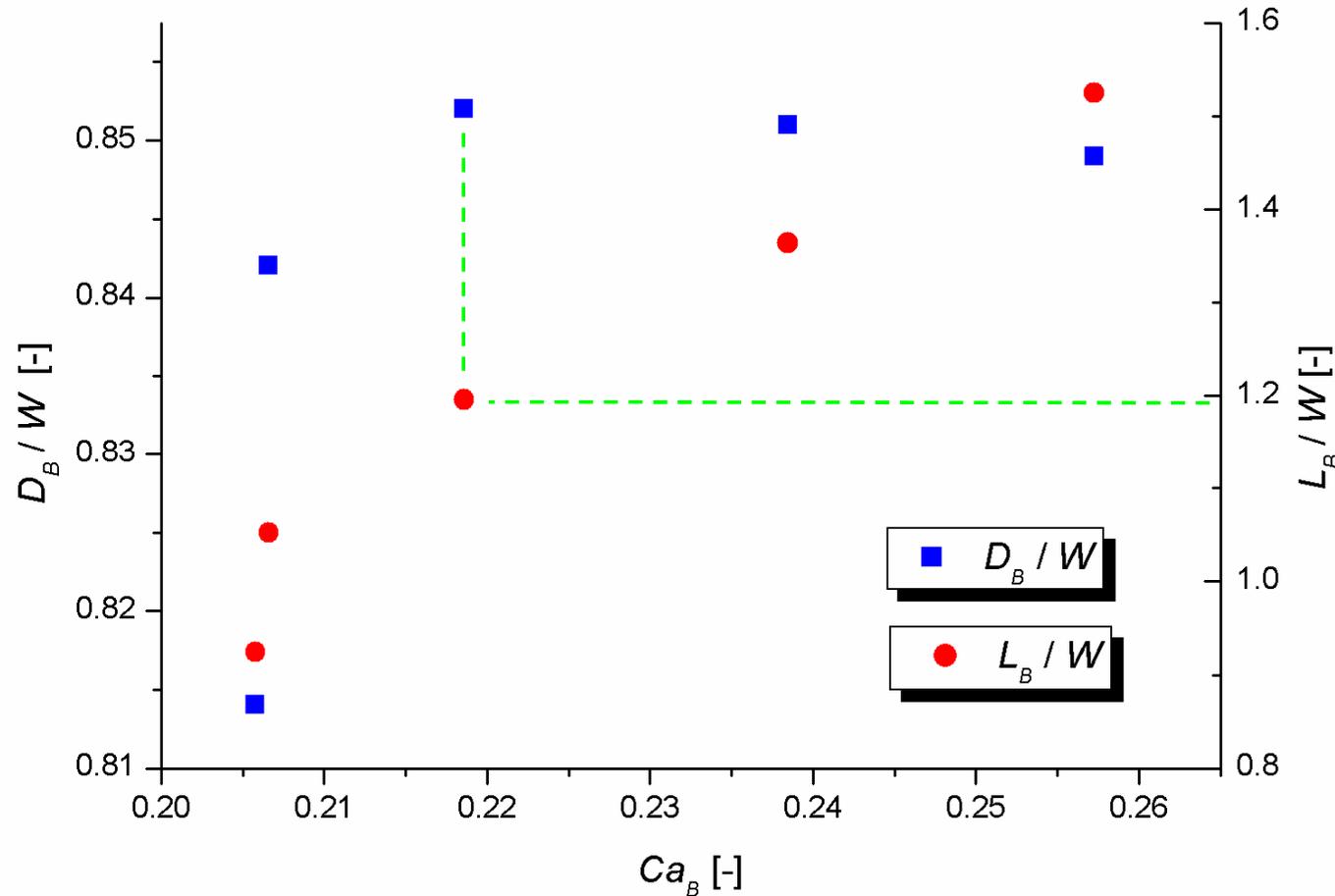


# $D_B$ measured along channel diagonal\*



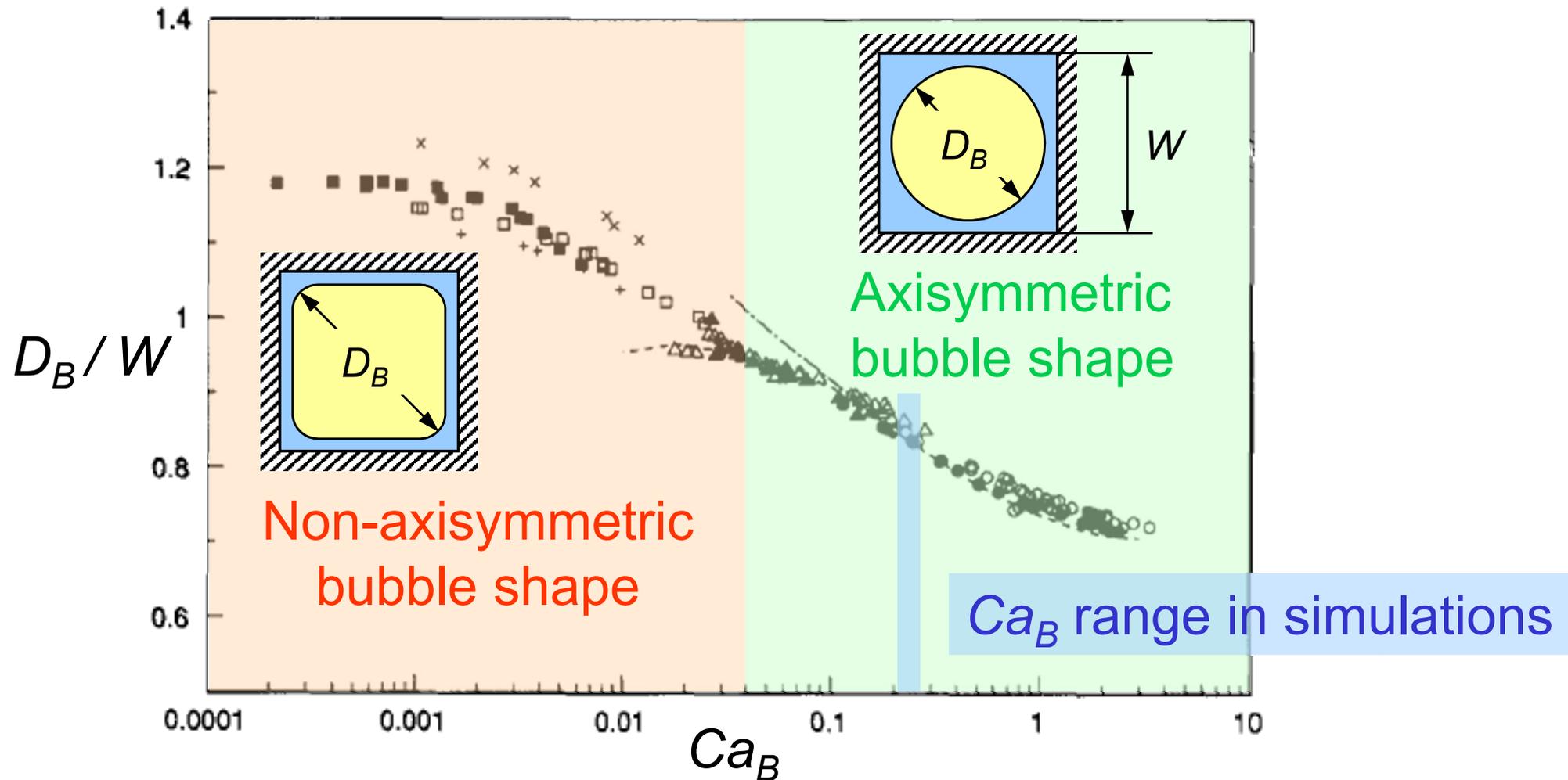
$\Rightarrow D_B/W$  decreases with increase of  $Ca_B$

# Bubble diameter in simulations



$D_B / W$  decreases with increase of  $Ca_B$  only if the bubble length  $L_B$  is larger than about 1.2 the channel width (this is the case in the experiments by Thulasidas et al.)

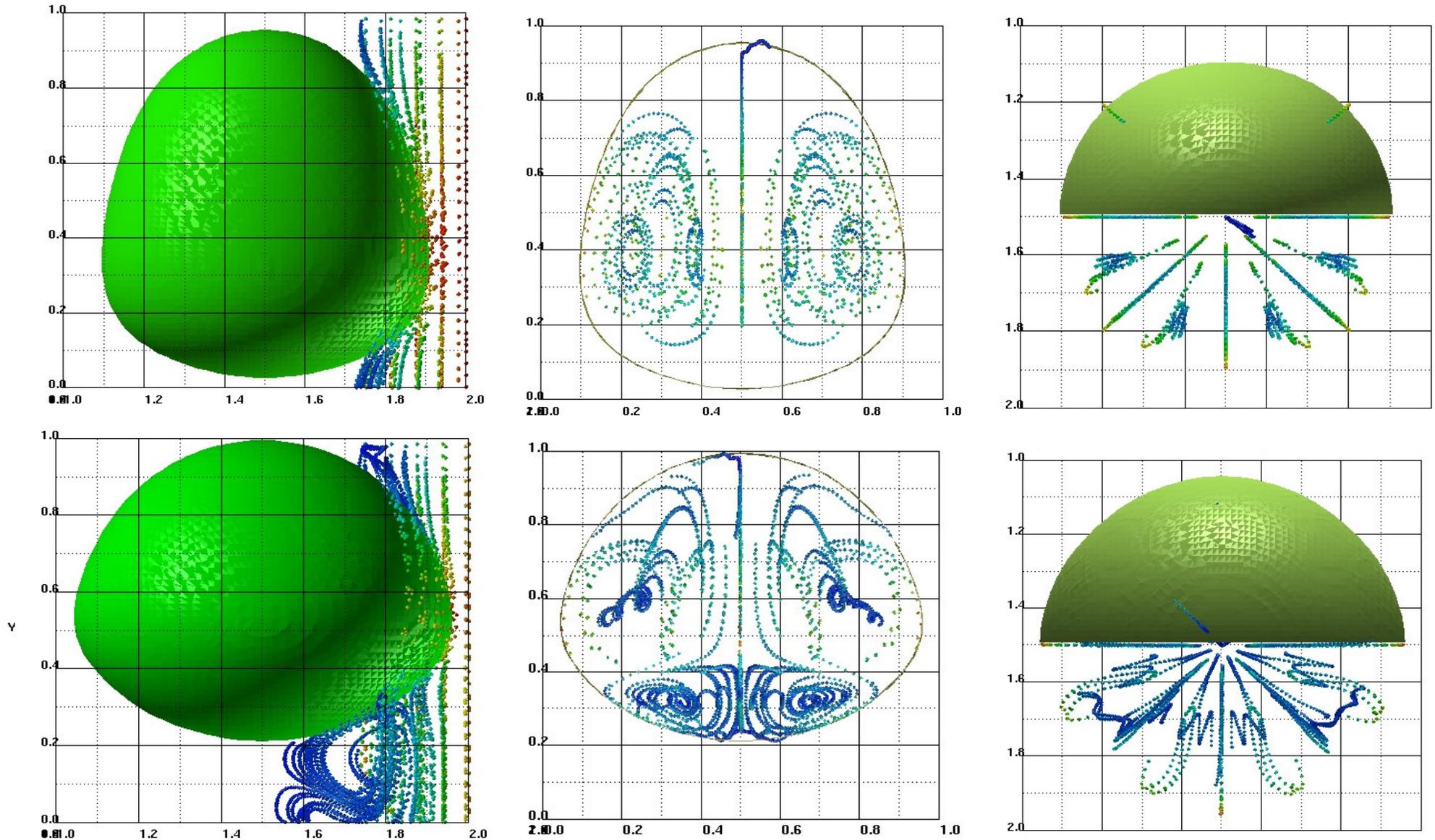
# $D_B$ measured along channel diagonal\*



$\Rightarrow D_B/W$  decreases with increase of  $Ca_B$

# Influence of capillary number

top row:  $Ca = 0.205$ , bottom row:  $Ca = 0.043$



# Turbulence modeling for bubbly flow

- No generally accepted model available in literature
- Analytical turbulence kinetic energy eq. for liquid phase\*:

$$\begin{aligned}
 \frac{\partial}{\partial t}(\alpha_L k_L) + \nabla \cdot (\alpha_L k_L \overline{\mathbf{u}}_L) &= \underbrace{\frac{1}{Re_{ref}} \nabla \cdot (\alpha_L \overline{\mathbb{T}}'_L \cdot \overline{\mathbf{u}}'_L)}_{\text{DIFFUSION}} - \nabla \cdot \left[ \alpha_L \left( \overline{p'_L \mathbf{u}'_L} + \frac{1}{2} \overline{(\mathbf{u}'_L \cdot \mathbf{u}'_L) \mathbf{u}'_L} \right) \right] \\
 &\quad \underbrace{-\alpha_L \overline{\mathbf{u}'_L \mathbf{u}'_L} : \nabla \overline{\mathbf{u}}_L}_{\text{PRODUCTION}} \quad \underbrace{-\frac{1}{Re_{ref}} \alpha_L \overline{\mathbb{T}}'_L : \nabla \overline{\mathbf{u}}_L}_{\text{DISSIPATION}} + \underbrace{\left[ \frac{1}{Re_{ref}} \overline{\mathbb{T}}'_{L,in} - p'_{L,in} \mathbb{I} \right] \cdot \overline{\mathbf{u}}'_{L,in} \cdot \mathbf{n}_{L,in} a_{in}}_{\text{INTERFACIAL TERM}}
 \end{aligned}$$

$$k_L \equiv \frac{1}{2} \overline{\mathbf{u}'_L \cdot \mathbf{u}'_L} = \frac{1}{2} \frac{1}{U_{ref}^{*2}} \overline{\mathbf{u}'_L \cdot \mathbf{u}'_L}$$

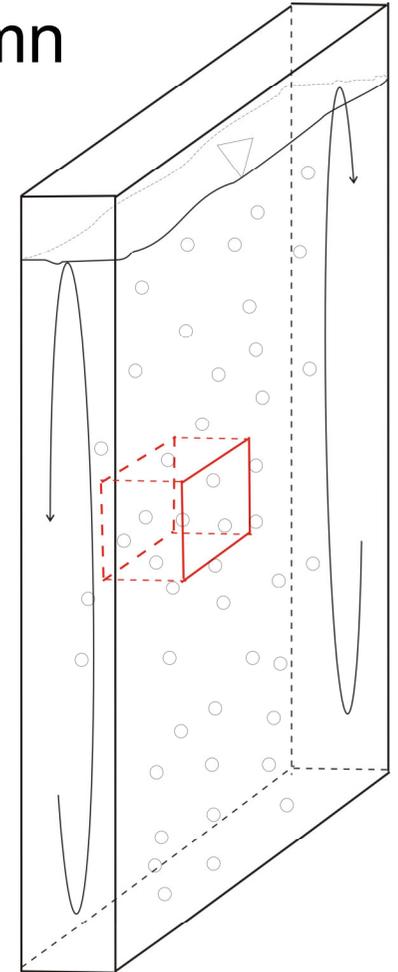
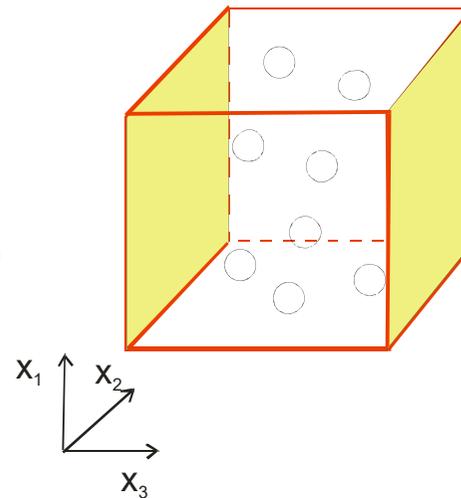
$$\overline{\mathbb{T}}'_L = \mu_L \left[ \nabla \overline{\mathbf{u}}_L + (\nabla \overline{\mathbf{u}}_L)^T \right]$$

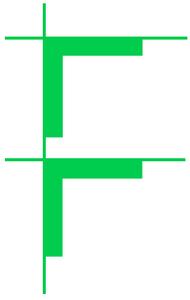
$$\overline{A}_L = \text{averaging} \quad \overline{A}_L = \overline{A_L X_L} / \overline{X_L} \quad \alpha_L = \overline{X_L}$$

$$A'_L = A_L - \overline{A}_L \quad A'_{Lin} = A_{Lin} - \overline{A}_L$$

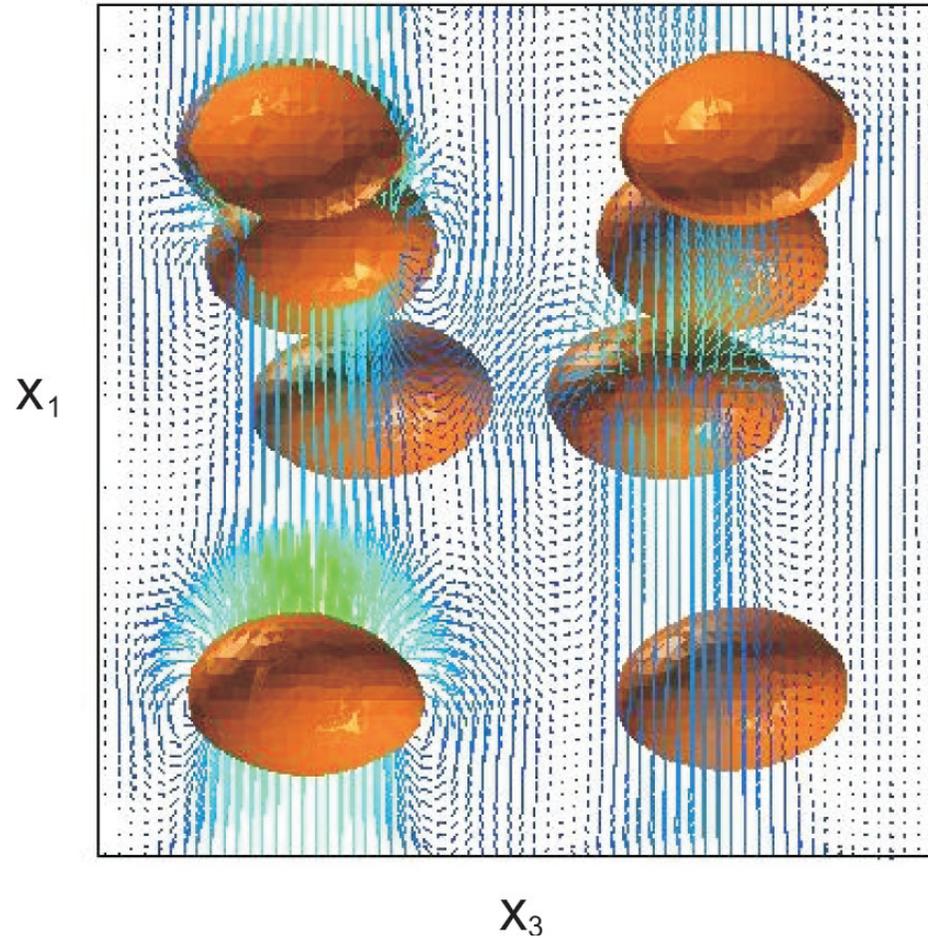
# Simulation of bubble swarm flow

- Simulation mimics section of a flat bubble column
  - periodic b.c. in vertical and span-wise directions
  - rigid lateral walls
- Domain:  $1 \times 1 \times 1$ , Grid:  $64 \times 64 \times 64$
- Eight bubbles with  $d_B/W = 0.25$  ( $\varepsilon = 6.5\%$ )
- Phase density ratio: 0.5
- Phase viscosity ratio: 1
- Bubble Eötvös number: 3.065
- Morton number:  $3.06 \cdot 10^{-6}$

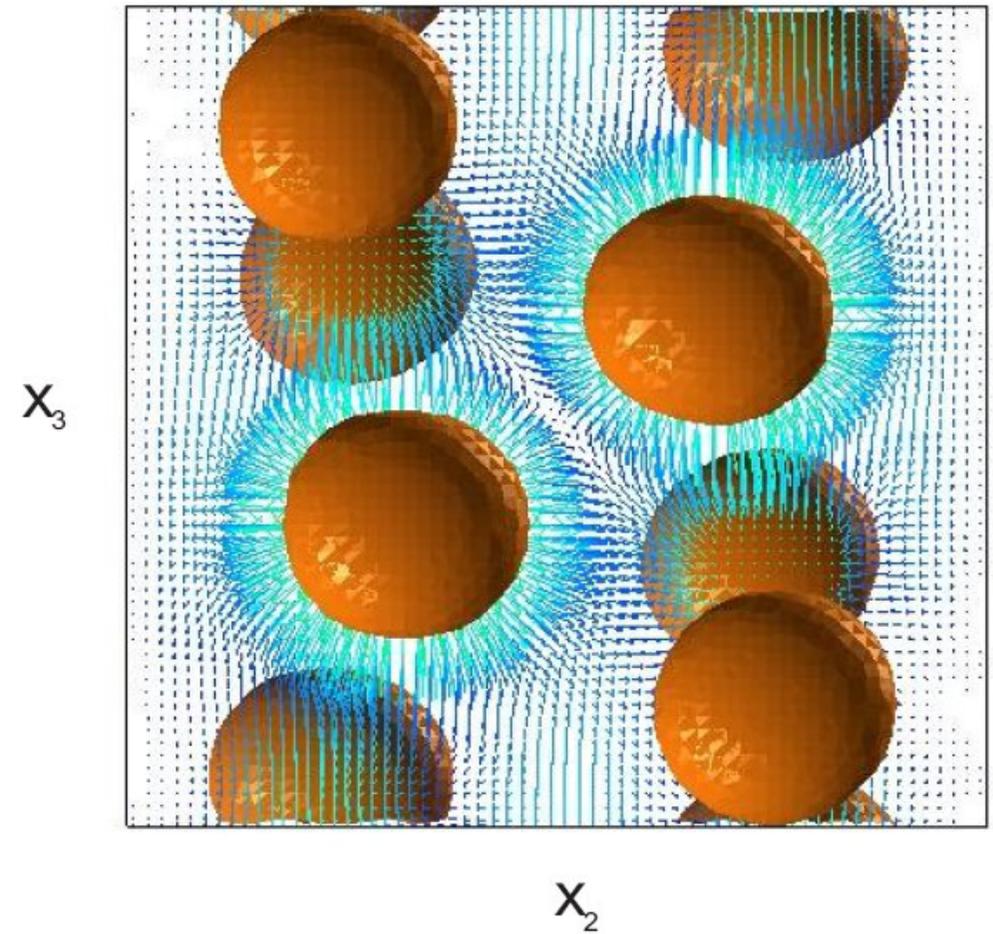




# Flow visualization



view from side



view from top

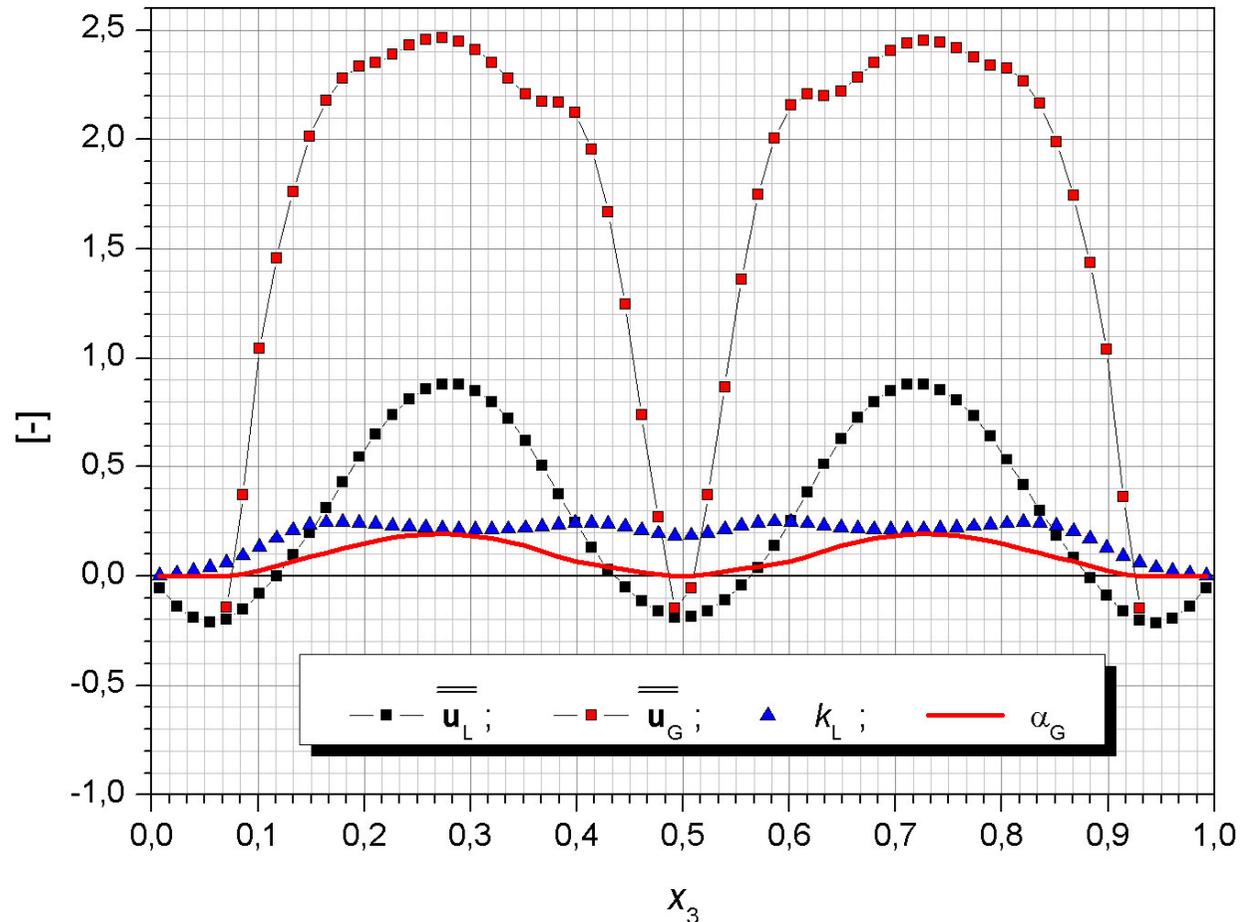
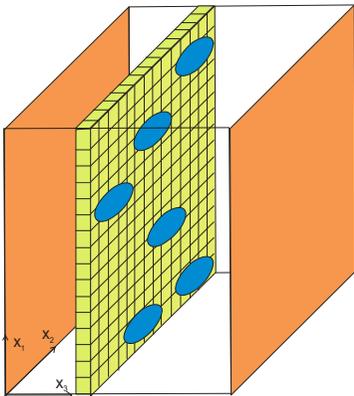
# Wall-normal profiles of mean quantities

Plane

averaging:

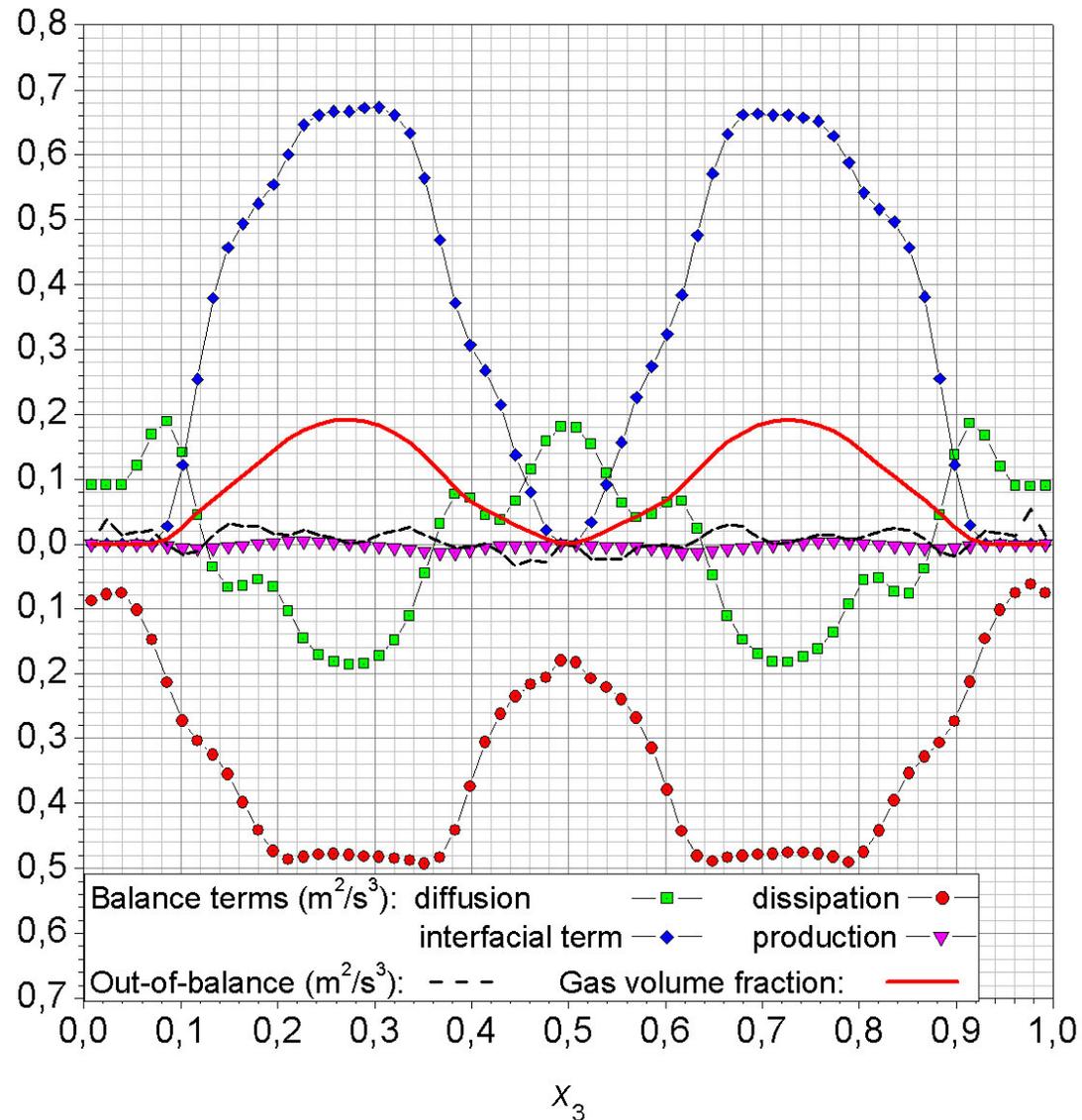
$$\overline{A_L(x_3)} = \frac{1}{N_1 N_2} \sum_{i=1}^{N_1} \sum_{j=1}^{N_2} f_{i,j,k} A_{i,j,k}$$

$$\overline{\overline{A_L(x_3)}} = \frac{\sum_{i=1}^{N_1} \sum_{j=1}^{N_2} f_{i,j,k} A_{i,j,k}}{\sum_{i=1}^{N_1} \sum_{j=1}^{N_2} f_{i,j,k}}$$





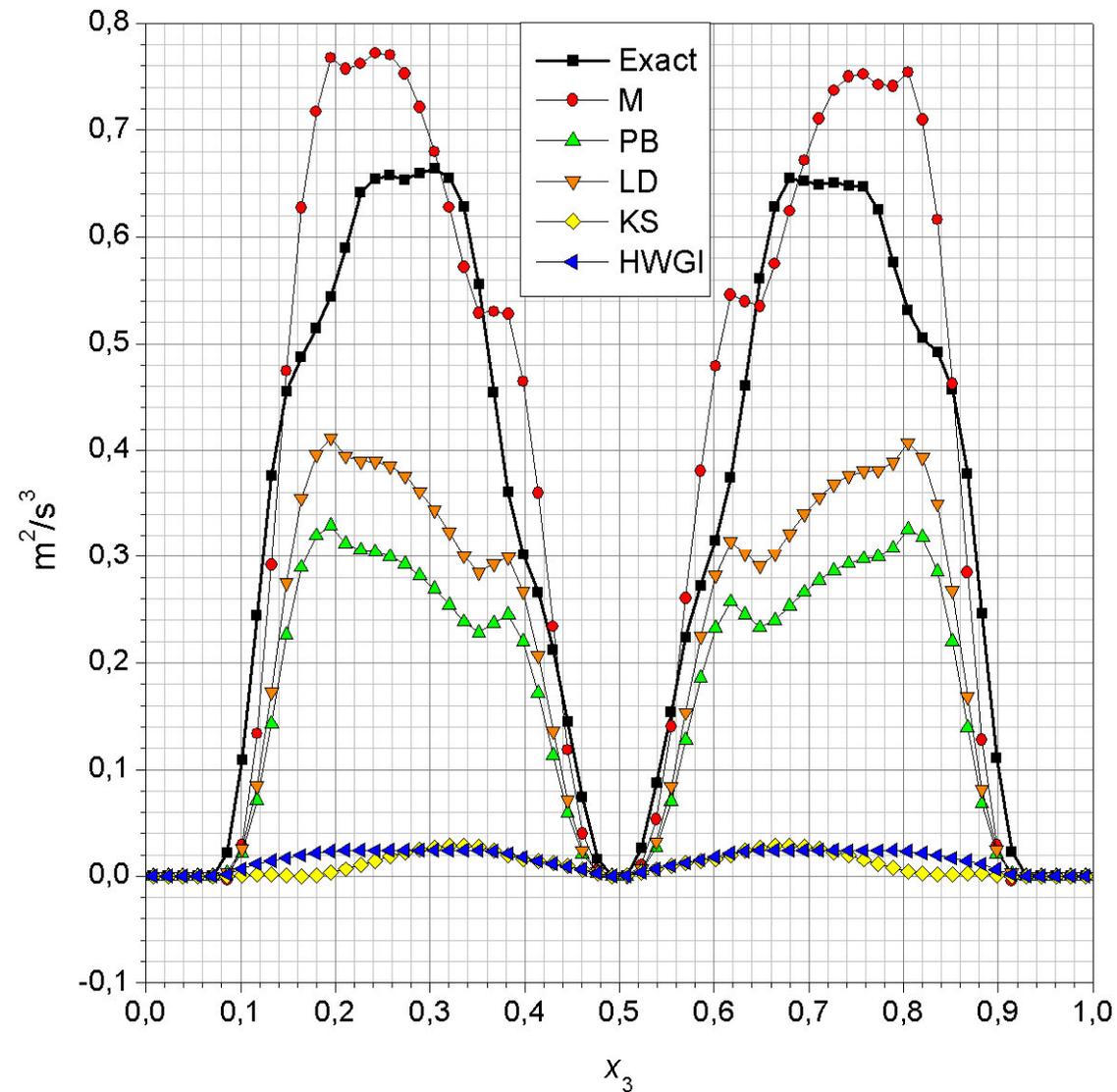
# Budget of $k_L$ -equation



# Models for interfacial term

| Reference                                  | Work of drag force, $W_D^*$  | Other contributions, $W_{ND}^*$   |
|--|--|---|
| Kataoka & Serizawa (1997)<br>Model 1, KS   | $0.075 f_w \left[ \frac{3}{4} \alpha_G \frac{C_D}{d_B^*} U_T^{*3} \right]$   | $-\alpha_G \frac{k_L^{*3/2}}{d_B^*}$  |
| Hill <i>et al.</i> (1995)<br>Model 2, HWGI | $\frac{3}{4} \frac{\alpha_G C_D}{d_B^*} \left  \overline{\mathbf{u}_R^*} \right  \left\{ \frac{\mu_L^* \overline{\mathbf{u}_R^*} \cdot \nabla^* \alpha_G}{0.3 \rho_L^* \alpha_L \alpha_G} + 2k_L^* (C_t - 1) \right\}$ | None  |
| Lahey & Drew (2000)<br>Model 3, LD         | $\frac{1}{4} \alpha_L (1 + C_D^{4/3}) \alpha_G \frac{\left  \overline{\mathbf{u}_R^*} \right ^3}{d_B^*}$   | None  |
| Morel (1997)<br>Model 4, M                 | $\frac{3}{4} \alpha_G \frac{C_D}{d_B^*} \left  \overline{\mathbf{u}_R^*} \right ^3$  | $\frac{1 + 2\alpha_G}{2\alpha_L} \alpha_G \left\{ \frac{D_G \overline{\mathbf{u}_G^*}}{Dt^*} - \frac{D_L \overline{\mathbf{u}_L^*}}{Dt^*} \right\} \cdot \overline{\mathbf{u}_R^*}$ |
| Pfleger & Becker (2001)<br>Model 5, PB     | $1.44 \alpha_L \left[ \frac{3}{4} \alpha_G \frac{C_D}{d_B^*} \left  \overline{\mathbf{u}_R^*} \right ^3 \right]$   | None  |

# Performance of models for interfacial term





# Conclusions

- Numerical simulation of bubble train flow
  - Square vertical mini-channel ( $2 \text{ mm} \times 2 \text{ mm}$ )
  - Good agreement with experimental data from literature
  - Investigation on influence of length of flow unit cell
  - Strong influence of capillary number
- Liquid turbulence kinetic energy equation for bubbly flow
  - production mainly by interfacial terms
  - no local equilibrium between interfacial production and dissipation  
⇒ significant redistribution by diffusion
  - interfacial terms can well be modeled by work of drag forces