

Analysis of bubble-induced turbulence and needs for model improvements

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Transmutation Systems – THIRS
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- Introduction
- Phenomena in bubble-turbulence interaction
- Direct numerical simulations of bubble swarm flows
- Analysis of transport equation of liquid phase turbulent kinetic energy (k_L) from DNS data
 - Budget of terms in k_L -equation
 - Assessment of closure assumptions
- Conclusions

- For thermal-hydraulic design of innovative reactor and transmutation systems, computational fluid dynamics (CFD) is of great importance
 - Experiments at full scale and for realistic conditions (temperature, pressure, flow rates) are often hardly possible
 - The scale-up from laboratory experiments and the design and optimization of the reactor relies almost entirely on CFD
 - Due to large dimensions, the flow is usually in the turbulent regime, thus for reliable CFD results turbulence models play a critical role
 - Assessment and needs for turbulence models
 - Single phase heat transfer, see presentation A05 by G. Grötzbach (Monday)
 - Here: two-phase bubbly flows

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Bubble-turbulence interaction

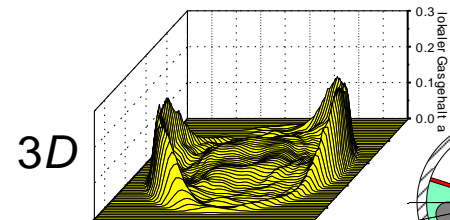
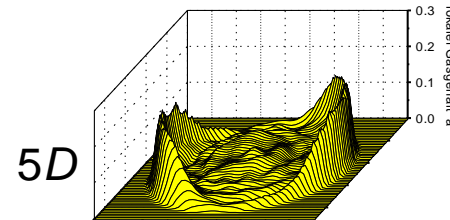
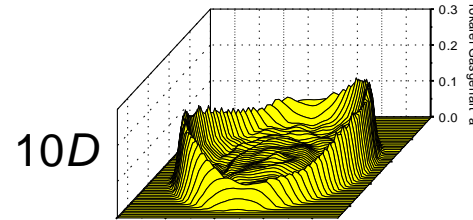
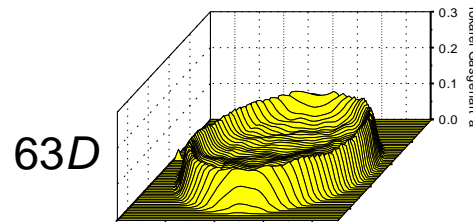
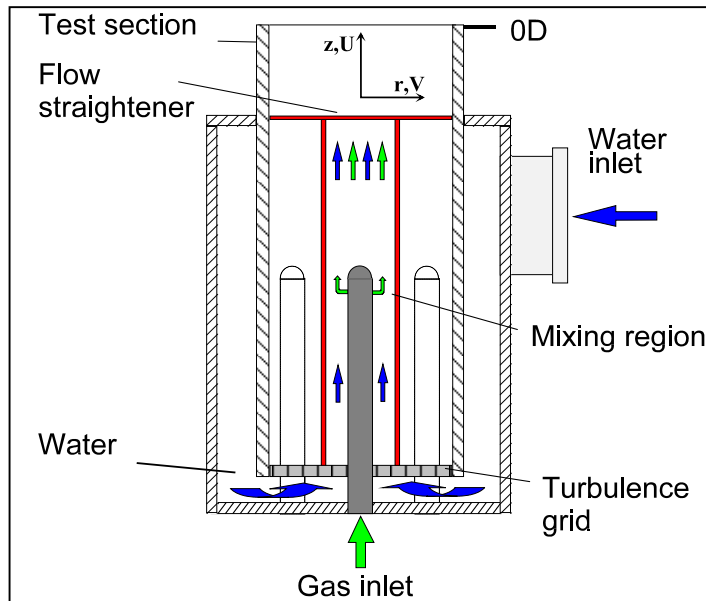
Experiments on cocurrent upward air-water flow in a vertical pipe
(Samstag, FZKA 5662, 1996)

$D=70\text{mm}$

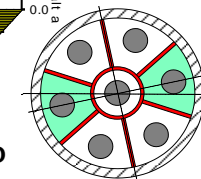
$j_l=0.36, 1.08, 1.44 \text{ m/s}$

$\beta = j_l / (j_l + j_g) = 0, 5, 10\%$

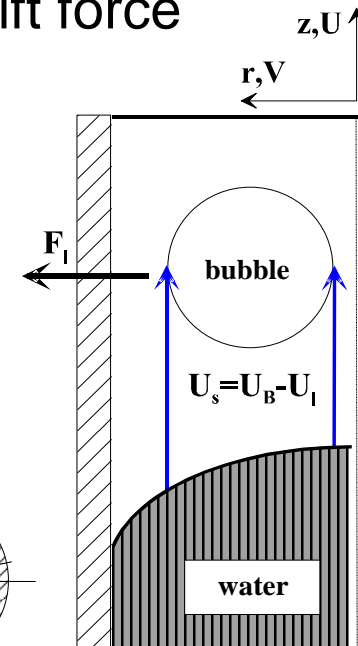
1 – 7 nozzles , $d_B \approx 2 - 5 \text{ mm}$



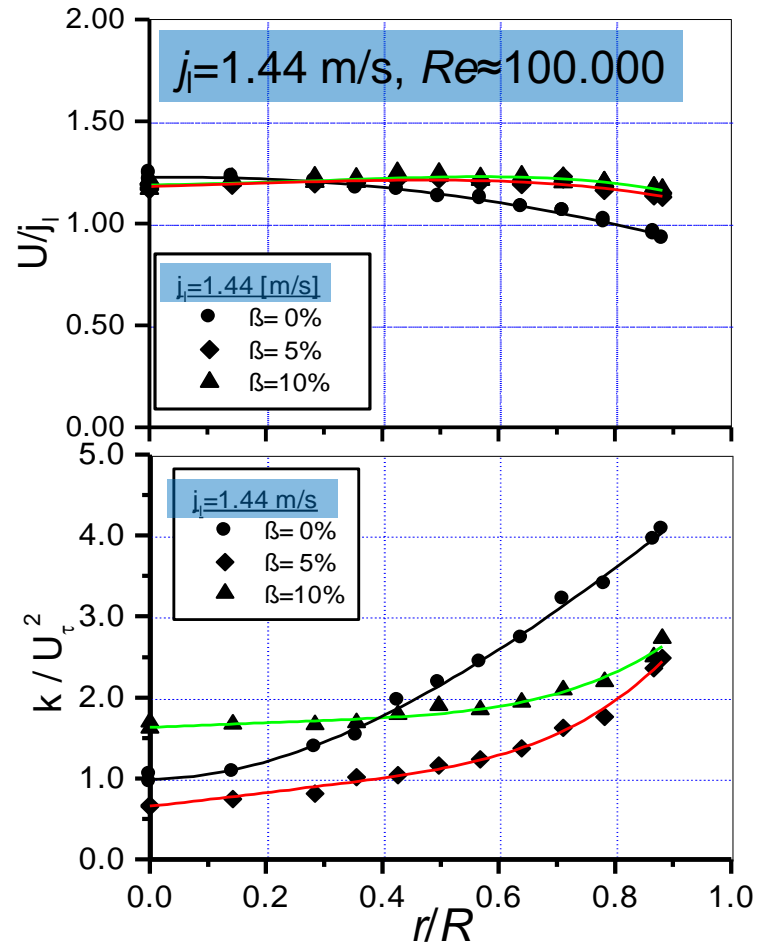
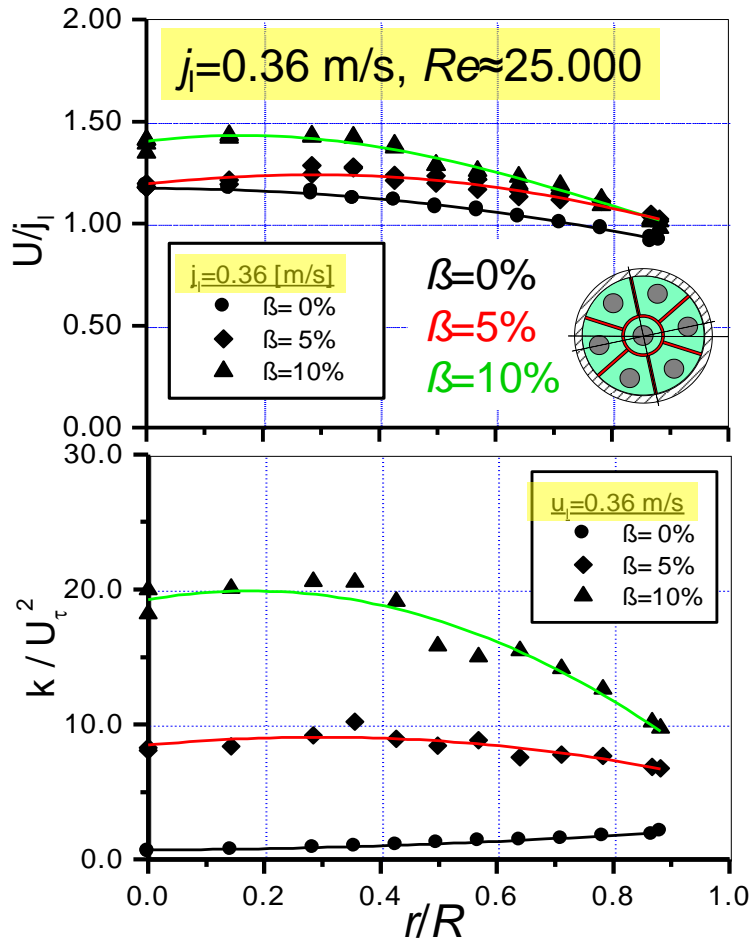
$j_l=1.08 \text{ m/s}, \beta=5\%$



- Fast azimuthal gas redistribution (anisotropic)
- “Wall peaking” due to lift force



Bubble-turbulence interaction



Radial profile of mean axial liquid velocity at $70D$

Radial profile of liquid turbulent kinetic energy at $70D$

Bubbles enhance turbulence for $0 \leq r/R < 1$ for both, $\beta = 5\%$ and $\beta = 10\%$

$\beta = 5\%$: bubbles damp turbulence for $0 \leq r/R < 1$
 $\beta = 10\%$: bubbles enhance turbulence for $r/R < 0.4$ and damp it for $r/R > 0.4$

- Mechanisms of bubble-turbulence interaction
 - Direct
 - Gas bubbles displace liquid and induce velocity fluctuations
 - Vortices in bubble wake induce velocity fluctuations
 - Dissipation of liquid phase turbulence kinetic energy by disperse elements
 - Indirect
 - Modification of mean liquid velocity profile by presence of bubbles
 - Modification of production rate of turbulent kinetic energy by shear stresses
 - Nonlinear feedback
 - Turbulence has strong influence on breakup and coalescence of bubbles and thus determines bubble size distribution
 - Bubbles of different size have different rise velocity and experience different magnitude and direction of lift force (toward wall or toward pipe center)
 - Bubble size distribution influences radial void fraction profile
 - Radial void fraction profile influences mean liquid velocity profile

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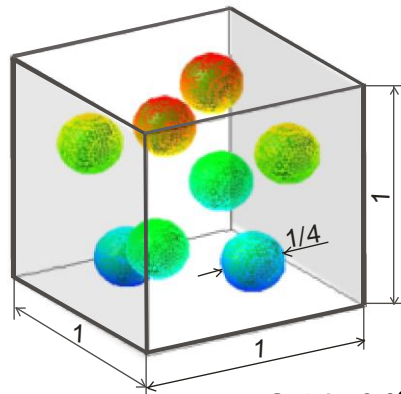
- Background and motivation for DNS
 - Experiments show that bubbles may enhance turbulence or damp turbulence as compared to single phase flow
 - Experimental data in literature are not conclusive and contradictory*
 - Qualitatively, phenomena are partly understood, but not quantitatively
 - A reliable model to account for turbulence in bubbly flows in Euler-Euler CFD codes (two-fluid model) is missing
 - Model development is hindered by difficulty to measure relevant correlations between various fluctuating quantities
- Goal: use DNS data to analyze turbulence kinetic energy equation for liquid phase and to test closure assumptions

* For recent literature overview see Hu et al, CES 62 (2007) 1199

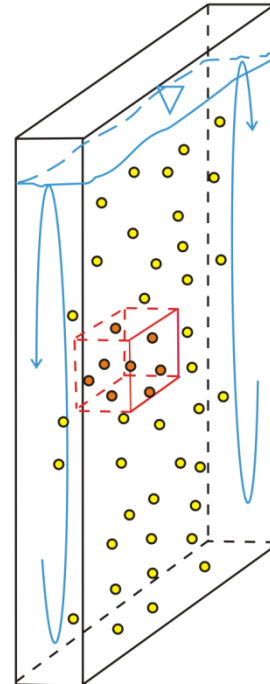
- Volume-of fluid method for interface tracking
 - Interface is locally approximated by a plane (PLIC method)
- Governing equations for two incompressible fluids
 - Single field momentum equation with surface tension term
 - Zero divergence condition for center-of-mass velocity
 - Advection equation for liquid volumetric fraction f
- Discretization in space and solution strategy
 - Projection method with 3rd order explicit Runge-Kutta time integration
 - Finite volume formulation for regular staggered grid
 - Second order central difference approximations
- Verification
 - Test problems with known analytical solution
 - Experimental results for single bubbles of various shape

Bubble swarm simulations

- Simulations for central region of a flat bubble column (Ilić, FZKA 7199, 2006)
 - No-slip side walls and periodic b.c. in vertical and lateral direction
 - Bubbles drive liquid flow and induces “pseudo-turbulence”



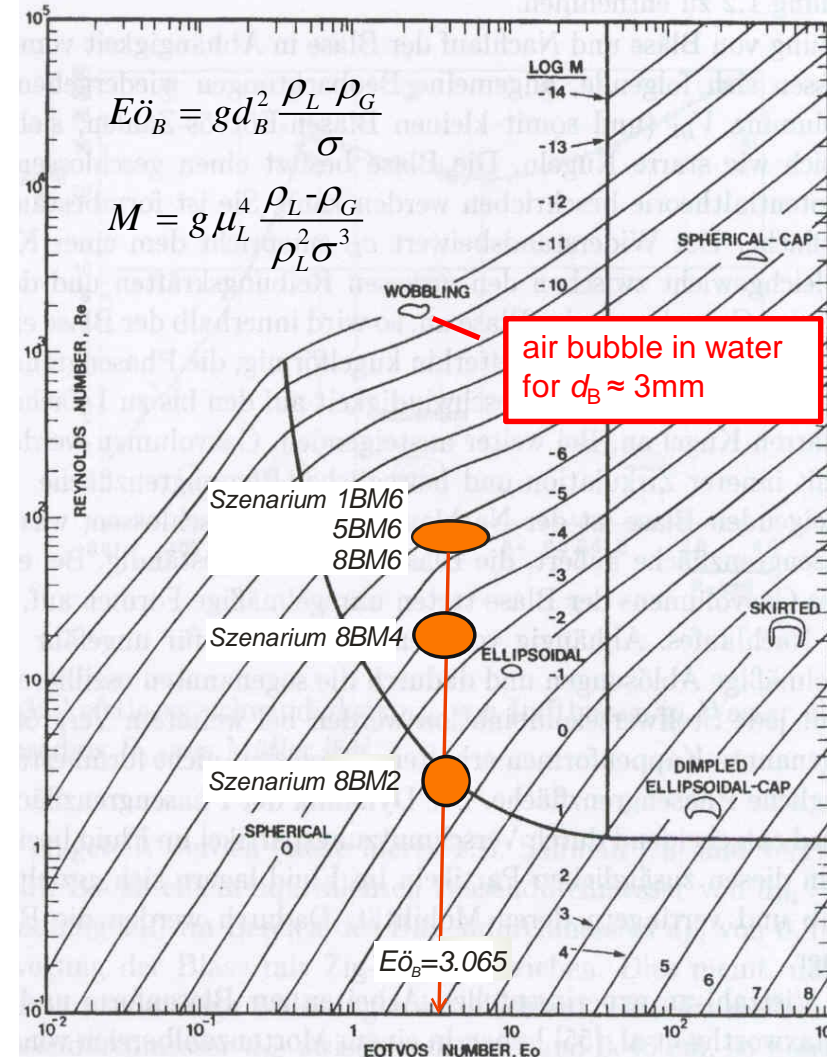
Grid: 64^3 and 100^3



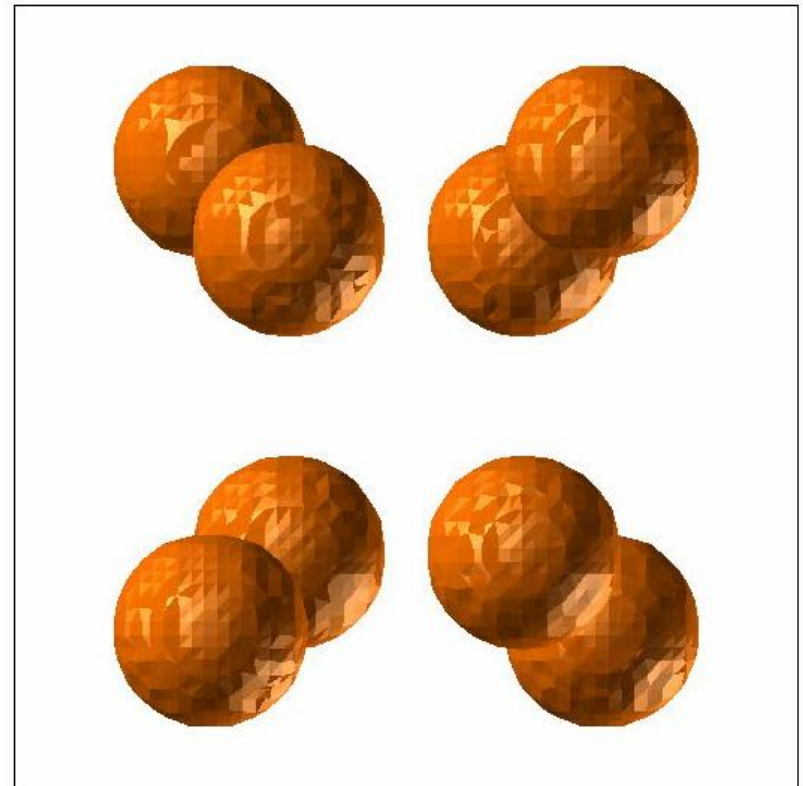
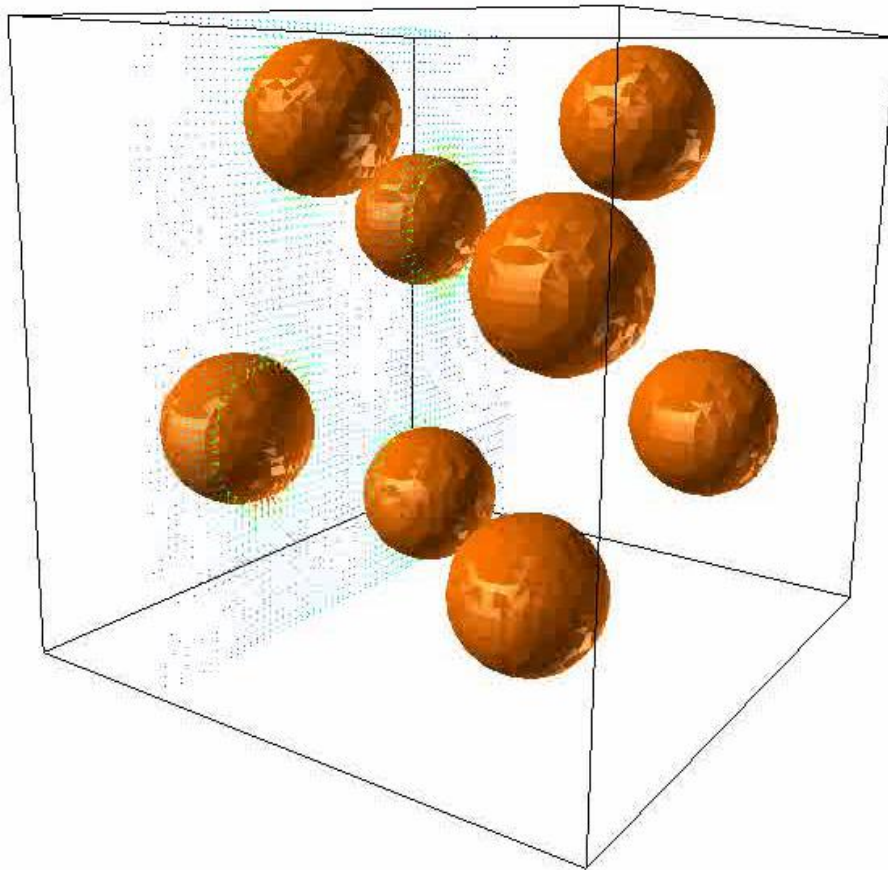
Influence of gas content

Scenario	1BM6	5BM6	8BM6	8BM4	8BM2
No. of bubbles	1	5	8	8	8
Gas content	0.818%	4.088%	6.544%	6.544%	6.544%
Morton number	$3.06 \cdot 10^{-6}$	$3.06 \cdot 10^{-6}$	$3.06 \cdot 10^{-6}$	$3.06 \cdot 10^{-4}$	$3.06 \cdot 10^{-2}$

$\rho_L/\rho_G = 2, \mu_L/\mu_G = 1$ Influence of bubble shape and velocity

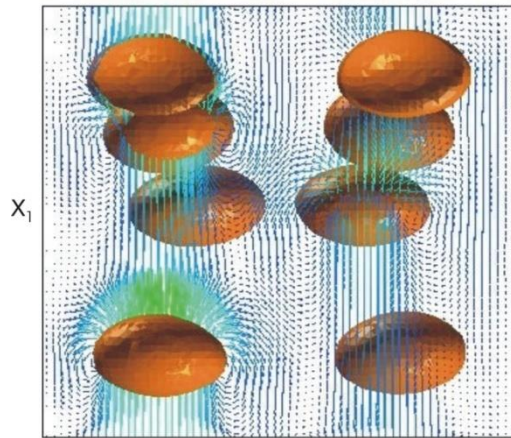


Visualization of bubble motion

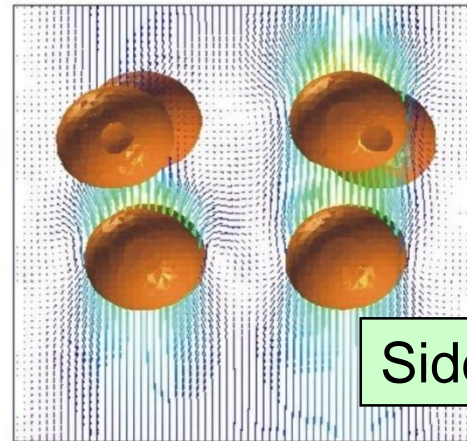


8BM6: $M = 3 \times 10^{-6}$

Bubble shape and path

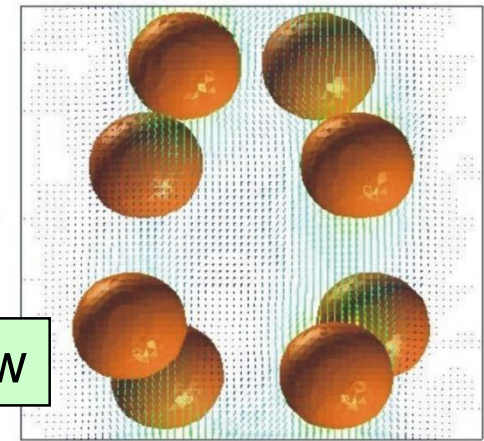


x_3

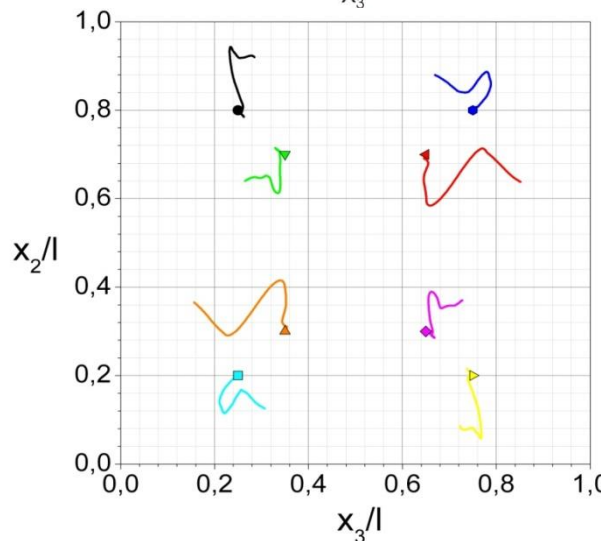


x_3

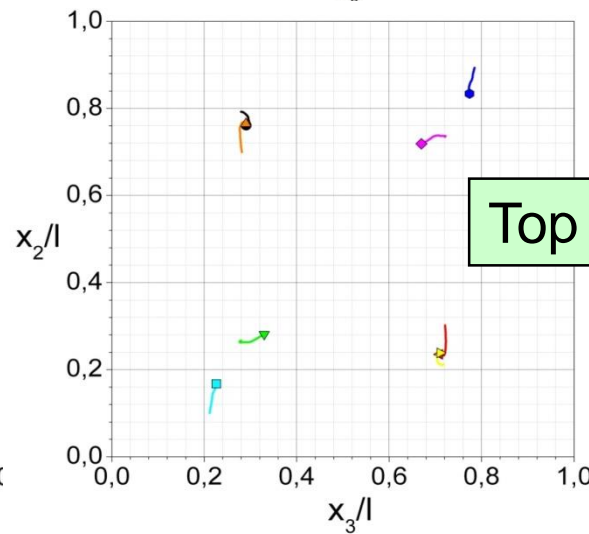
Side view



x_3

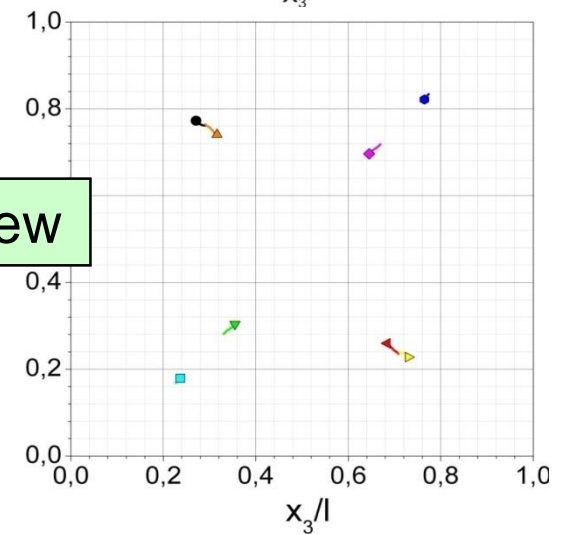


8BM6: $M = 3 \times 10^{-6}$



Top view

8BM4: $M = 3 \times 10^{-4}$

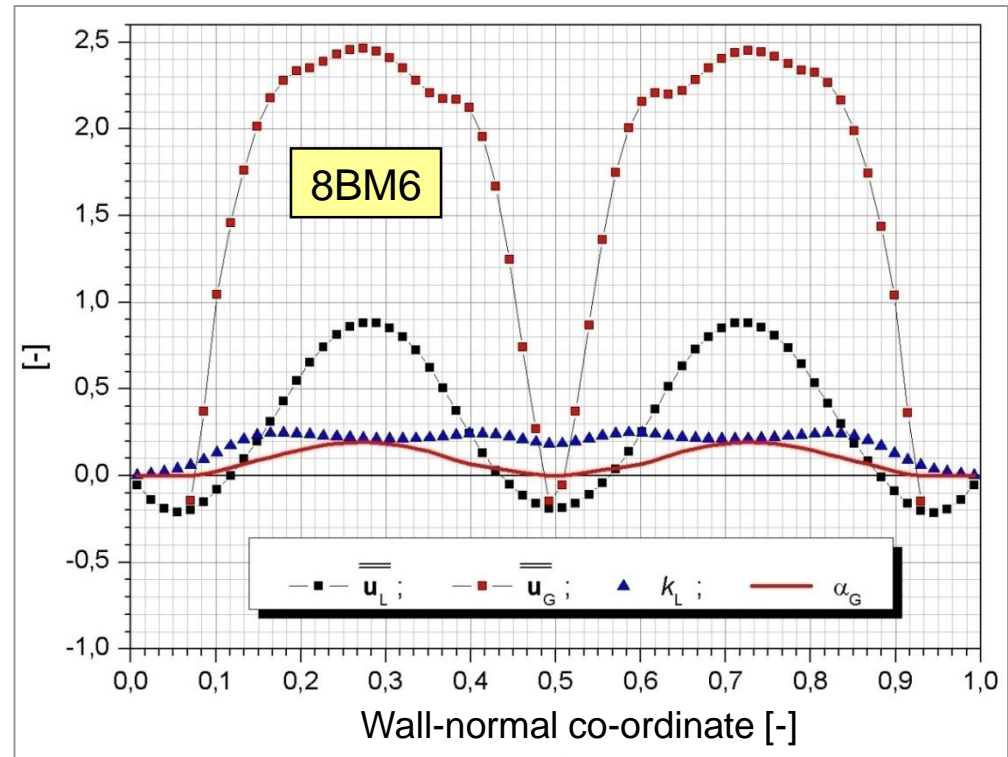
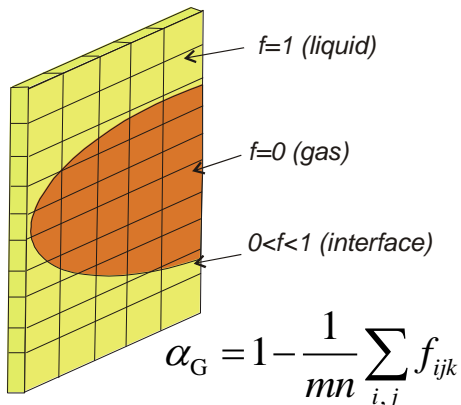
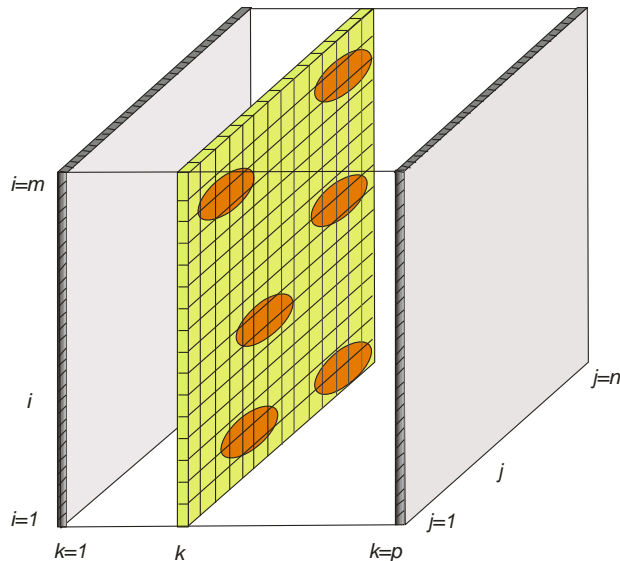


8BM2: $M = 3 \times 10^{-2}$

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Averaging of simulation results

Averaging over planes parallel to the side walls



Mean quantities:

$$\overline{A}_{Lk} = \frac{1}{mn} \sum_{i,j} A_{Lijk} \quad \text{average}$$

$$\overline{A}_{Lk} = \frac{1}{\alpha_{Lk}} \sum_{i,j} f_{ijk} A_{Lijk} \quad \text{phase-average}$$

Fluctuating quantities:

$$A'_{Lijk} = A_{Lijk} - \overline{A}_{Lk} \quad \text{in bulk}$$

$$A'_{Lijk,in} = A_{Lijk,in} - \overline{A}_{Lk} \quad \text{at interface}$$

Terms in exact k_L -equation and budget

$$\frac{\partial}{\partial t} (\alpha_L k_L) + \nabla \cdot (\alpha_L k_L \overline{\mathbf{u}}_L) =$$

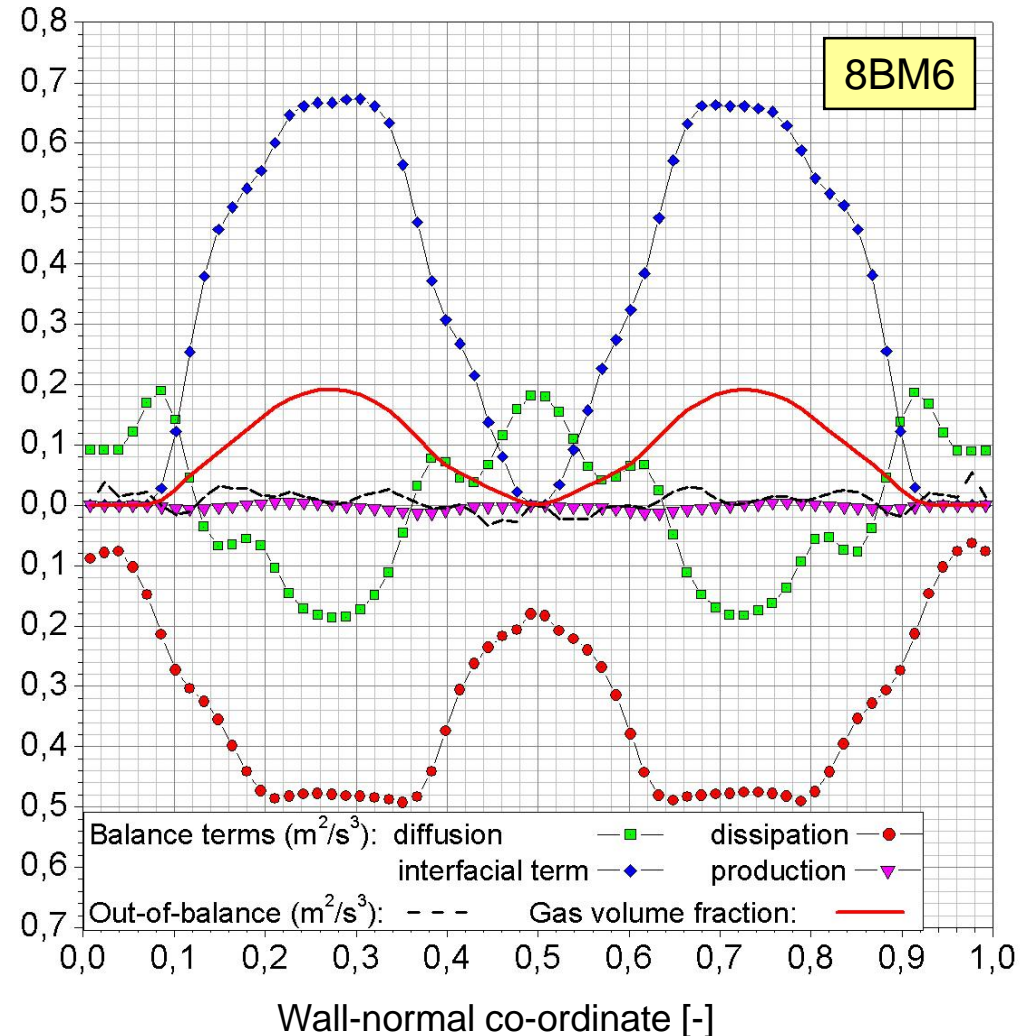
$$\underbrace{\frac{1}{Re_{ref}} \nabla \cdot (\alpha_L \overline{\mathbb{T}}'_L \cdot \overline{\mathbf{u}}'_L) - \nabla \cdot \left[\alpha_L \left(\overline{p}'_L \overline{\mathbf{u}}'_L + \frac{1}{2} \overline{(\mathbf{u}'_L \cdot \mathbf{u}'_L)} \overline{\mathbf{u}}'_L \right) \right]}_{\text{DIFFUSION}}$$

$$\underbrace{-\alpha_L \overline{\mathbf{u}}'_L \overline{\mathbf{u}}'_L : \nabla \overline{\mathbf{u}}_L}_{\text{PRODUCTION}}$$

$$\underbrace{-\frac{1}{Re_{ref}} \alpha_L \overline{\mathbb{T}}'_L : \nabla \overline{\mathbf{u}}_L}_{\text{DISSIPATION}}$$

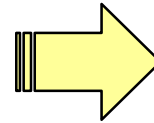
$$+ \underbrace{\left[\frac{1}{Re_{ref}} \overline{\mathbb{T}}'_{L,in} - \overline{p}'_{L,in} \mathbb{I} \right] \cdot \overline{\mathbf{u}}'_{L,in} \cdot \mathbf{n}_{L,in} a_{in}}_{\text{INTERFACIAL TERM}}$$

All terms on right hand side of k_L -equation must be modeled

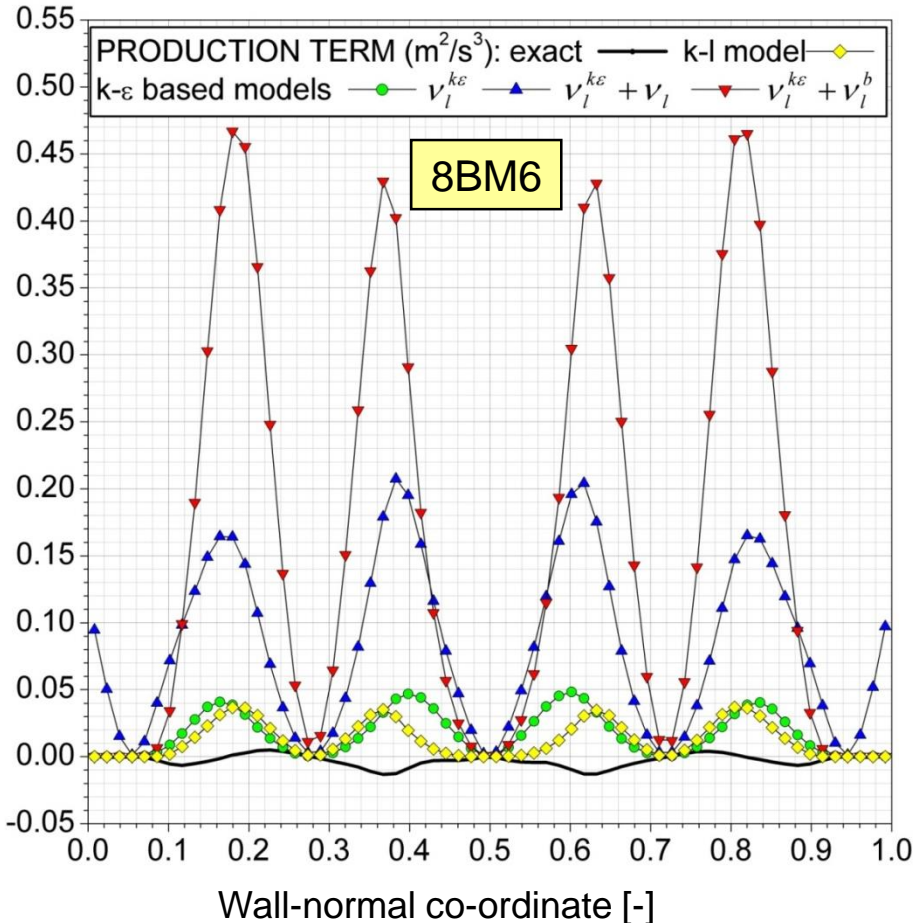


$$\underbrace{\alpha_L \overline{u'_L u'_L}}_{\text{EXACT}} : \nabla \overline{u'_L} \Rightarrow \underbrace{\alpha_L \nu_L^t (\nabla \overline{u'_L} + \nabla \overline{u'_L}^T)}_{\text{MODELLED}} : \nabla \overline{u'_L}$$

Boussinesq ansatz



Modeling of eddy viscosity:
extension of single phase concepts



k-l model:

$$\nu_L^t = 0.56 \underbrace{\alpha_G d_B / 3}_{\text{turbulence length scale}} \sqrt{k_L} \quad \sqrt{k_L} \quad \text{turbulence velocity scale}$$

α_G = gas content
 d_B = bubble diameter

Two-equation models:

$$\nu_L^t = \nu_L^{k\varepsilon} = 0.09 \frac{k_L^2}{|\varepsilon_L|}$$

$$\nu_L^t = \nu_L^{k\varepsilon} + \nu_L$$

$$\nu_L^t = \nu_L^{k\varepsilon} + 0.6 \alpha_G d_B \underbrace{|\overline{u_r}|}_{\nu_L^B}$$

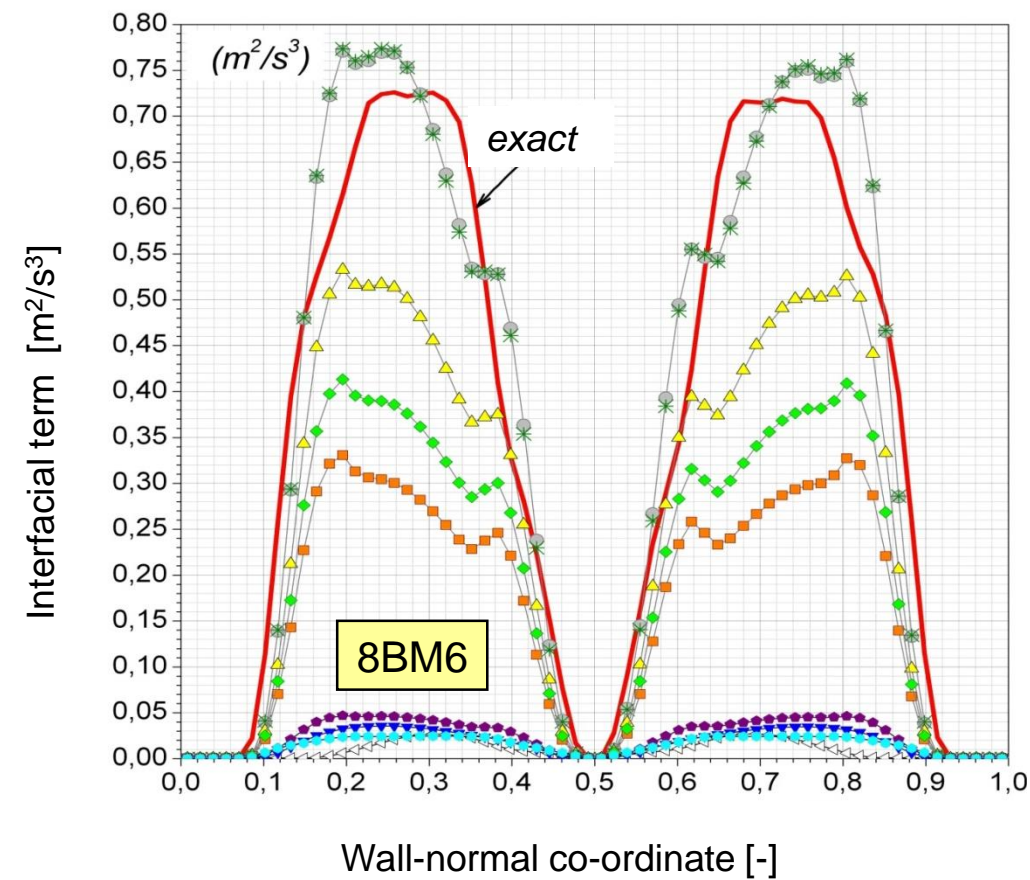
u_r = relative velocity
 ν_L = molecular viscosity of liquid phase
 ν_L^B = bubble-induced eddy viscosity (Sato)

All models overestimate production by shear stresses!

Models for interfacial term

exact:
$$-\rho_{L,in} \overline{u_{L,in} n_{L,i} a_{in}} + \tau_{Lij,in} \overline{u_{L,i} n_{L,j} a_{in}}$$

CLOSURE ASSUMPTIONS		
DRAG CONTRIBUTION Defined in the form of:	Other contributions	Model of:
Mean quantities:		
<u>As power of drag force</u>		
$W_D = 0.75 C_D \alpha_G \rho_L \overline{\mathbf{u}_T} ^3 / d_B$	$M_{vm} \overline{\mathbf{u}_T}$	Morel ☞
	none	Troshko&Hassan ⑥
<u>As part of power of drag force</u>		
0.05 $\alpha_G W_D$	none	Boisson et al. ③
0.75 W_D	none	Olmos et al. ☞
1.44 W_D	none	Pfleger et al. ☞
0.075 W_D	$\alpha_G \rho_L k_L^{2/3} / d_B$	Kataoka et al. ✖
<u>Drag force not explicitly included:</u>		
$0.25 \alpha_L \alpha_G \rho_L (1 + C_D^{4/3}) \overline{\mathbf{u}_T} ^3 / d_B$	none	Lahey et al. ☹
Mean and turbulent quantities:		
<u>Only liquid turbulence properties</u>		
$0.45 C_D \alpha_G \rho_L k_L \overline{\mathbf{u}_T} / d_B$	$2.53 \alpha_G \alpha_L \Pi$	Sheng et al. ☞
<u>Turbulence properties of both phases</u>		
$\frac{3}{4} C_D \frac{ \overline{\mathbf{u}_T} }{d_B} \left\{ 2 \alpha_G \rho_L (C_t - 1) k_L - \frac{\nu^{ke} \overline{\mathbf{u}_T} \nabla \alpha_G}{\alpha_L \alpha_G} \right\}$	none	Hill et al. ☞



– Modeling as power of drag force gives good results (non-drag forces are insignificant here)
– Which correlation to use for C_D ?

C_D = drag coefficient (different formulations used)

$$C_t = \overline{u'_G u'_G} / \overline{u'_L u'_L} \propto f(l_e, Re_t, \alpha_G, d_B, \overline{\mathbf{u}_T}, C_D)$$

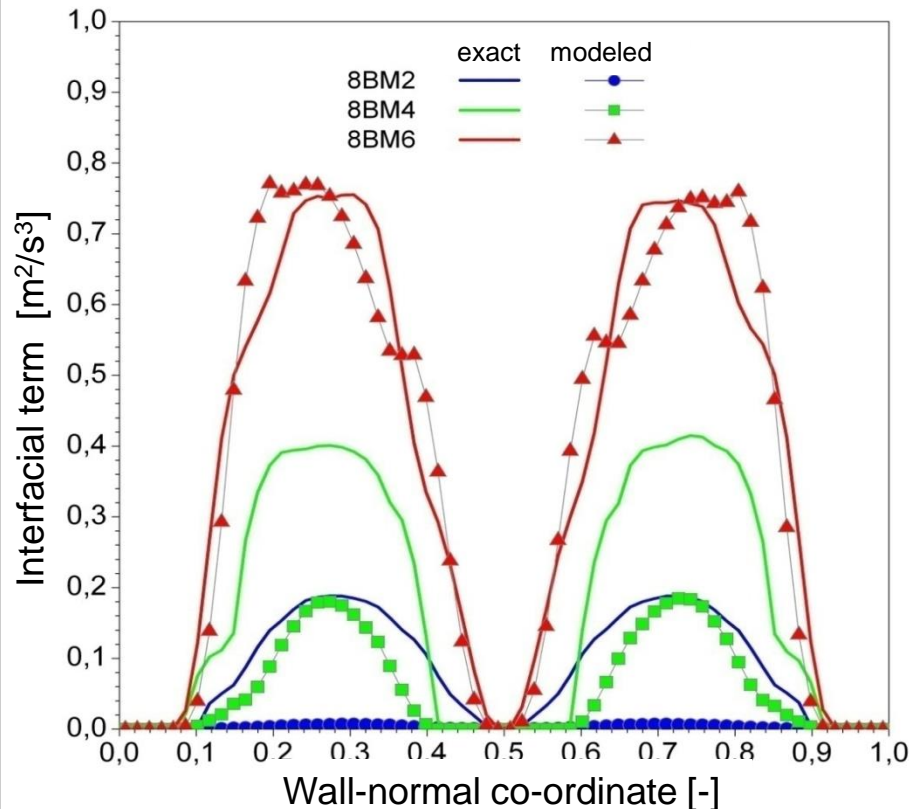
Models for interfacial term

Model of Morel (1997), Troshko&Hassan (2001):

$$W_D = 0.75(C_D / d_B) \alpha_G \rho_L \left| \overline{\mathbf{u}_r} \right| \overline{\mathbf{u}_r \cdot \mathbf{u}_r}$$

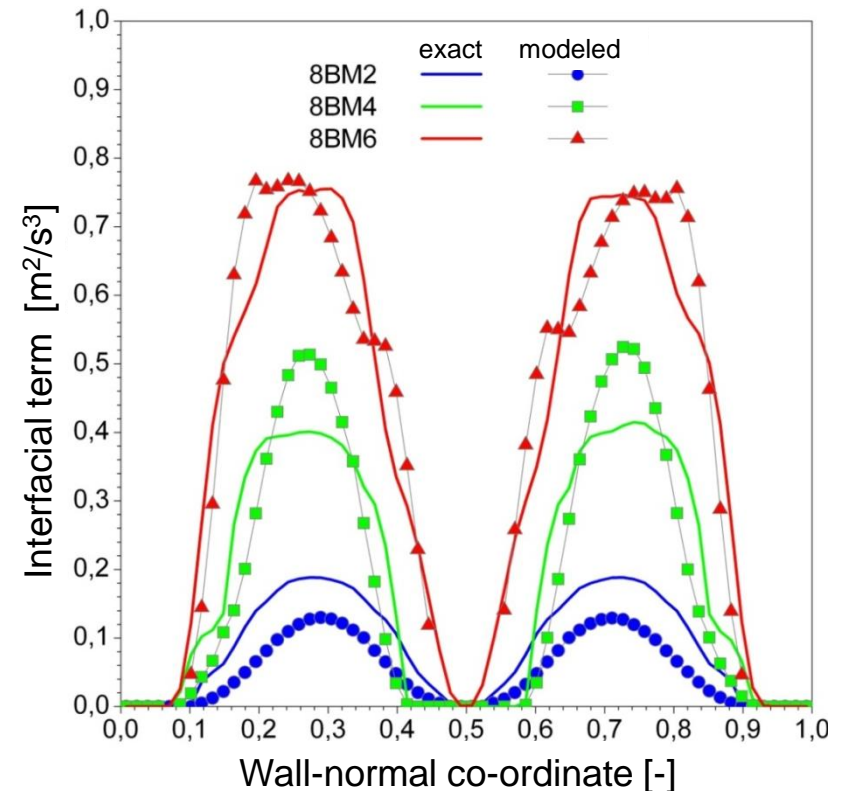
Correlation of Ishii for air-water ($M \approx 10^{-10}$)

$$C_D = \frac{2}{3} \sqrt{E\ddot{o}_B}$$



Correlation of Tomiyama (valid for $M = 10^{-14} - 10^7$)

$$C_D = \max \left\{ \min \left[\frac{16}{Re_B} (1 + 0.15 Re_B^{0.687}), \frac{48}{Re_B} \right], \frac{8}{3} \frac{E\ddot{o}_B}{E\ddot{o}_B + 4} \right\}$$



Conclusions

- Experimental data reveal complex bubble-turbulence interaction (enhancement/damping of shear turbulence)
- DNS of bubble driven liquid flow and analysis of transport eq. for liquid turbulence kinetic energy for pseudo-turbulence
 - Production by shear stresses is negligible (as expected)
 - Importance of interfacial term and diffusion term
- Evaluation of model assumptions
 - Production term and diffusion term: poor performance of standard single-phase type models (PT is over-, DT is underestimated)
 - Interfacial term: modeling as work of drag force together with Tomiyama correlation for C_D shows good performance
- Turbulence models for bubbly flows have strong deficiencies
 - Combined theoretical, experimental and numerical efforts are required to develop physically sound and general models for CFD