

Numerical evidence for a novel non-axisymmetric bubble shape regime in square channel Taylor flow

Dr. Martin Wörner

Karlsruhe Institute of Technology Institute for Nuclear and Energy Technologies





www.kit.edu

Outline



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 - Computational set-up
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 - Bubble shape for different capillary numbers
 - Local interface curvature and pressure field
 - Transition from slug flow to annular flow

Conclusions

Background



Fischer-Tropsch-Synthesis (conversion of CO & H₂ into liquid fuels)

- Sasol Inc: 6 Mio. t fuel per year by bubble column reactors (diam. 12 m)
- Monolith reactors with Taylor flow offer higher yield by similar selectivity Güttel et al. Ind. Eng. Chem. Res. 47 (2008) 6589



Simulation model



Governing equations*

Navier-Stokes equation in single field formulation with surface tension term for two incompressible immiscible Newtonian fluids with constant physical properties

Numerical method*

- Volume-of-fluid method with PLIC reconstruction
- Finite volume discretization an a staggered 3D Cartesian grid
- Projection method for pressure-velocity coupling
- Explicit 3rd order Runge-Kutta time integration scheme

In-house computer code TURBIT-VOF

* for details see Öztaskin, Wörner, Soyhan, Phys. Fluids **21** (2009) 042108



Computational set-up for Taylor flow



Set-up

- co-current downward flow of squalane (C₃₀H₆₂) and nitrogen
- consideration of one unit cell
- square channel with internal cross section 1 mm × 1 mm
- periodic boundary conditions in vertical (axial) direction

Parameter

- **gas content:** $\varepsilon_{G} = 0.2$ or 0.4
- unit cell length: L_y = 4 or 6 mm (grid up to 80×480×80 cells)
- pressure drop across the unit cell is prescribed (but varied for different cases)







Comparison of bubble shape*





*Keskin et al., AIChE Journal, in press

Steady bubble shape at different Ca





Front and rear curvature





Steady bubble shape for higher Ca









Bubble front and bubble body are axisymmetric, bubble rear is not!

Approximate relation for the local interface curvature κ at a point \mathbf{x}_i located on the interface

$$\kappa(\mathbf{x}_{i}) \approx \frac{p_{B} - p_{L,i}(\mathbf{x}_{i})}{\sigma}$$



Axial profiles of liquid pressure





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Transition to annular flow



- For large prescribed pressure drop no steady bubble velocity rise is obtained
- The liquid slug length decreases and becomes zero





Conclusions



- The curvature of the front (rear) meniscus at the channel axis increases (decreases) with increase of the capillary number
- The values of the non-dimensional front/rear curvature in a square channel are similar to those in a circular channel
- At high values of the capillary number we find a novel bubble shape regime, not reported in literature so far, where the bubble nose and body are axisymmetric, while the bubble rear is not; instead the rear meniscus shows a symmetry with respect to the channel mid-planes and diagonals
- This shape of the rear meniscus has its origin in substantial variations of the liquid pressure in cross-sections at the bubble rear; the large pressure differences are presumably caused by inertial effects along curved streamlines
- Can this shape be confirmed by experiments? What is the range of (Ca, Re) where this novel shape may exist?



Thank you for your attention

Theoretical analysis of curvature



Dynamic boundary condition normal to the interface

$$p_{\mathrm{L},i} - p_{\mathrm{G},i} + \tau_{\mathrm{L},i}^{\perp} - \tau_{\mathrm{G},i}^{\perp} = \boldsymbol{\sigma}\boldsymbol{\kappa} \qquad \tau_{\mathrm{L},i}^{\perp} = \mu_{\mathrm{L}} \left(\nabla \mathbf{v}_{\mathrm{L}} + (\nabla \mathbf{v}_{\mathrm{L}})^{\mathrm{T}} \right) : \mathbf{n}_{\mathrm{i}} \mathbf{n}_{\mathrm{i}} \tau_{\mathrm{G},i}^{\perp} = \mu_{\mathrm{G}} \left(\nabla \mathbf{v}_{\mathrm{G}} + (\nabla \mathbf{v}_{\mathrm{G}})^{\mathrm{T}} \right) : \mathbf{n}_{\mathrm{i}} \mathbf{n}_{\mathrm{i}}$$

Assumptions: $p_{G,i} = p_B = const.$, $\mu_G / \mu_L = 0.0006 \implies \tau_{G,i}^{\perp} \approx 0$ Non-dimensional interface curvature

$$K \equiv \kappa D_{\rm h} \approx \frac{D_{\rm h}}{\sigma} \left(p_{\rm L,i} - p_{\rm B} + \tau_{\rm L,i}^{\perp} \right)$$

At front and rear stagnation point:

$$\tau_{\mathrm{L,tip/rear}}^{\perp} = \mu_{\mathrm{L}} \left(\nabla \mathbf{v}_{\mathrm{L}} + (\nabla \mathbf{v}_{\mathrm{L}})^{\mathrm{T}} \right) \Big|_{\mathrm{tip/rear}} : \mathbf{e}_{\mathrm{y}} \mathbf{e}_{\mathrm{y}} = 2 \mu_{\mathrm{L}} \left. \frac{\partial v_{\mathrm{L}}}{\partial y} \right|_{\mathrm{tip/rear}}$$

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Theoretical analysis of curvature



At front and rear stagnation point:



